CAE 465/526 Building Energy Conservation Technologies Fall 2023

September 21, 2023

OpenStudio and Advanced HVAC Systems

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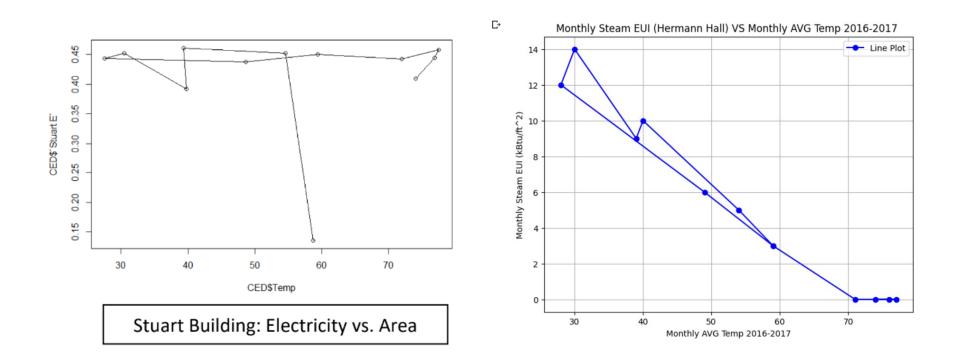
ANNOUNCEMENTS

Announcements

- Assignment 3 is posted
- Review the Q&A file regularly and please feel free to ask questions!

ASSIGNMENT 2 FEEDBACK

Feedback: Common improvements



• Feedback: Common improvements

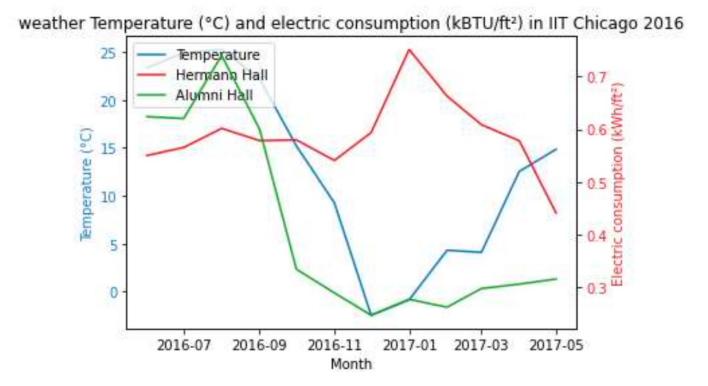
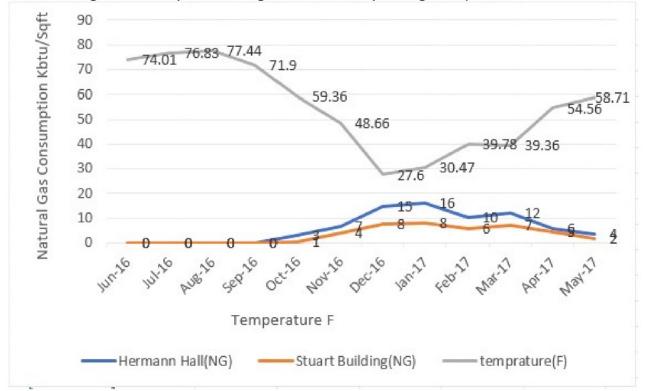


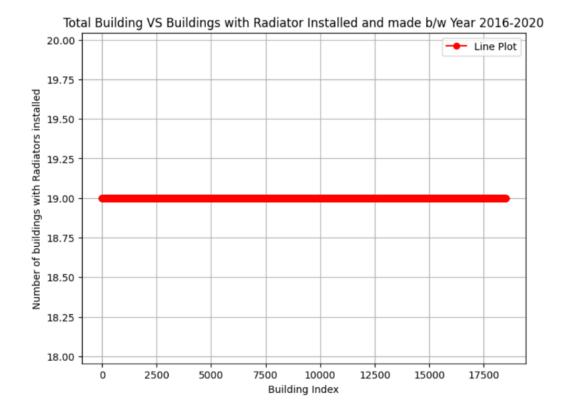
Figure 10- Electric consumption (kBTU/ft²) of the 2 buildings per month compared with the outdoor temperature (°C)

• Feedback: Common improvements

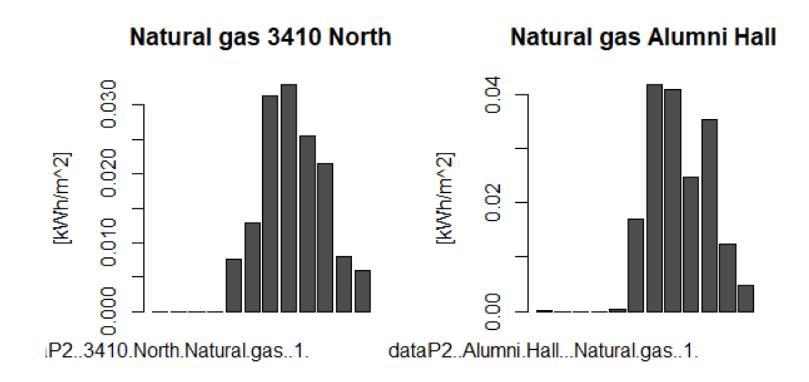
The Natural gas divided by the building area vs. monthly average temperature



• Feedback: Common improvements



• Feedback: Common improvements



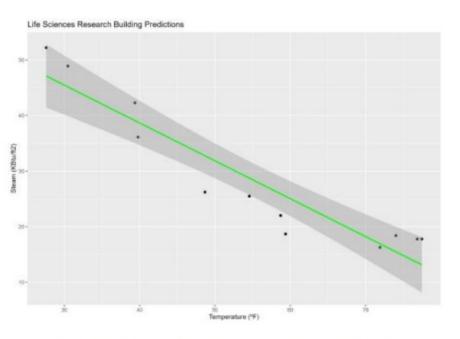


Figure 21: Figure 20: Regression line S-T (LSRB)

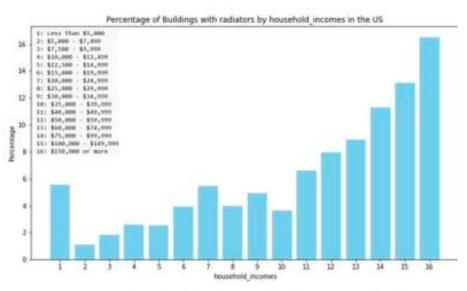
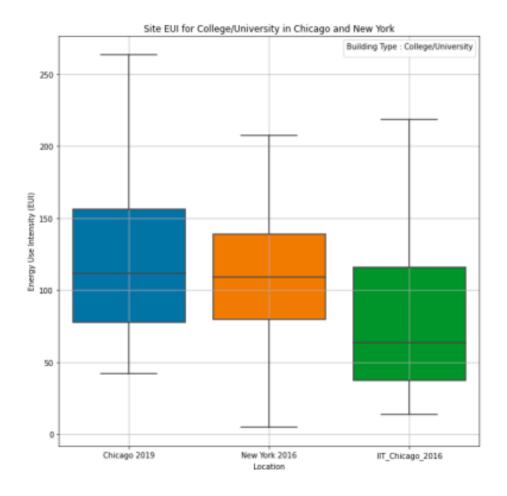


Figure 5- Percentage of weighted distribution of buildings equipped with a radiator in the US according to the household incomes



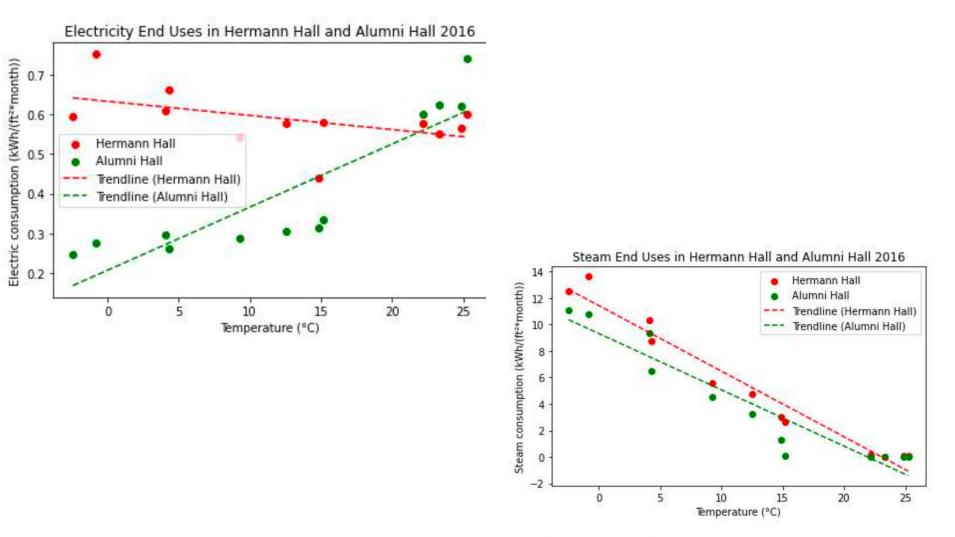


Figure 12 – Steam Energy End Uses pattern

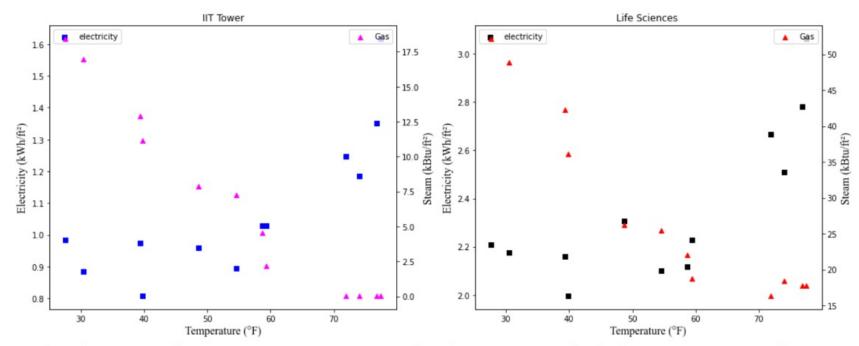
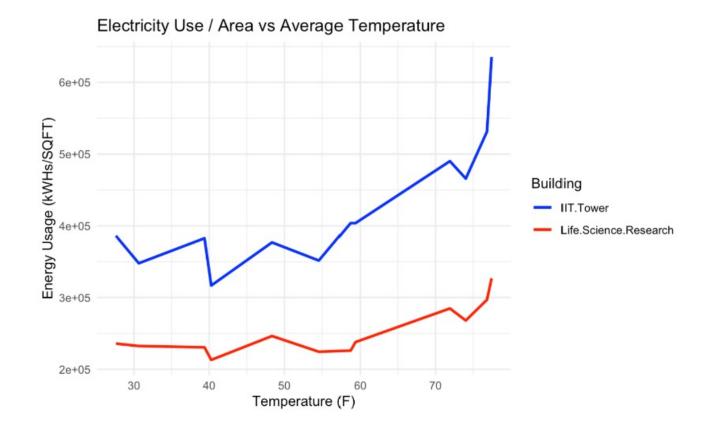
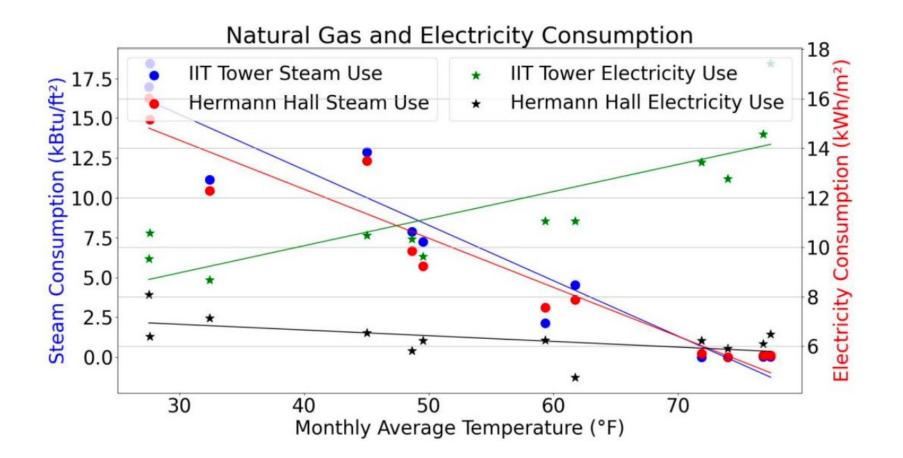
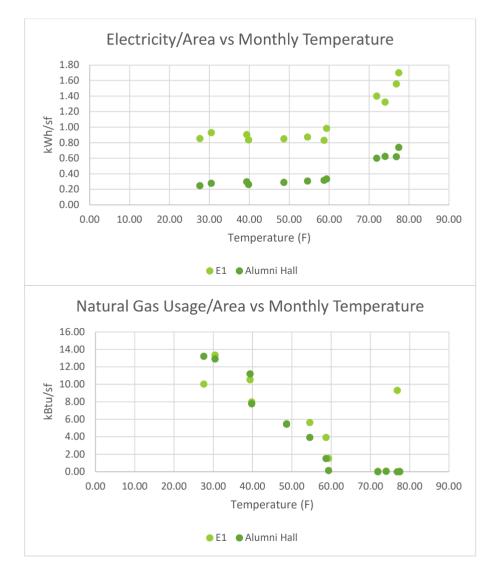
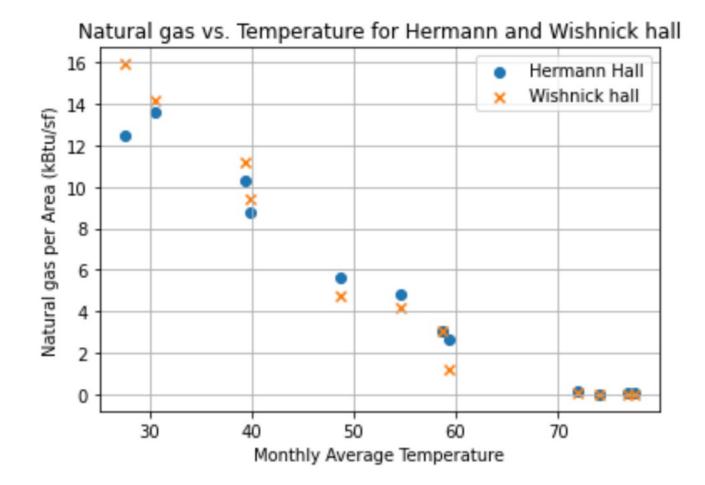


Fig.15 Electricity and steam consumption per squire foot for IIT tower and Life Science against monthly average Temperature

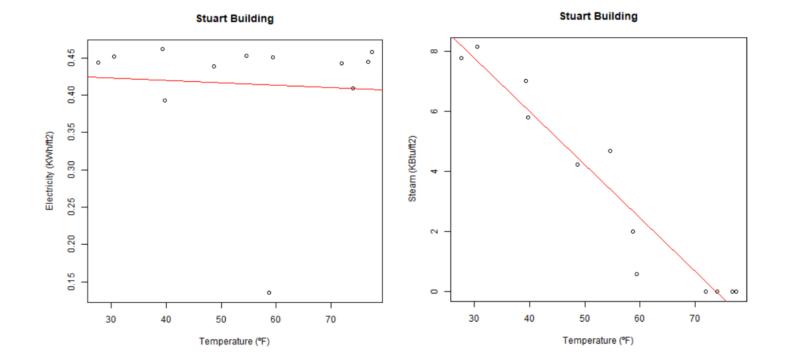




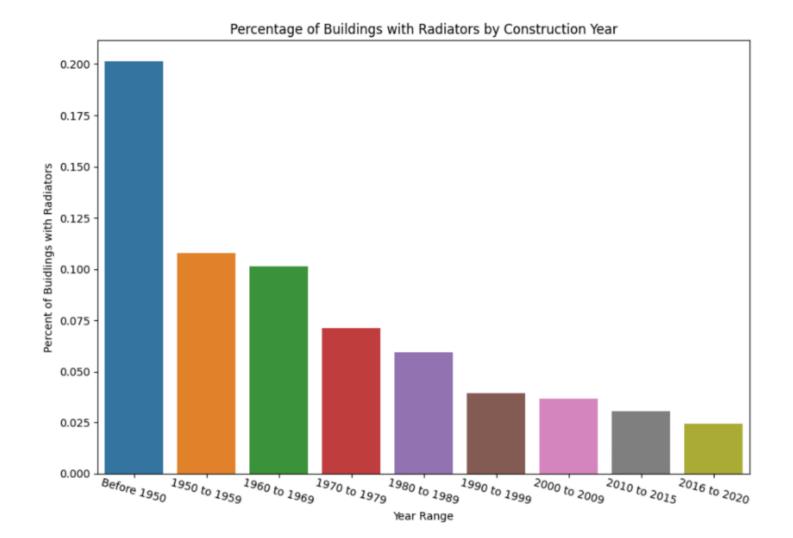




Building name	Electricity consumption per year (kBtu)	Chilled water consumption per year (kBtu)	Steam consumption per year (kBtu)	Building areas (sf)	EUI per year (kBtu/sf)
3410 North	701703,179	0	1920668,4	41 576	63.07
Alumni Hall	662978,802	0	2219287,8	39542	72.89
ASA	268310,217	0	0	15616	17.18
Carman	1380551,84	0	0	69559	19.85
Crown	0	0	0	53901	0.0
E1	5961725,13	0	9095772,6	133990	153.93
Galvin	1387321,53	0	0	92978	14.92
Gunsaulus	1145864,85	0	955,2	82 898	13.83
Hermann Hall	2674230,6	14787900	8174840,4	111 135	230.68
IITTower	17378172,3	0	31939500	392 894	125.52
Keating	1986957,36	0	0	53 163	37.37
Life Sciences	3642766,54	2524100	5413237,8	123454	93.80
Life Science Research	10312961,1	21146700	36528042	106758	636.84
МТСС	6649578,43	0	0	93 667	70.99
Perlstein	1971978,07	0	4260430,8	102517	60.79
Siegel	2750833,15	0	5543622,6	63711	130.19
Stuart Building	1426209,69	5567400	3374482,8	83 906	60.45
Wishnick	2325646,38	1641400	4800835,2	62913	139.37

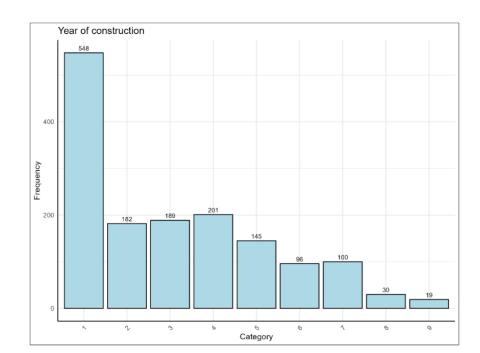


• Feedback: Good practice

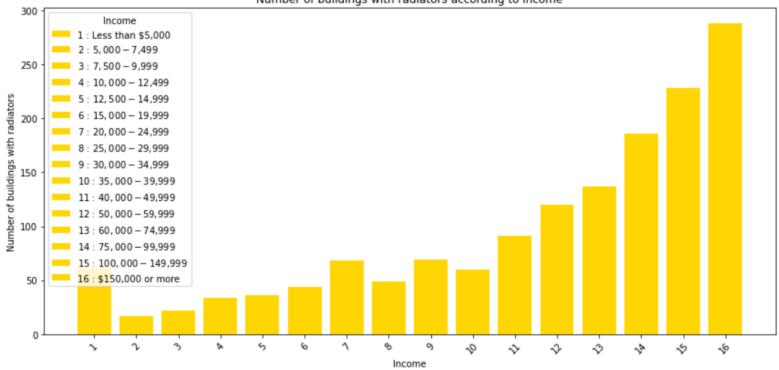


• Feedback: Good practice

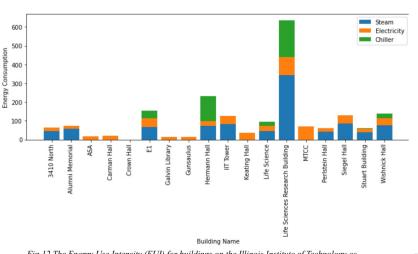
Year of construction		Count	Percentage for buildings with radiator	
1	Before 1950	548	36.3%	
2	1950 to 1959	182	12.1%	
3	1960 to 1969	189	12.5%	
4	1970 to 1979	201	13.3%	
5	1980 to 1989	145	9.6%	
6	1990 to 1999	96	6.4%	
7	2000 to 2009	100	6.6%	
8	2010 to 2015	30	2.0%	
9	2016 to 2020	19	1.3%	
	Total	1,510	100.0%	



Feedback: Good practice

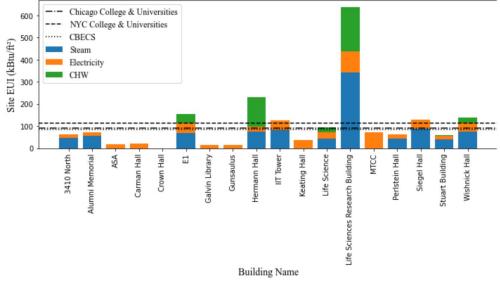


Number of buildings with radiators according to income



Feedback: Good practice •

Fig.12 The Energy Use Intensity (EUI) for buildings on the Illinois Institute of Technology ca



Site EUI for IIT in comparison to the Chicago and NYC benchmarking data

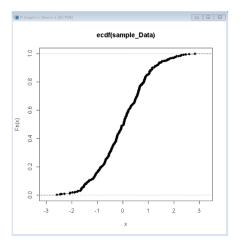
Fig. 13 The Energy Use Intensity (EUI) for buildings on the Illinois Institute of Technology campus

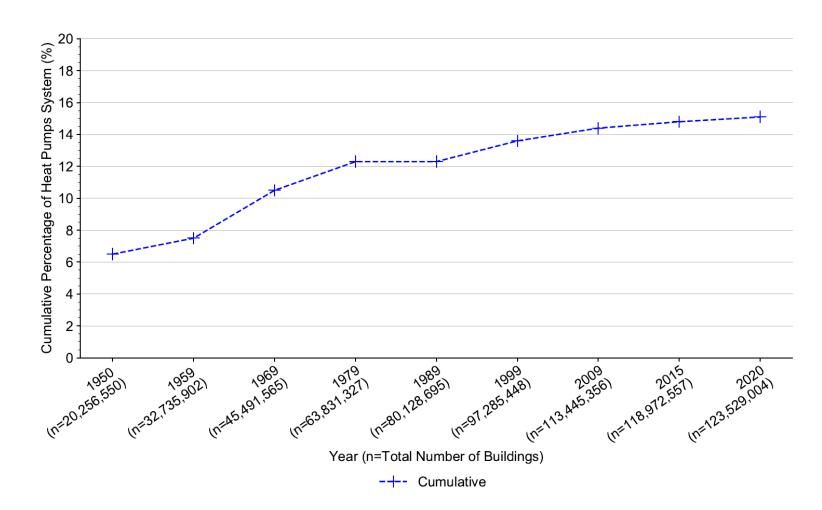
 Feedback: Good practice (maybe bar graph is not the best option here)
 Example 1: Cumulative distribution function in base R

Here, is an example of a basic Cumulative Distribution Function Plot in the R Language.



Output:





Feedback: Good practice

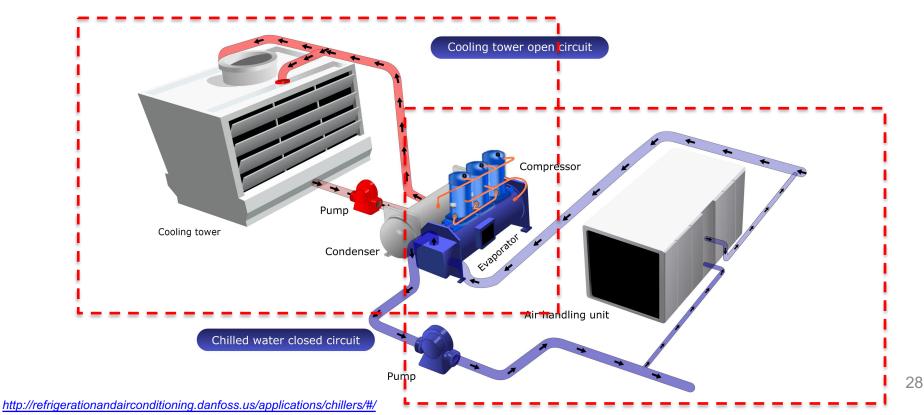
BUILDING MECHANICAL SYSTEMS

- Purpose of building mechanical systems is to control indoor air parameters within required:
 - □ Thermal comfort
 - □ Indoor air quality

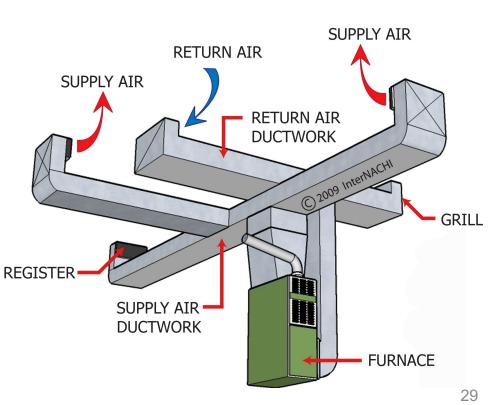
- To achieve required indoor air parameters, the system needs to conduct the following heat transfer processes:
 - Heat

 - Humidify
 - Dehumidify
 - Filter outdoor air

- HVAC system consists of four main parts:
 - □ Primary systems or central plant
 - Distribution system
 - Terminal devices
 - Controls



- Air distribution systems include air handlers, ductwork, and associated components for heating, ventilating, and air-conditioning buildings
 - □ Air distribution components:
 - Air distribution devices
 - Ductwork
 - Dampers
 - Fans
 - Controls

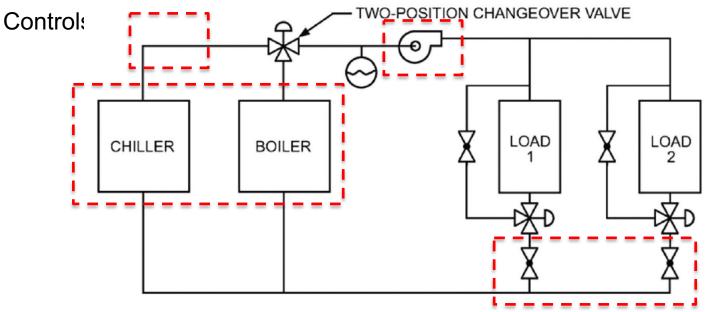


AIR DISTRIBUTION SYSTEM

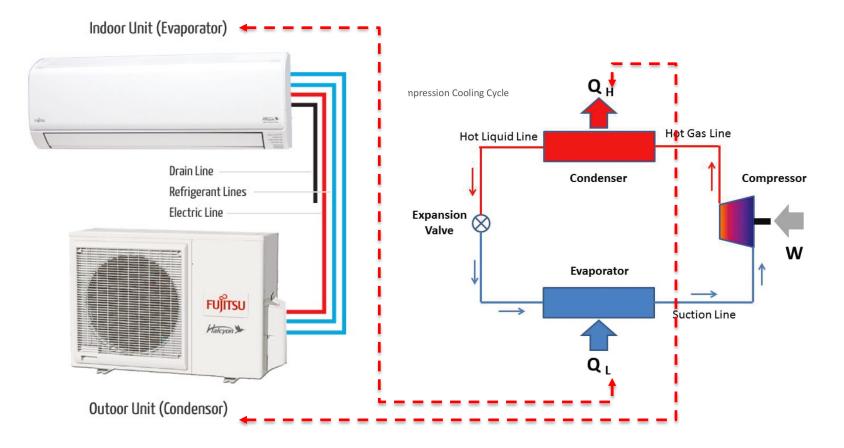
• Do we use this system at IIT?



- Hydronics refers to systems focused on heating or cooling with water:
 - □ Steam or chiller water systems
 - Boiler or chiller
 - Piping
 - Valves
 - Pumps



 Refrigeration refers to the process of removing heat from a low-temperature reservoir and transferring it to a hightemperature reservoir.



- HVAC systems categories in terms of their distribution are:
 - Unitary
 - Local systems
 - Each room has an HVAC system
 - Centralized
 - Central systems (all HVAC equipment in one room)
 - Semi-central systems
 - District
 - Central systems (all HVAC equipment in one room)
 - Semi-central systems

Few examples of the graphical symbols:

Refrigeration

Compressors

Centrifugal

Reciprocating

Rotary

Rotary screw

Condensers

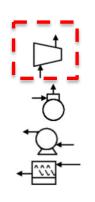
Air cooled

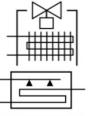
Evaporative

Water cooled, (specify type)

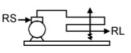
Condensing Units

Air cooled^b









Air Moving Devices and Components

Fans (indicate use)^a

Axial flow

Centrifugal

Propeller

Roof ventilator, intake

Roof ventilator, exhaust

Roof ventilator, louvered

Ductwork^b

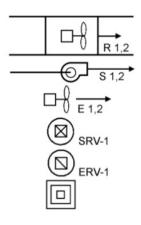
Direction of flow

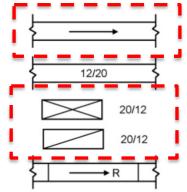
Duct size, first figure is side down

Duct section, positive pressure, first figure is top

Duct section, negative pressure

Change of elevation rise (R) drop (D)





- Heating production:
 - Equipment: Boiler, furnace, heat pump
 - □ Energy: combustion, electrical, waste heat
 - Distribution: air, steam, water

- Cooling production:
 - Equipment: Air conditioner, chiller, heat pump
 - □ Energy: electrical, natural gas, steam, waste heat
 - □ Cycle: vapor compression, absorption
 - Distribution: chilled water, air

• Three main ventilation strategies are:

□ Constant Air Volume (CAV):

- Hold the system airflow rate constant
- Let the space thermostat modulate the supply air temperature
- □ Variable Air Volume (VAV):
 - Modulate supply airflow rate
 - Hold the supply air inlet temperature constant
- Dedicated Outdoor Air System (DOAS):
 - Consist of two parallel systems
 - Deliver outdoor to handle both latent and sensible loads
 - Include a parallel system to handle mostly sensible loads

ASHRAE 90.1 APPENDIX G

ASHRAE Appendix G 90.1

• From ASHRAE Appendix G 2019:

Table G3.1.1-4 Baseline System Descriptions

System No.	<i>System</i> Type	Fan <i>Control</i>	Cooling Type ^a	Heating Type ^a
1. PTAC	Packaged terminal air conditioner	Constant volume	Direct expansion	Hot-water fossil fuel boiler
2. PTHP	Packaged terminal heat pump	Constant volume	Direct expansion	Electric heat pump
3. PSZ-AC	Packaged rooftop air conditioner	Constant volume	Direct expansion	Fossil fuel furnace
4. PSZ-HP	Packaged rooftop heat pump	Constant volume	Direct expansion	Electric heat pump
5. Packaged VAV with reheat	Packaged rooftop VAV with reheat	VAV	Direct expansion	Hot-water fossil fuel boiler
6. Packaged VAV with PFP boxes	Packaged rooftop <i>VAV</i> with parallel fan power boxes and <i>reheat</i>	VAV	Direct expansion	Electric resistance
7. VAV with reheat	VAV with reheat	VAV	Chilled water	Hot-water fossil fuel boiler
8. VAV with PFP boxes	VAV with parallel fan-powered boxes and <i>reheat</i>	VAV	Chilled water	Electric resistance
9. Heating and ventilation	Warm air furnace, gas fired	Constant volume	None	Fossil fuel furnace
10. Heating and ventilation	Warm air furnace, electric	Constant volume	None	Electric resistance
11. SZ-VAV	Single-zone VAV	VAV	Chilled water	See note (b).
12. SZ-CV-HW	Single-zone system	Constant volume	Chilled water	Hot-water fossil fuel boiler
13. SZ-CV-ER	Single-zone system	Constant volume	Chilled water	Electric resistance

a. For purchased chilled water and purchased heat, see G3.1.1.3.

b. For Climate Zones 0 through 3A, the heating type shall be *electric resistance*. For all other climate zones the heating type shall be hot-water fossil-fuel boiler.

ASHRAE Appendix G 90.1

• From ASHRAE Appendix G 2019:

Table G3.1.1-3 Baseline HVAC System Types

<i>Building</i> Type, Number of <i>Floors</i> , and Gross Conditioned Floor Area	Climate Zones 3B, 3C, and 4 to 8	Climate Zones 0 to 3A
Residential	System 1—PTAC	System 2—PTHP
Public assembly <120,000 ft ²	System 3—PSZ-AC	System 4—PSZ-HP
Public assembly ≥120,000 ft ²	System 12—SZ-CV-HW	System 13—SZ-CV-ER
Heated-only storage	System 9—Heating and ventilation	System 10—Heating and ventilation
Retail and 2 floors or fewer	System 3—PSZ-AC	System 4—PSZ-HP
Other nonresidential and 3 <i>floors</i> or fewer and <25,000 ${\rm ft}^2$	System 3—PSZ-AC	System 4—PSZ-HP
Other nonresidential and 4 or 5 <i>floors</i> and <25,000 ft^2 or 5 <i>floors</i> or fewer and 25,000 ft^2 to 150,000 ft^2	System 5—Packaged VAV with reheat	<i>System</i> 6—Packaged <i>VAV</i> with PFP boxes
Other nonresidential and more than 5 floors or >150,000 ${\rm ft}^2$	System 7—VAV with reheat	System 8—VAV with PFP boxes

Notes:

1. Residential building types include dormitory, hotel, and multifamily. Residential space types include guest rooms, living quarters, private living space, and sleeping quarters. Other building and space types are considered nonresidential.

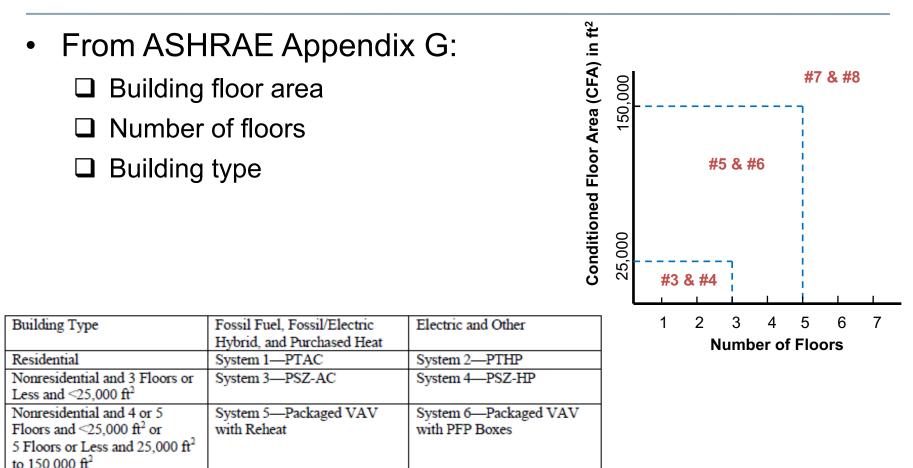
2. Where attributes make a *building* eligible for more than one baseline *system* type, use the predominant condition to determine the *system* type for the entire *building* except as noted in Section G3.1.1.

3. For laboratory spaces in a building having a total laboratory exhaust rate greater than 15,000 cfm, use a single system of type 5 or 7 serving only those spaces.

4. For hospitals, depending on *building* type, use *System* 5 or 7 in all climate zones.

5. Public assembly *building* types include houses of worship, auditoriums, movie theaters, performance theaters, concert halls, arenas, enclosed stadiums, ice rinks, gymnasiums, convention centers, exhibition centers, and natatoriums.

ASHRAE Appendix G 90.1



System 8-VAV with PFP

System 10-Heating and

Boxes

Ventilation

Ventilation

Nonresidential and More than 5

Floors or >150,000 ft²

Heated Only Storage

Note: PTAC - packaged terminal air conditioner PTHP - packaged terminal heat pump PSZ- packaged single zone VAV - variable air volume PFP - parallel fan power

System 7-VAV with Reheat

System 9-Heating and

Advanced Energy Design Guide (AEDG)

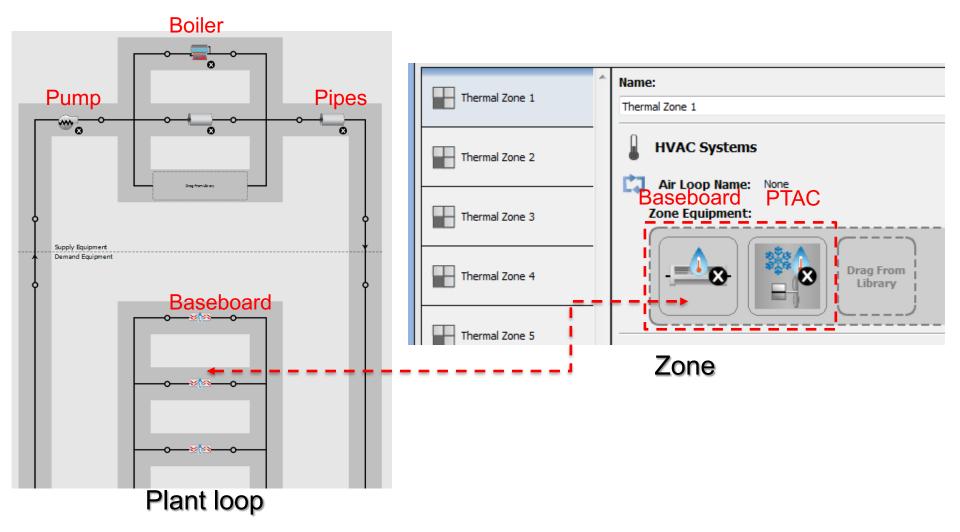
- This guideline:
 - Provide opportunities to save 30% or 50% (or net zero) site energy reductions when compared to those same facilities designed to meet the minimum code requirements of ANSI/ASHRAE/IESNA Standard 90.1-2004
 - Recommend design for a low-energy-use building and is not a minimum code or standard
 - □ Is a voluntary guidance document
 - Do not supplement, replace, or supersede existing codes and standards
 - Represents a way, but not the only way, to build energy-efficient small to medium office buildings with 50% energy savings

System Loops and Zone Equipment

- System loops vs. zone equipment
 Loops:
 - Plant
 - Chilled water
 - Condenser
 - Air

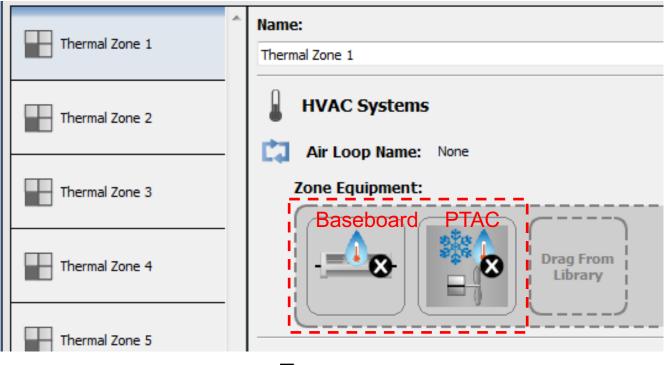
□ Zone level

No. 1: PTAC & Baseboard



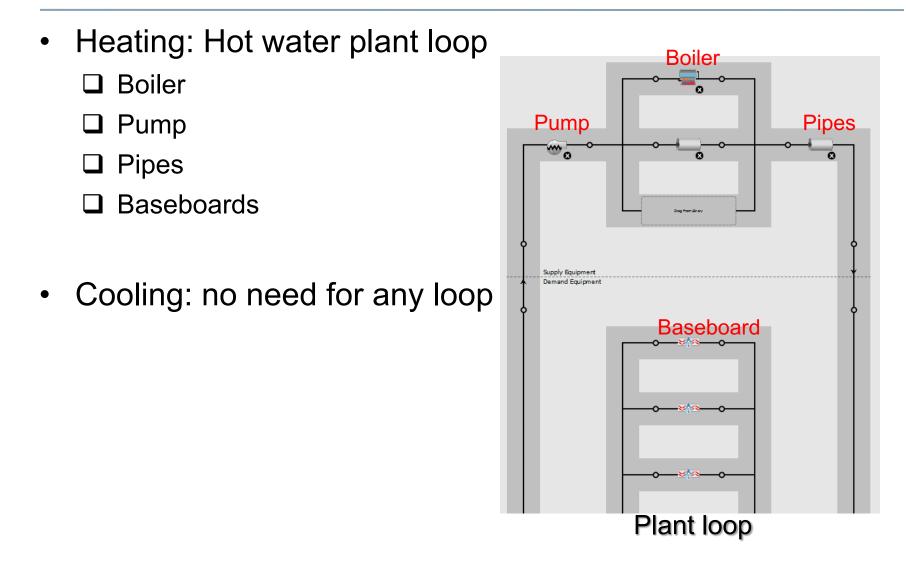
No. 1: PTAC & Baseboard

- Heating: Zone baseboard
- Cooling: PTAC



Zone

No. 1: PTAC & Baseboard



No. 1: PTAC

• Example:

- □ LG PTAC 15,000 BTU with electric heat
- Self-contained heating and AC system commonly found in hotels, condominiums, apartment buildings

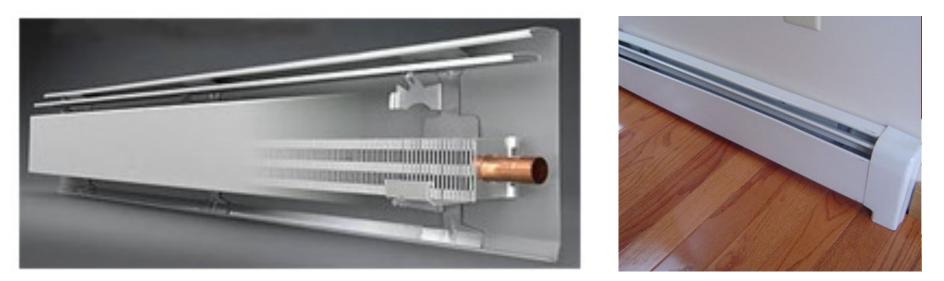


No. 1: PTAC



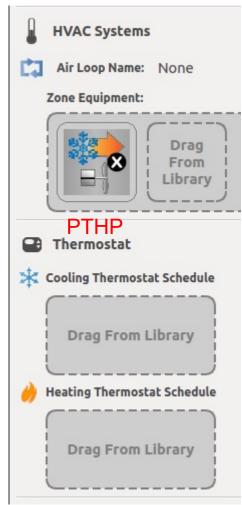
No. 1: PTAC

- Hydronic vs electric baseboards considerations:
 Initial cost
 - □ Energy efficiency
 - □ Performance (e.g., warm up, duration)



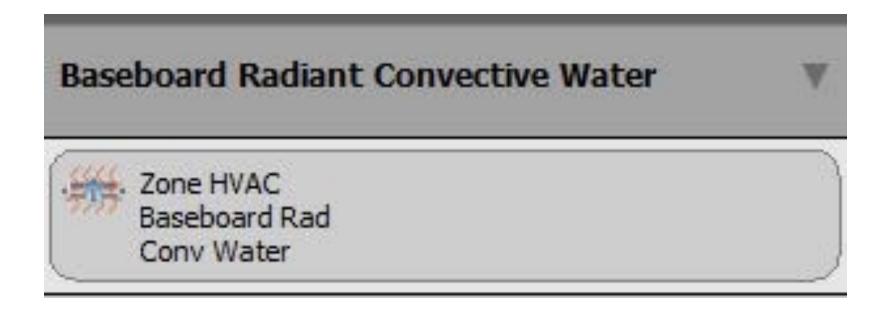
System No. 2: PTHP

- Packaged Terminal Heat Pump system includes only:
 No loop
 - □ Zone level equipment



System No. 2: PTHP

Radiative

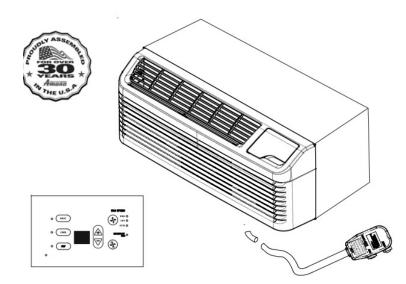


System No. 2: PTHP

- Example:
 - □ Amana 14,000 BTU (3.5 KW)
 - □ Automatic 2nd stage electric heat:
 - If the room temperature falls to 4 degrees below the set point temperature, the heat pump compressor is shut off and the electric heat strip is turned on
 - 3-Minute Compressor Lockout

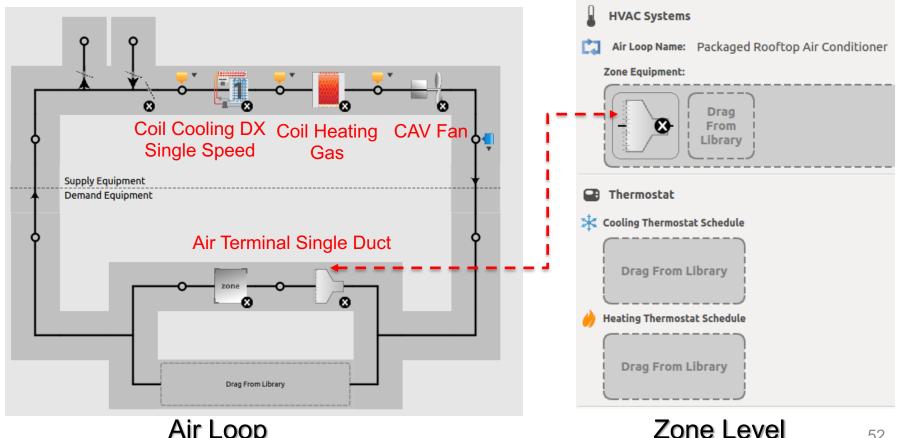
PACKAGE TERMINAL AIR CONDITIONER/HEAT PUMP INSTALLATION INSTRUCTIONS & OWNER'S MANUAL

Standard and Remote Applications with LED Control



No. 3: Packaged Rooftop Air Conditioner

This system has: One air loop □ Zone level equipment



Air Loop

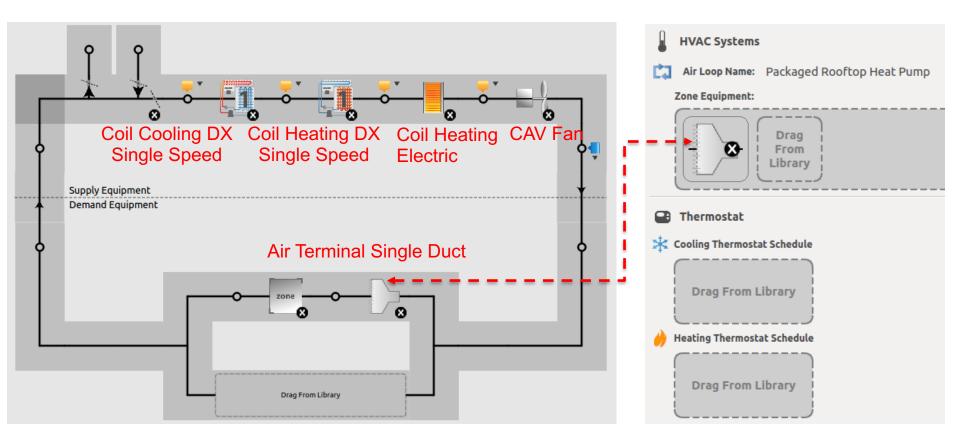
No. 3: Packaged Rooftop Air Conditioner

- Example:
 - Goodman 3.5 Ton 14 SEER 80,000 BTU Gas/Electric Package Unit
 - □ Refrigerant Type: R-410A



No. 4: Packaged Rooftop Heat Pump

This system has:
One air loop
Zone level equipment



Air Loop

Zone Level 54

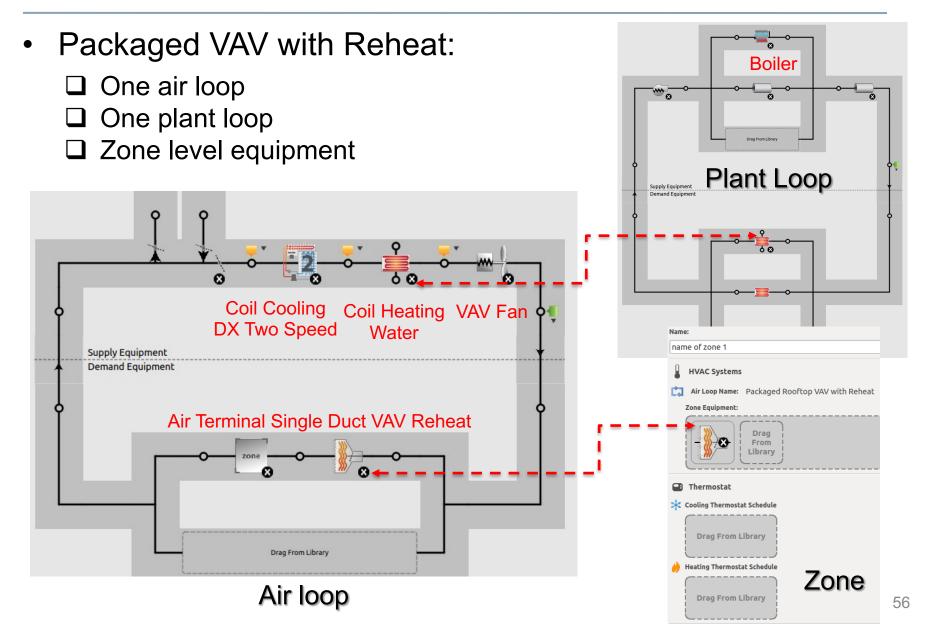
No. 4: Packaged Rooftop Heat Pump

- Example:
 - GOODMAN GPH1642H41
 - 3.5 Ton, 16 SEER Self-Contained Packaged Heat Pump, Dedicated Horizontal

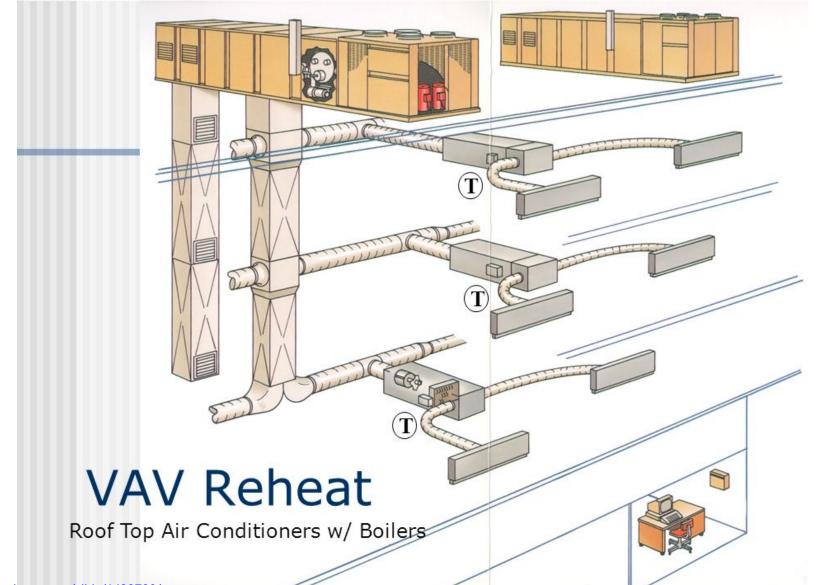


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No. 5: Packaged VAV with Reheat



No. 5: Packaged VAV with Reheat



No. 5: Packaged VAV with Reheat

• Using economizer based on ASHRAE 90.1 Appendix G:

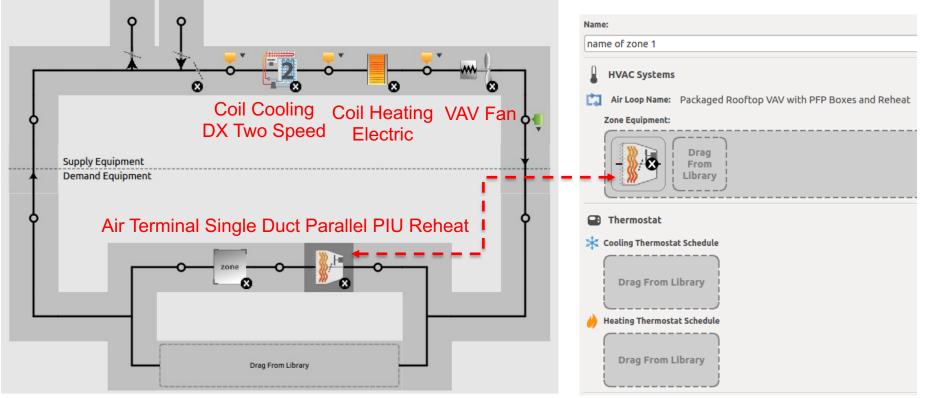
TABLE G3.1.2.6 Climate Conditions under which Economizers are Included for Comfort Cooling for Baseline Systems 3 through 8 and 11, 12, and 13

Climate Zone	Conditions
1a, 1b, 2a, 3a, 4a	NR
Others	Economizer Included

Note: NR means that there is no conditioned building floor area for which economizers are included for the type of zone and climate.

No. 6: Packaged Rooftop VAV with PFP Boxes & Reheat

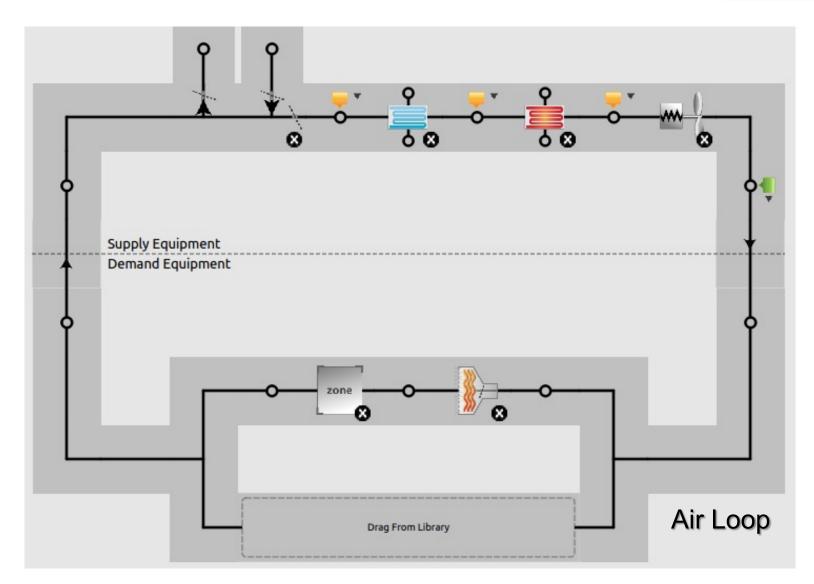
- Packaged Rooftop VAV with PFP Boxes and Reheat has:
 - One air loop
 - Zone level equipment



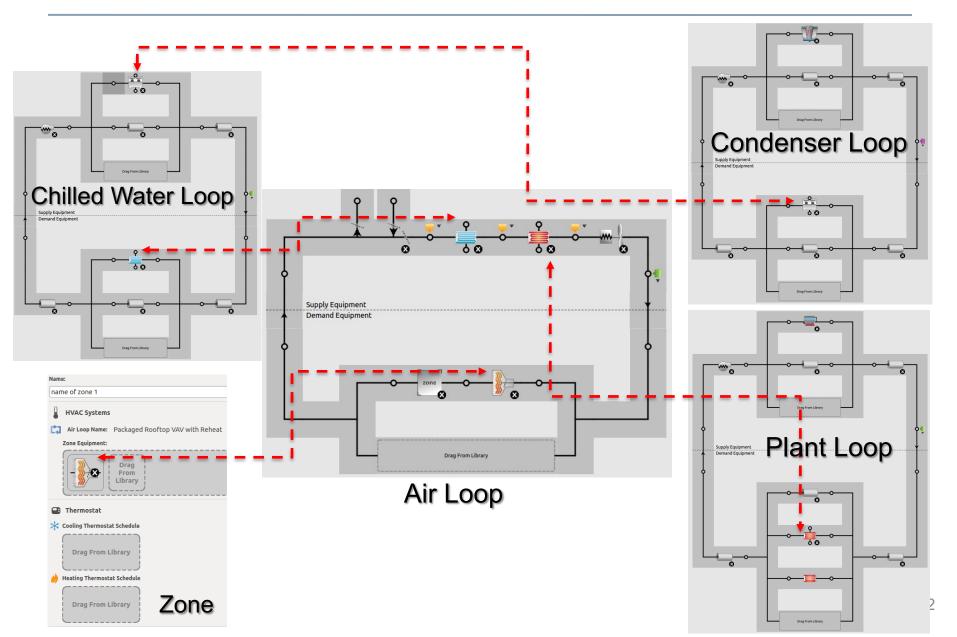
Air Loop

No. 6: Packaged Rooftop VAV with PFP Boxes & Reheat

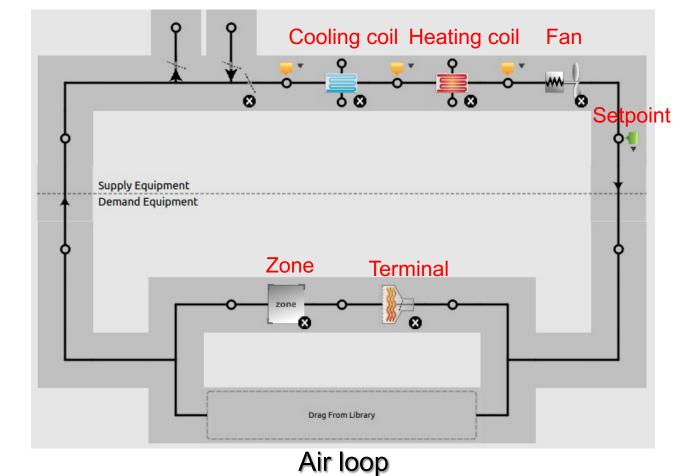
- Fan Powered Boxes:
 - □ Series: fan-powered terminals have fans that must run throughout the occupied mode in order to deliver ventilation air to the zone:
 - Act as boosters for the air handler
 - Move the air the rest of the way to the zone
 - Allow AHU to run at system pressure far lower than other type of terminals
 - Provide constant air and more air changes than other type of terminals
 - Have constant sound levels, unlike other types of terminal units that vary air volumes and/or cycle fans
 - □ **Parallel:** Parallel fan-powered terminals have fans that only switch on during the heating mode to pull warm return air from ceiling plenum:
 - The unit fan is off during the cooling mode
 - Some engineers do not specify parallel fan units because the fan cycling is often noticeable to occupants



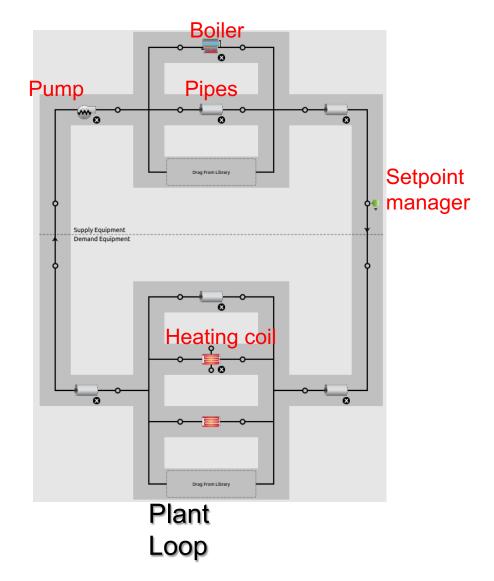
How many loops should we have?



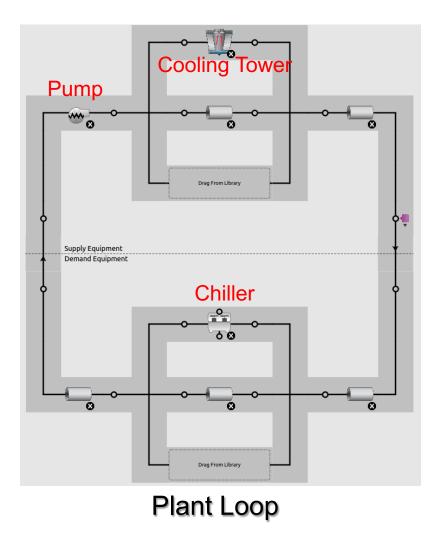
- Air loop
 - Cooling coil
 - Heating coil
 - Fan
 - Setpoint
 - Zone
 - Terminal



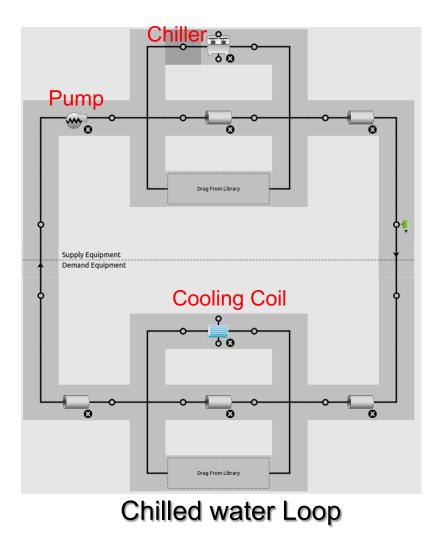
- Plant Loop
 - Boiler
 - Fan
 - Pipes
 - Heating Coil



- Condenser Loop
 - Cooling Tower
 - Fan
 - Pipes
 - Chiller



- Chilled water Loop
 - Chiller
 - Fan
 - Pipes
 - Cooling coil

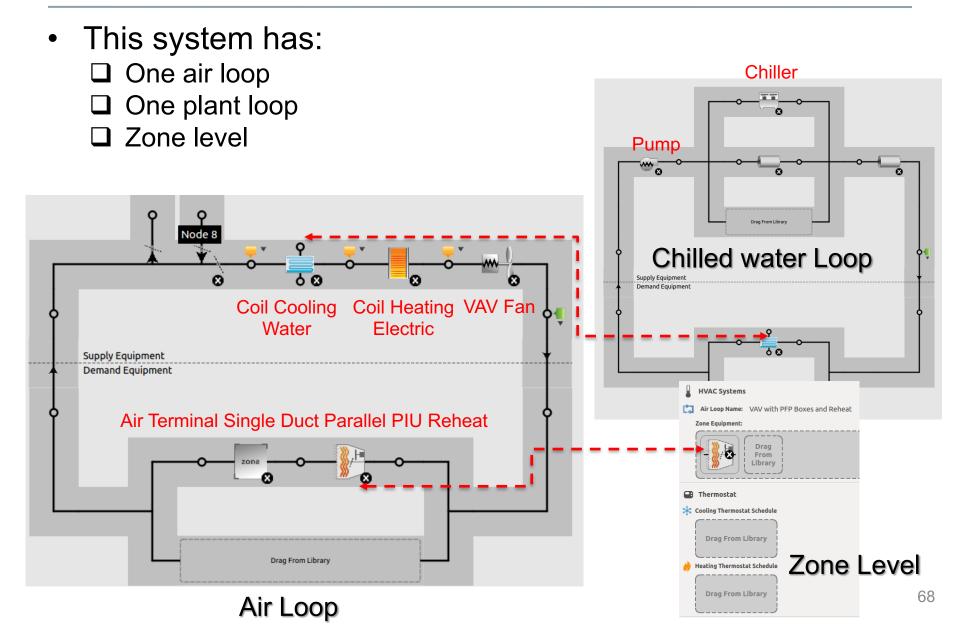


 Number and type of chillers based on ASHRAE 90.1 Appendix G:

Building Peak Cooling Load	Number and Type of Chiller(s)
≤300 tons	1 water-cooled screw chiller
>300 tons, <600 tons	2 water-cooled screw chillers sized equally
≥600 tons	2 water-cooled centrifugal chillers minimum with chillers added so that no chiller is larger than 800 tons, all sized equally

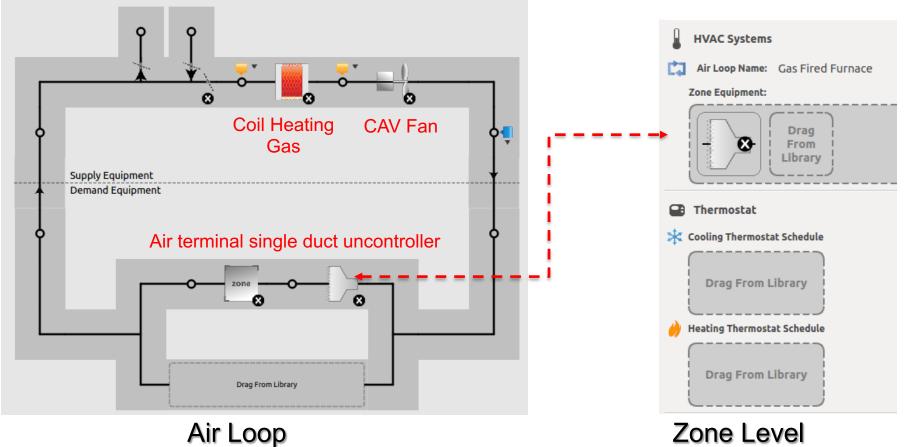
TABLE G3.1.3.7 Type and Number of Chillers

No. 8: VAV with PFP Boxes and Reheat



No. 9: VAV with PFP Boxes and Reheat

This system includes only: • □ An air loop □ Zone level

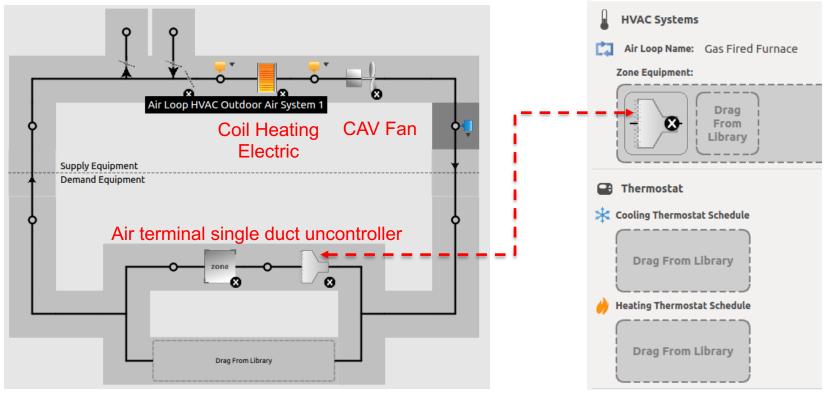


Zone Level

No. 10: Electric Furnace

This system includes only:
 An air loop
 Zone level

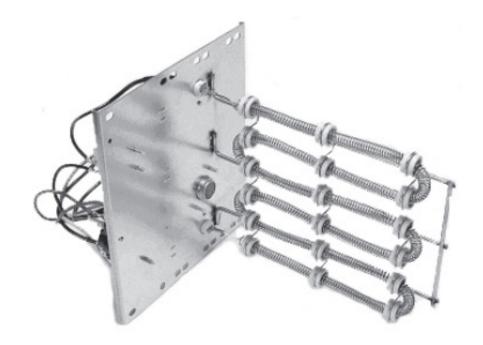
Air Loop



Zone Level

No. 10: Electric Furnace

• It includes a blower and a heater:





ASHRAE ADVANCED ENERGY DESIGN GUIDES

Advanced Energy Design Guide (AEDG)

Posted originally, 4/28/11 Reposted withminor changes, 9/27/1



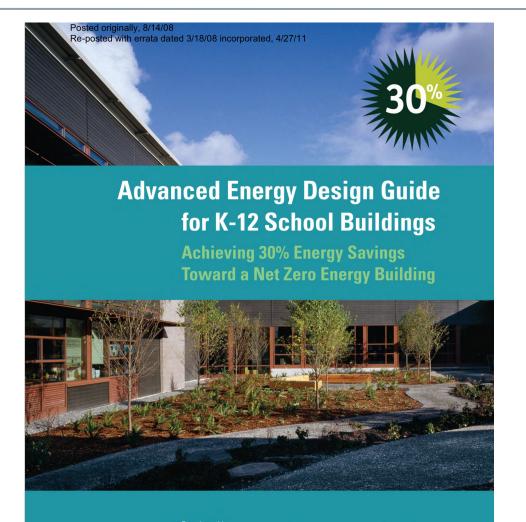
Advanced Energy Design Guide for Small to Medium Office Buildings

Achieving 50% Energy Savings Toward a Net Zero Energy Building

Developed by:

American Society of Heating, Refrigerating, and Air-Conditioning Engineers The American Institute of Architects Illuminating Engineering Society of North America U.S. Green Building Council U.S. Department of Energy

Reference System No.	Ventilation Type	Cooling Type	Heating Type	Additional Information
1	DOAS	Air-source heat pump	Heat pump w/ electric supplemental heating	Packaged single zone unit
2	DOAS	Water-source heat pump or Ground- source heat pump	Heat pump	Packaged single zone unit
3	MZ VAV	DX	 (1) Hydronic system; (2) Indirect gas furnace; or (3) Inter electric heating w/ perimeter electric convection heating 	Packaged rooftop units
4	MZ VAV	Air-cooled chiller	Gas-fired boiler	Air- handling units
5	DOAS	Air-cooled chiller	Gas-fired boiler	Fan-coils system
6	DOAS	Air-cooled chiller	Condensing boiler	Radiant system



Developed by: American Society of Heating, Refrigerating, and Air-Conditioning Engineers The American Institute of Architects Illuminating Engineering Society of North America U.S. Green Building Council U.S. Department of Energy

Posted originally, 9/28/2011 Reposted with errata dated 2/19/14 incorporated, 2/19/2014



Advanced Energy Design Guide for K–12 School Buildings

Achieving 50% Energy Savings Toward a Net Zero Energy Building

Developed by:

American Society of Heating, Refrigerating and Air-Conditioning Engineers The American Institute of Architects Illuminating Engineering Society of North America U.S. Green Building Council U.S. Department of Energy



ENERG

Posted originally, 1/11/2018 Reposted with errata dated 1/31/18 incorporated, 2/1/2018

ACHIEVING ZERO ENERGY

Advanced Energy Design Guide for K–12 School Buildings



Climate Zone 5 Recommendation Table for K-12 School Buildings

	Item	Component	Recommendation	How-to Tips
		Insulation entirely above deck	R-30.0 c.i.	EN2,17,19,21,22
	Deefe	Attic and other	R-49.0	EN3,17,19,20,21
	Roofs	Metal building	R-25.0 + R-11 Ls	EN4,17,19,21,22
		Solar Reflectance Index (SRI)	Comply with Standard 90.1*	
		Mass (HC > 7 Btu/ft ²)	R-13.3 c.i.	EN5,17,19,21
		Steel framed		
			R-13.0 + R-15.6 c.i.	EN6,17,19, 21
	Walls	Wood framed and other	R-13.0 + R-10.0 c.i.	EN7,17,19, 21
		Metal building	R-0.0 + R-19.0 c.i.	EN8,17,19, 21
		Below grade walls	R-7.5 c.i.	EN9,17,19, 21,22
		Mass	R-14.6 c.i.	EN10,17,19, 21
	Floors	Steel framed	R-38.0	EN11,17,19, 21
	FIDUIS			
		Wood framed and other	R-38.0	EN11,17,19, 21
		Unheated	Comply with Standard 90.1*	EN17,19, 21
	Slabs	Heated	R-20 for 24 in.	EN13,14,17,19,
		Heated	R-20 10r 24 In.	21.22
		Swinging	U-0.50	EN15,17
	Doors	Nonswinging	U-0.50	EN16,17
	Vestibules	At building entrance	Yes	EN17,18
		Thermal transmittance	Nonmetal framing = U-0.35	EN24
1			Metal framing = U-0.44	L1424
	View	Fenestration-to-floor-area ratio (FFR)	E or W orientation = 5% maximum N or S orientation = 7% maximum	EN24-25
	Fenestration		E or W orientation = 0.42	
		Solar heat gain coefficient (SHGC)	N orientation = 0.62	EN24,32–33
		Education and all	S orientation = 0.75	EN100.00
1	and the second	Exterior sun control	S orientation only = PF-0.5	EN26,33
1	Daylight	Visible transmittance (VT)	See Table 5-5 for appropriate VT value	DL1,5-6,23
	Fenestration	Interior/exterior sun control (S orientation only)	S orientation = no glare during school hours	DL1,9,12,13,31
		Classroom, resource rooms, cafeteria, gym, and	Daylight 100% of floor area for 2/3 of school hours	DL1-5,7-21,
	Daylighting	multipurpose rooms	Daylight perimeter floor area (15 ft) for 2/3 of school	24-30,32-41
		Administration areas Interior surface average reflectance for davlighted	hours Ceilings = 80%	DL1-5,8-12
	Interior Finishes	rooms	Wall surfaces = 70%	DL14
	Interior Lighting	Lighting power density (LPD)	Classrooms, art rooms, kitchens, libraries, media centers= 0.8 W/ft ² Cafterrias, lobbies = 0.7 W/ft ² Offices = 0.60 W/ft ² Auditoriums, restrooms = 0.5 W/ft ² Corridors, mechanical rooms = 0.4 W/ft ²	EL12–19
		Light source lamp efficacy (mean lumens per watt)	To $3 \times 5 > 2 \text{ ft} = 92$, T8 $3 \times 5 > 2 \text{ ft} = 85$, All other > 50	EL4-6
		T8 ballasts	Non-dimming = NEMA Premium Instant Start	
		10 0010303	Dimming= NEMA Premium Program Start	FLA C
		T5/T5HO ballasts	Electronic program start	EL46
		CFL and HID ballasts	Electronic	
		Dimming controls daylight harvesting	Dim all fixtures in daylight zones	EL8.9.11-19
		Summing controls daying in that vosting	Manual ON, auto/timed OFF in all areas as	
		Lighting controls	possible	EL8,9,11-20
			LPD = 0.075 W/ft ² in LZ-3 & LZ-4	
		Façade and landscape lighting	LPD = 0.05 W/ft ² in LZ-2 Controls = auto OFF between 12am and 6am	EL23
			LPD = 0.1 W/ft ² in LZ-3 & LZ-4	
	Exterior Lighting	Parking lots and drives	LPD = 0.06 W/ft ² in LZ-2 Controls = auto reduce to 25% (12am to 6am)	EL21
	5		LPD = 0.16 W/ft ² in LZ-3 & LZ-4	
		Walkways, plaza, and special feature areas	LPD = 0.14 W/ft ² in LZ-2	EL22
			Controls = auto reduce to 25% (12am to 6am)	
		All other exterior lighting	LPD = Comply with Standard 90.1*	EL25
		All outer exterior righting	Controls = auto reduce to 25% (12am to 6am)	
	-	Laptop computers	Minimum 2/3 of total computers	PL2.3
	Equipment	ENERGY STAR equipment	All computers, equipment, and appliances	PL3,5
	Choices			
		Vending machines	De-lamp and specify best in class efficiency	PL3,5
		Computer power control	Network control with power saving modes and control off during unoccupied hours	PL2,3
	Controls/ Programs	Power outlet control	Controllable power outlets with auto OFF during unoccupied hours for classrooms, office, library/ media spaces All plug-in equipment not requiring continuous operation to use controllable outlets	PL3,4
		Delision	Implement at least one:	DIA
		Policies	 District/school policy on allowed equipment School energy teams 	PL3,4

School energy teams
 Note: Where the table says "Comply with Standard 90.1," the user must meet the more stringent of either the applicable version of ASHRAE/IES Standard 90.1 or the local code requirements.

	Item	Component	Recommendation	How-to Tips
		Cooking equipment	ENERGY STAR or California rebate-qualified equipment	KE1,2
Kitchen	Kitchen Equipment ENERCY STRA or California rebate-qualified equipment KE1.2 Kitchen Equipment Walk-in refrigeration equipment Insulation on low-temp walk-in-equipment, Insulation event Web Das KE2.5 Ground Source Heat-Pump (GSHP) System with Water-cooled chiller efficiency KE3.6 KE3.6 KE3.6 Ground Source Water-cooled chiller efficiency UVD on fams HV1.1 <td>Walk-in refrigeration equipment</td> <td>Insulated floor, LED lighting, floating-head pressure controls, liquid pressure amplifier, subcooled liquid</td> <td>KE2,5</td>	Walk-in refrigeration equipment	Insulated floor, LED lighting, floating-head pressure controls, liquid pressure amplifier, subcooled liquid	KE2,5
		KE3,6		
		Gas water heater (condensing)		WH1-5
				WH1-5
	Service Water		0.81 EF or 81% Er	WH1-5
			1	
	5			
		-		
		,		
		• •		
		· ·		
			•	
Kitchen Equipment ENERGY STAR or California rebate-qualified equipment Kitchen Equipment Kitchen Equipment Walk-in refrigeration equipment In-balation on low-temp valit-in-beat operative controls, liquid pressure amplifier, subcooled liquid freigherant, exoportability foods, news and subcools Kitchen Service Water Gas water heater (condensing) Electric storage EF (512 KW, 220 gal) Electric heat-pum water heater efficiency Solar hot-water heating Solar hot-water heating Solar hot-water heating Solar hot-water heating GSHP coning efficiency SSN efficiency 200 (12 K Volume Water-colusion pumps (SSHP) solaring efficiency GSHP coning efficiency VM demand-based exhaust WM VD and NEMA Premium Efficiency 30% solar hot-water heater efficiency 30% solar hot-water heater efficiency GSHP compressor capacity control Water-colusion pumps (SSHP) System with DOAS VM demand-based exhaust Water-colusion pumps Bolier efficiency 0.4 W/tc/m VM demand-based exhaust 17.1 EER Water-colusion pumps WrD and NEMA Premium Efficiency WrD on fans Heat-Pump Water-colusion pumps WrD and NEMA Premium Efficiency WrD and NEMA Premium				
	Heat-Pump	boking equipmentENERGY STAR or California rebate-qualified equipmentKE1,2alk-in refrigeration equipment6 in. insulation on low-temp walk-in equipment, Insulated floor, LED lighting, floating-head pressure controls, liquid pressure amplifier, subcooled liquid refrigerant, evaporative condenserKE2,5thaust hoodsSide panels, larger overhangs, rear seal at appliances, proximity hoods, VAV demand-based exhaustKE3,6thaust hoodsSide panels, larger overhangs, rear seal at appliances, proximity hoods, VAV demand-based exhaustKE3,6thaust hoods0.81 EF o 19,9 - 0.0012 x VolumeWH1-5there heater selection0.81 EF o 18% Er_ 0.90 - 0.0012 x VolumeWH1-5the hot-water heater efficiencyCOP 3.0 (interior heat source)WH1-5be insulation (d < 1.5 in/d ≥ 1.5 in.)		
	with DOAS	Maximum fan power		HV12
	with DOAS	Exhaust air energy recovery in DOAS	B (dry) zones = 60% dry-bulb temp reduction	HV4,5
		DOAS ventilation control	DCV with VFD	HV4,10,15
		Water-cooled chiller efficiency	Comply with Standard 90.1*	HV2.6.11
		-	6	
	Fan-Coil System			
			A (humid) zones = 60% enthalpy reduction B (dry) zones = 60% dry-bulb temp reduction	
	Kitchen Equipment equipment equipment Kitchen Equipment Walk-in refrigeration equipment 6 in: insulation no low-temp walk-in equipment controls, liquid pressure amplifier, subcooled refrigerant, evaporative condenser Service Water Exhaust hoods 95% efficiency Service Water Gas water heater (condensing) 95% efficiency Heating Electric storage EF (s12 KW, ≥20 gal) EF > 0.99 - 0.0012 x Volume Service Water Point-of-use heater selection 0.81 EF or 81% efficiency Solar hot-water heating Solar hot-water heating 30% solar hot-water fraction when LCC effec ShiP cooling efficiency 3.6 COP 3.6 COP Ground Source Keat-Pump Solar hot-water fraction when LCC effec Water-coiculation pumps VFD and NEMA Premium Efficiency 2.6 COP Cooling tower/fluid cooler VFD on fans 4 (numid) zones = 60% enthalpy reduction DOAS Boiler efficiency Cos water circulation pumps VFD and NEMA Premium Efficiency DOAS wertilation control DCV with VFD Maximum fan power 0.4 Wicfm Maximum fan power 0.4 Wicfm Comply with Standard 90.1* Wa		HV4,10,15	
		,	Comply with Standard 90.1*	
		Cooking equipment ENERCY STAR or California rebate-qualified equipment KR chen upment Walk-in refrigeration equipment Finalition on low-temp valk-in equipment. KR walk-in refrigeration equipment Side panels, larger overhangs, rear seal at appliances, proximity hoods, VAR demand-based exhaust WR controls. liquid pressure amplifier, subcooled liquid refrigerant, evaporative condenser KR vice Water Gas water heater (condensing) 95% efficiency WR period Controls. liquid pressure amplifier, subcooled liquid refrigerant, evaporative condenser WR solar hot-water heater efficiency COP 3.0 (Interior heat source) WR solar hot-water heater efficiency 171.5 lin. WR GSHP heating efficiency 171.5 lin. solar hot-water heater specific oncy 3.6 COP HR HR KR Solar hot-water heater specific oncy 90% Eg. HR HR Solar hot-water heater specific oncy 90% Eg. HR HR Solar hot-water heater specific oncy 90% Eg. HR HR		
	Kitchen Equipment ENERGY STAR or California rebate-qualified equipment KE1 Kitchen Equipment Walk-in refrigeration equipment Insulation on low-temp walk-in equipment, Insulated foor, LED lighting, floating-head pressure cartificant, exponsitive condensare KE2 Service Water Exhaust hoods Side panels, larger overhangs, rear seal appliances, proximity hoods, VAV demand-based schaust KE3 Service Water Electric istorage EF (512 kW, 220 gal) 95% efficiency WH Fleating Electric istorage EF (512 kW, 220 gal) 95% efficiency WH Service Water Pointo-fuse heater selection 0.61 EF or 81% E _i WH Ground Source Heat-Pump Gool (rd + 15 in //d ± 1.5 in.) 1/1.5 in. WH Ground Source Gooling efficiency 17.1 EER HV1 Water-circulation pumps VFD and NEMA Premium Efficiency HV2 Ground Source Kabust air energy recovery in DOAS B (dry) zones = 60% enthalpy reduction HV4 Maximum fan power O4 W/cfm HV1 HV4 Ground Source Heat-Pump Ho1/S issee = 60% enthalpy reduction HV4 Maximum fan power O4 W			
Yet Cooking equipment ENERCY STAR or California rebate-qualified equipment Kitchen Euripment Cooking equipment 6.in. issuitation on low-temp walk-in-equipment. Service Water Exhaust hoods applicates, procinity hoods, VW demand-based exhaust Service Water Gas water heater (condensing) 95% efficiency Heating Service Water File Condensing) 95% efficiency Service Water File Condensing) 95% efficiency COP 3.0 (interior heat source) Solar hot-water heating Solar hot-water fraction when LCC effective 17.1 EER Solar hot-water fraction when LCC effective 17.1 EER 30% solar hot-water fraction when LCC effective ON GSHP cooling efficiency 30 GOP Two stage or variable speed With DOAS With Point Cooling lower/fluid cooler 0% E, 0.4 Wicfm With DOAS Exhaust air energy recovery in DOAS Comply with Standard 90.1* Par-coll System with DOAS Exhaust air energy recovery in DOAS Comply with Standard 90.1* VED and NEMA Premium Efficiency 90% E, 0.4 Wicfm Comply with Standard 90.1* VAVA-in-Handing System with DOAS Exhaust air energy rec				
	DOAS		A (humid) zones = 60% enthalpy reduction B (dry) zones = 60% dry-bulb temp reduction	
	Ducts and			
		Duct seal class	Seal Class A	HV20
	Dumpers	Insulation level	R-6	HV19
		Electrical submeters	general 120V, renewables, and whole building	QA14–17
		Benchmarking	warranty period Benchmark monthly energy use	QA14–17

Climate Zone 5 Recommendation Table for K-12 School Buildings (Continued)

*Note: Where the table says "Comply with Standard 90.1," the user must meet the more stringent of either the applicable version of ASHRAE/IES Standard 90.1 or the local code requirements.

Provide training on benchmarking

Climate Zone	Plug/Process Loads, kBtu/ft ² ·yr	Lighting, kBtu/ft ² ⋅yr	HVAC, kBtu/ft ^{2.} yr	Total, kBtu/ft ^{2.} yr
1A			20	37
2A			20	37
2B			20	37
3A			15	32
3B:CA		6	8	25
3B	11		14	31
3C			10	27
4A			19	36
4B			15	32
4C			15	32
5A			22	39
5B			17	34
6A			27	44
6B			22	39
7			30	47
8			45	62

Table 3-2	Secondary School Energy	Use Targets for 50%	6 Energy Savings
-----------	-------------------------	---------------------	------------------

Climate Zone	Plug/Process Loads, kBtu/ft ^{2.} yr	Lighting, kBtu/ft ^{2.} yr	HVAC, kBtu/ft ^{2.} yr	Total, kBtu/ft ² ⋅yr
1A		7	21	36
2A			21	36
2B			21	36
ЗA			18	33
3B:CA			10	25
3B			17	32
3C			13	28
4A	8		22	37
4B	0		18	33
4C			19	34
5A			25	40
5B			21	36
6A			31	46
6B			26	41
7			34	49
8			48	63

ASHRAE Advanced Energy Design Guide



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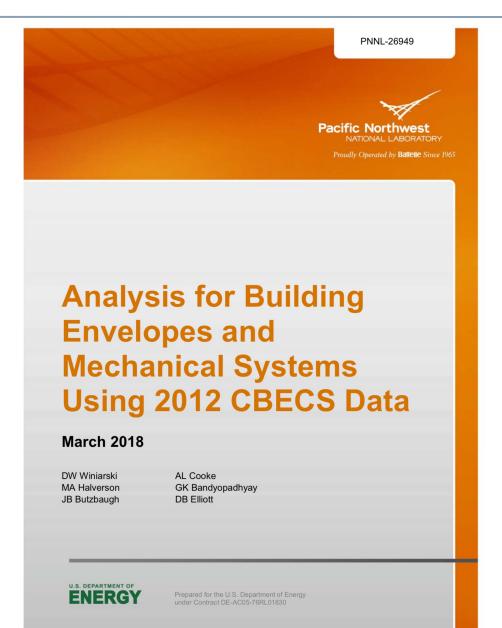
To promote building energy efficiency, ASHRAE and its partners are making the Advanced Energy Design Guides available for free download (PDF). The zero energy Guides offer designers and contractors the tools needed for achieving zero energy buildings. The 50% Guides offer designers and contractors the tools needed for achieving a 50% energy savings compared to buildings that meet the minimum requirements of Standard 90.1-2004, and the 30% Guides offer a 30% energy savings compared to buildings that meet the minimum energy requirements of Standard 90.1-1909.

ASHRAE, in collaboration with AIA (American Institute of Architects), IES (Illuminating Engineering Society), USGBC (U.S. Green Building Council) and the DOE (Department of Energy) continues to develop the Advanced Energy Design Guide (AEDG) Series.



https://www.ashrae.org/technical-resources/aedgs

CBECS HVAC SYSTEMS



- Main heating equipment are:
 - Boilers inside the building
 - District steam or hot water
 - Furnaces that heat air directly
 - Heat pumps for heating
 - □ Individual space heaters (ISH)
 - Packaged central units (PCUs), roof mounted
 - □ Some other heating equipment

- Main cooling equipment are:
 - □ Central chillers inside the building
 - District chilled water
 - □ Heat pumps for cooling
 - □ Individual room air conditioners (IRAC)
 - Packaged air-conditioning units (PACU)
 - □ Residential-type central air conditioners (Res CAC)
 - □ Swamp coolers or evaporative coolers
 - □ Some other cooling equipment.

			PNNL Determination ^(a)		
Number	Туре	Heating	Cooling	Air Distribution	
1	Large Office	PCU	Chiller	MZ VAV	
2	Medium Office	PCU	PACU	MZ VAV	
3	Small Office	PCU	PACU	SZ CAV	
4	Warehouse	PCU	PACU	SZ CAV	
5	Stand-alone Retail	PCU	PACU	SZ CAV	
6	Strip Mall	PCU	PACU	SZ CAV	
7	Primary School	Boiler	Chiller	SZ CAV	
8	Secondary School	Boiler	Chiller	MZ VAV	
9	Grocery Store	PCU	PACU	SZ CAV	
10	Quick Service Restaurant	PCU	PACU	SZ CAV	
11	Full Service Restaurant	PCU	PACU	SZ CAV	
12	Hospital	Boiler	Chiller	FCU, CAV and MZ VAV ^(b)	
13	Outpatient Health Care	PCU	PACU	MZ VAV ^(c)	
14	Small Hotel	ISH	IRAC	SZ CAV	
15	Large Hotel	ISH/PCU	IRAC/PACU(d)	SZ CAV	

Table S.4. Most Common HVAC Equipment in Post-1990 Buildings

(a) PNNL's determinations of the most common building envelope construction and mechanical system prevalence are based on analysis of CBECS data. PNNL utilizes the research and expertise of the authors to make determinations when either CBECS doesn't capture the data, or its data are conflicting or uncertain.

(b) Hospitals may utilize CV systems in some operating and critical care type areas with variable air flow used for pressurization, but classic VAV multi-zone systems in other areas like offices. CBECS guidance seems limited and other sources should be consulted.

(c) Unclear if single zone or multi-zone is more common globally, but where PCU and PACU are used VAV and likely multi-zone is more common.

(d) Large hotels may be best characterized with two system types serving different areas. Both multizone systems (VAV or CAV) may serve public spaces (lobby/conference rooms), whereas single zone IRAC or individual room heat pump systems may be most common for room space. Chiller fan coil systems appear more uncommon in new hotels. VAV appears to be found in the majority of large hotel buildings.

(e) System types

PACU - packaged air-conditioning unit	ISH - individual space heater
IRAC - individual room air conditioner	SZ – single zone
MZ-multi-zone	CAV - constant air volume
VAV – variable air volume	FCU – fan coil unit
	PCU – packaged central unit

Prototype	Prototype	By Number of Buildings		By Fl	oor Area
Number	Building	Heating	Cooling	Heating	Cooling
1	Large Office	PCU 67% Boilers 17% HP 10%	PACU 61% Chillers 27% HP 10%	PCU 48% Boilers 24% District 15%	Chillers 52% PACU 30% District 10%
2	Medium Office	PCU 50% Furnace 22% Boilers 9%	PACU 42% Res CAC 31% HP 18%	PCU 54% Boilers 18% Furnace 11%	PACU 56% HP 18% Res CAC 13%
3	Small Office	PCU 56% Furnace 22% HP 16%	Res CAC 43% PACU 33% HP 18%	PCU 67% Furnace 16% HP 12%	PACU 46% Res CAC 32% HP 16%
4	Warehouse	None 55% PCU 26% ISH 9%	None 57% PACU 16% Res CAC 16%	PCU 51% None 24% ISH 9%	PACU 49% None 23% Res CAC 13%
5	Stand-alone Retail	PCU 63% HP 15% Furnace 11%	PACU 49% Res CAC 25% HP 17%	PCU 81% HP 6% None 6%	PACU 73% Res CAC 10% HP 7%
6	Strip Mall	PCU 76% Furnace 11% HP 6%	PACU 63% Res CAC 18% Heat Pumps 16%	PCU 85% Furnace 4% Other 4%	PACU 80% Res CAC 9% Heat Pumps 9%
7	Primary School	PCU 59% HP 14% Boilers 14% Furnace 7% None 3%	PACU 48% HP 15% Res CAC 14% Chillers 11% District 8%	Boilers 45% PCU 37% HP 13% ISH 2% Furnace 2%	Chillers 39% PACU 31% HP 15% Res CAC 6% District 4%
8	Secondary School	PCU 55% Boilers 15% District 11% HP 10% ISH 4%	PACU, 34% Res CAC, 22% Chillers, 15% District, 14% HP, 9%	Boilers 41% District 26% PCU 20% HP 10% Furnace 1%	Chillers, 39% District, 30% PACU, 16% HP, 10% Res CAC, 4%
9	Grocery Store	PCU 66% None 20% ISH 7%	PACU 53% Res CAC 21% IRAC 10%	PCU 87% Boilers 6% Furnace 4%	PACU 78% Res CAC 16% None 5%
10	Quick Service Restaurant	PCU 72% None 11% HP 7%	PACU 65% Res CAC 12% None 9%	PCU 78% None 9% HP 7%	PACU 67% Res CAC 15% None 8%
11	Full Service Restaurant	PCU 62% HP 11% Furnace 10%	PACU 40% Res CAC 33% HP 14%	PCU 69% HP 7% Furnace 7%	PACU 42% Res CAC 24% HP 13%
12	Hospital	Boilers 67% PCU 14% District 14%	Chillers 76% PACU 17% District 7%	Boilers 76% District 17% PCU 6%	Chillers 79% District 12% PACU 9%
13	Out Patient Health Care	PCU 67% Furnace 13% Boilers 10%	Res CAC 44% PACU 32% HP 13%	PCU 57% Boilers 29% Furnace 6%	PACU 45% Chillers 31% Res CAC 17%
14	Small Hotel	ISH 40% Boilers 24% PCU 17%	IRAC 73% Res CAC 17% None 6%	Boilers 40% ISH 36% PCU 17%	IRAC 74% Res CAC 17% HP 5%
15	Large Hotel	ISH 39% HP 34% PCU 21%	IRAC 45% PACU 30% HP 20%	ISH 27% PCU 26% HP 21%	IRAC 37% PACU 22% Chillers 16%

Table 3.1. HVAC Equipment in Post-1990 Buildings in 2012 CBECS

DISTRICT HEATING AND COOLING

• What does district heating and cooling systems mean?

- District Heating:
 - □ Heat generated in a centralized location
 - Delivered through insulated systems
 - □ Used for space heating and water heating
- What are the pros and cons?
- Is there an example of district heating and cooling system?

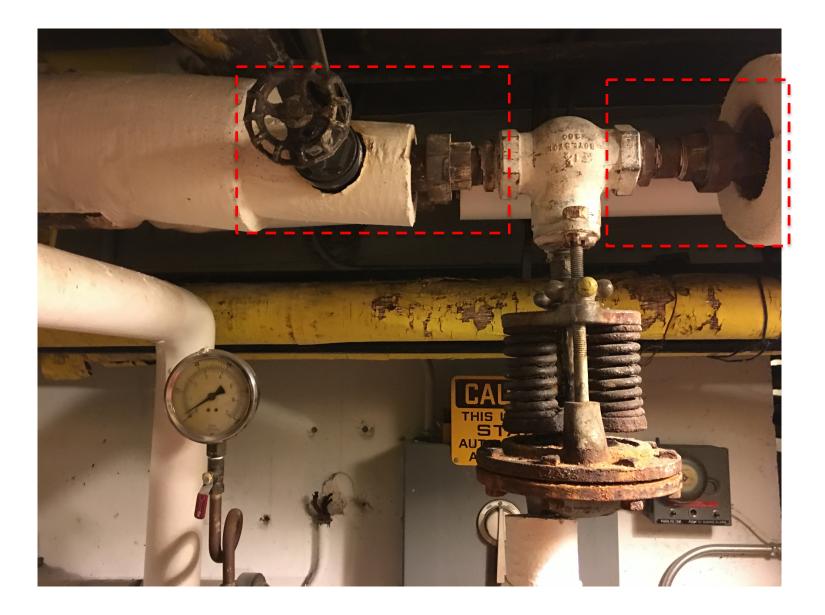


https://www.snipsmag.com/articles/93220-new-york-medical-center-saves-energy-with-insulation



• Heating plant

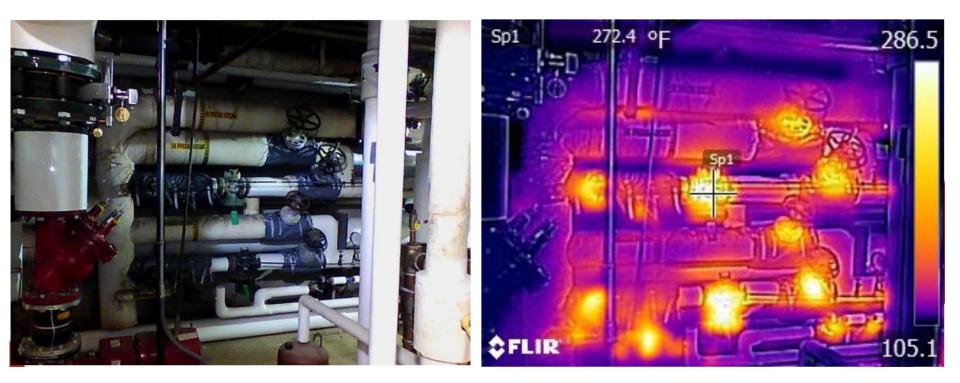




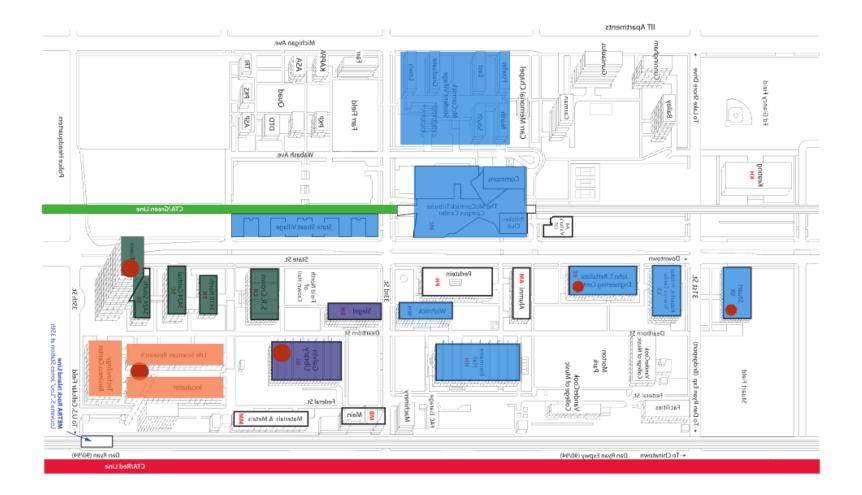
- Energy efficiency measure for steam reduction
 - Steam and condensate pipe insulation
 - Uninsulated steam pipe can have surface temperatures up to 350 degrees F.
 - Insulated over 7,800 linear feet of pipe in 28 buildings on campus
 - Steam traps
 - Steam traps are used to remove condensate from steam system.
 Many steam traps fail open, wasting steam.
 - Replaced 185 failed steam traps on campus
 - Annual energy savings of:
 - 20 million KBTU



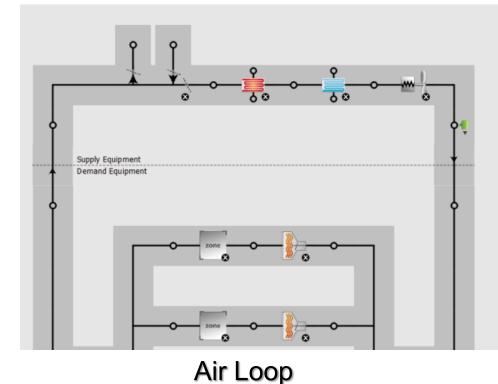
• Examples of insulating the system

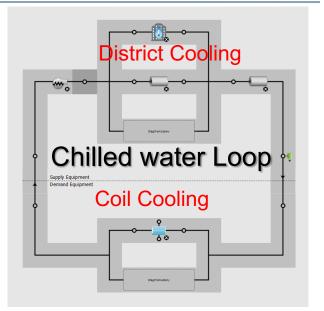


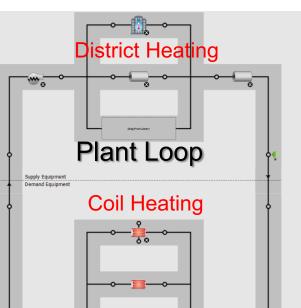
Cooling plant



- District heating and cooling:
 No assumption required on:
 - Steam
 - □ Chilled water generation





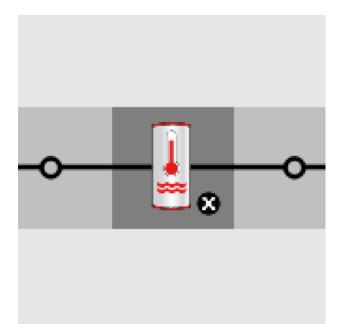


BCL

é OpenStudio	File	Preferences	Components & Measures	Help
			Apply Measure Now # Find Measures Find Components	M

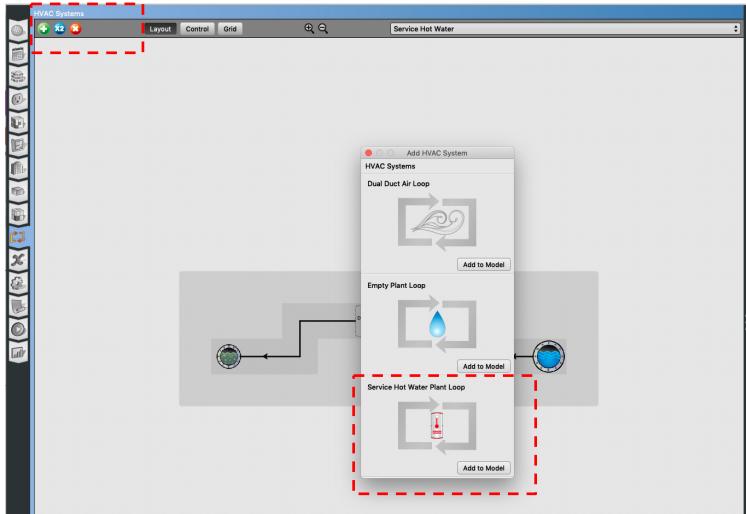
0.	_	Online BCL	_		
۹ 🗌		Chu	eck All	Attributes	
Categories				Effective R-value	6.50
Construction Assembly		Construction Assembly II 2 3 4 5 F	3913	Film Coefficients	false
Material HVAC			_	Insulation Minimum R-value (ft ² F h/Btu)	R-38
		Name: 189.1-2009 Nonres 1A Attic Floor Type: OS:Construction		Construction Type	Attic
				Construction	Attic
		Name: 189.1-2009 Nonres 1A Door Non-Swinging		Climate Zone	ASH
		Type: OS:Construction		OpenStudio Type	OS:C
		Name: 189.1-2009 Nonres 1A Door Swinging		Standard	ASH
		Type: OS:Construction		Standard Type	Nonr
	Type: OS: Name: 18	Name: 189.1-2009 Nonres 1A Exposed Floor Mass Type: OS:Construction		Arguments	
		Name: 189.1-2009 Nonres 1A Ext Slab Unheated- 4in Slab without Carpet Type: OS:Construction		• Files	
		Name: 189.1-2009 Nonres 1A Ext Slab Unheated- 4in Slab with Carpet Type: OS:Construction		189.1-2009 Nonres 1A Attic Floor_v7.1.0.idf 189.1-2009 Nonres 1A Attic Floor_v0.9.3.os 189.1-2009 Nonres 1A Attic Floor_v0.9.3.os	sm
		Name: 189.1-2009 Nonres 1A Ext Slab Unheated- 8in Slab without Carpet Type: OS:Construction		Sources	
		Name: 189.1-2009 Nonres 1A Ext Slab Unheated- 8in Slab with Carpet Type: OS:Construction		Tags Construction Assembly.Floor.Attic Floor	
		Name: 189.1-2009 Nonres 1A Ext Wall Mass Type: OS:Construction			
		Name: 189.1-2009 Nonres 1A Ext Wall Metal Building			

SERVICE HOT WATER

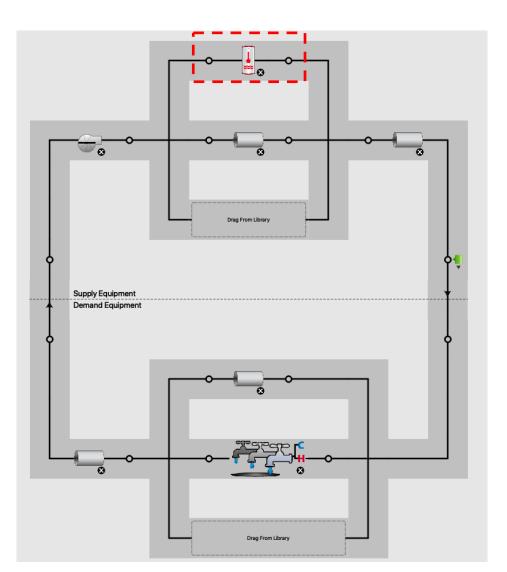




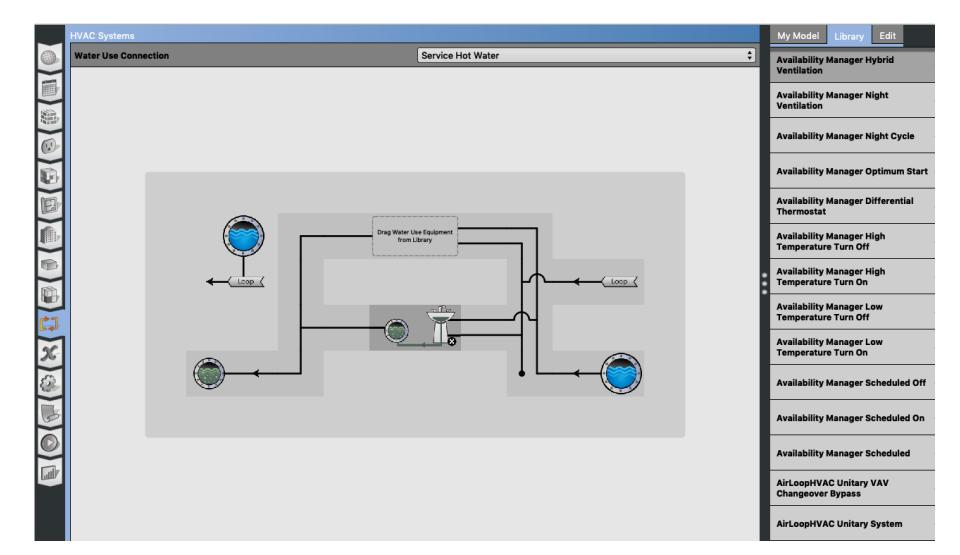
• Add a water heater tank to a plant loop:

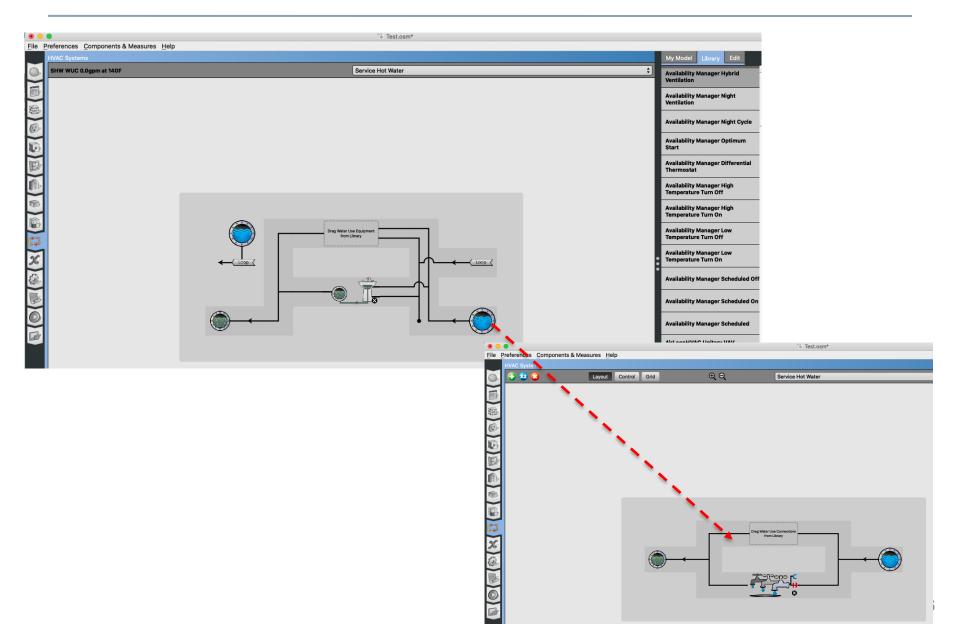


• Add service hot water plant loop:

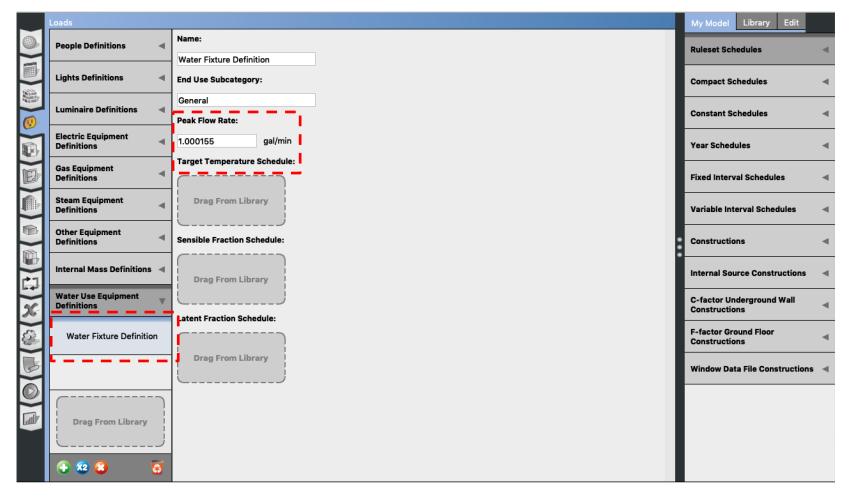


Add a fixture:





• Add service hot water definition



• DOE Reference Buildings

□ Section 5.1.6 Service Water Heater Demand

	Use Rate		Temp. at Fixture		
Space Type	gal/h	L/h	٩F	°C	Data Sources
Guest room (small hotel)	1.75	6.6	110	43	Jiang et al. 2008, ASHRAE 2007
Guest room (large hotel)	1.25	4.7	110	43	Jiang et al. 2008, ASHRAE 2007
Laundry (small hotel)	67.5	255.5	140	60	Jiang et al. 2008, ASHRAE 2007
Laundry (large hotel)	156.6	592.8	140	60	Jiang et al. 2008, ASHRAE 2007
Restrooms (primary school)	56.5	214.0	110	43	ASHRAE 2007
Restrooms (secondary school)	104.4	395.0	110	43	ASHRAE 2007
Gym (secondary school)	189.5	717.2	110	43	ASHRAE 2007
Small office	3.0	11.4	110	43	Jarnagin et al. 2006, ASHRAE 2007
Medium office (per floor)	9.9	37.5	110	43	Jarnagin et al. 2006, ASHRAE 2007
Large office (per floor)	21.3	80.6	110	43	Jarnagin et al. 2006, ASHRAE 2007
Apartment	3.5	13.2	110	43	Gowri et al. 2007
Outpatient healthcare	30.0	113.5	110	43	Doebber et al. 2009
Hospital					
ER waiting room	1.0	3.8	120	49	Engineering judgment
Operating/surgical cystoscopic	2.0	7.6	120	49	Engineering judgment
Laboratory	2.0	7.6	120	49	Engineering judgment
Patient room	1.0	3.8	120	49	Engineering judgment

Table 11 Peak Service Hot Water Demand and Data Sources

- Make reasonable assumptions for the water heater temperature:
 - Most households require about 120 °F
 - Some manufacturers set water heater thermostats at 140 °F, which also slows mineral buildup and corrosion in your water heater and pipes
 - Water heated at 140°F also poses a safety hazard (scalding)

- Make reasonable assumptions for the flow rates and sizes. For example for residential units:
 - Small size: A 50 to 60-gallon storage tank is usually sufficient for 1 to 3 people
 - Medium size: A 80-gallon storage tank works well for 3 to 4 people
 - Large size: A large tank is appropriate for four to six people

• You can use the OpenStudio measures:

	Online BCL	
Q [Check All
Envelope Electric Lighting Equipment People HVAC Refrigeration Service Water Heating Water Use Water Heating Distribution Onsite Power Generation Whole Building Economics Reporting	Service Water Heating	8
	Name: Set Water Heater Efficiency, Heat Loss, and Peak Water Flow Rate Measure Type: ModelMeasure	
	Name: Set Site Water Mains Temperature Measure Type: ModelMeasure	Г
	Name: AedgK12Swh Measure Type: ModelMeasure	M
	Name: AedgOfficeSwh Measure Type: ModelMeasure	M
	Name: ZEDG K12 SWH Measure Type: ModelMeasure	M
	Name: Add SWH Loop Measure Type: ModelMeasure	₽
	Name: Water Heater Mixed Multiplier Measure Type: ModelMeasure	Г.
	Name: Water Heater Mixed Percent Change Measure Type: ModelMeasure	–

- You can use the OpenStudio measures:
 - First, use "Add SHW Loop"

People	A Name	
HVAC	Add SWH Loop	
Refrigeration	Description	
	Simply adds a SWH loop based on usua	l inputs.
Service Water Heating	4	
Water Use	Modeler Description	
▼ Water Heating	4	
BCL Add SWH Loop		
BCL AedgK12Swh		
BCL AedgOfficeSwh	System Name.	
BCL ZEDG K12 SWH		
Distribution	Space Type.	

- Second, use "ZEDG K12 SHW"

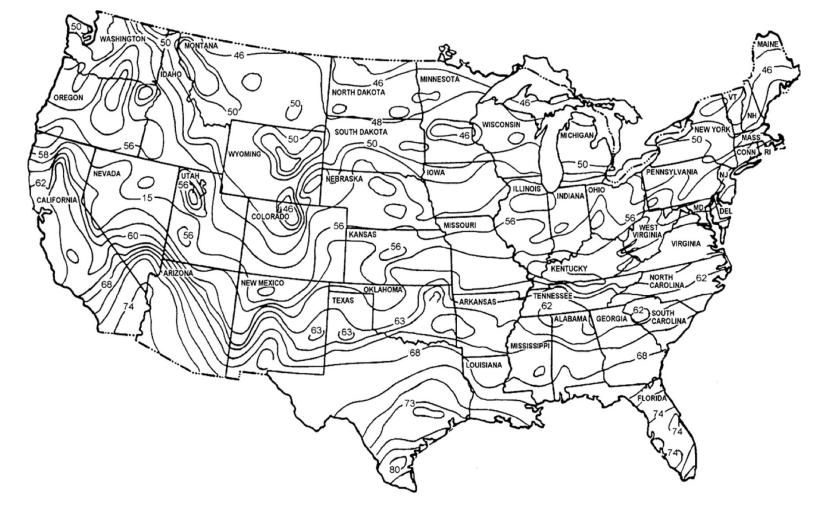
People	▲ Name
HVAC	ZEDG K12 SWH
Refrigeration	Description
Service Water Heating	4 Use 90% efficient natural gas-fired storage tank water heater. Water use demand is caluciated per student.
▶ Water Use ▼ Water Heating	4 Modeler Description
BCL Add SWH Loop	
BCL AedgK12Swh	
BCL AedgOfficeSwh	Total Cost for Kitchen System (\$).
BCL ZEDG K12 SWH	0
Distribution	Total Number of Students.

GROUND SOURCE HEAT PUMP (EXTRA)

- Ground Source Heat Pump (GHP) benefits from:
 Relative constant ground temperature (about 30 ft)
 - 50 °F (10 °C) to 59 °F (15 °C)
 - Soil temperature warmer than air in winter and colder than the air temperature in summer
 - □ Thermal storage capacity of the ground
 - Sink in summer time
 - Source in winter time

• Undisturbed ground temperature in the U.S.:

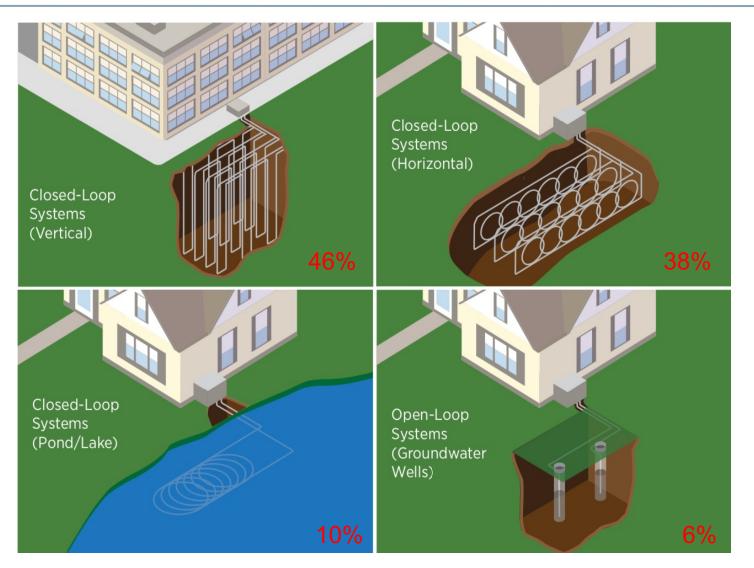
ORNL 2001-03084/abl



- Key components are:
 - □ A heat exchanger
 - A group of pipes buried in the ground
 - Immersed in a surface water body
 - Exchanging heat directly with ground water
 - Distribution systems
 - Ductwork for forced air heating/cooling and/or,
 - In-floor piping for radiant heating

□ Heat Pump (HP)

Connect the distribution system with the ground heat exchangers



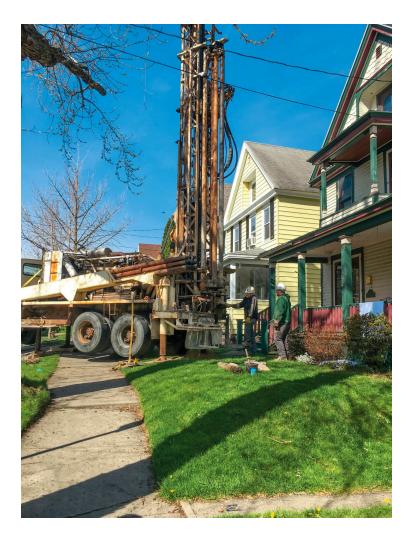
What is the benefits and setbacks of each system? Which is one is more common in the US?

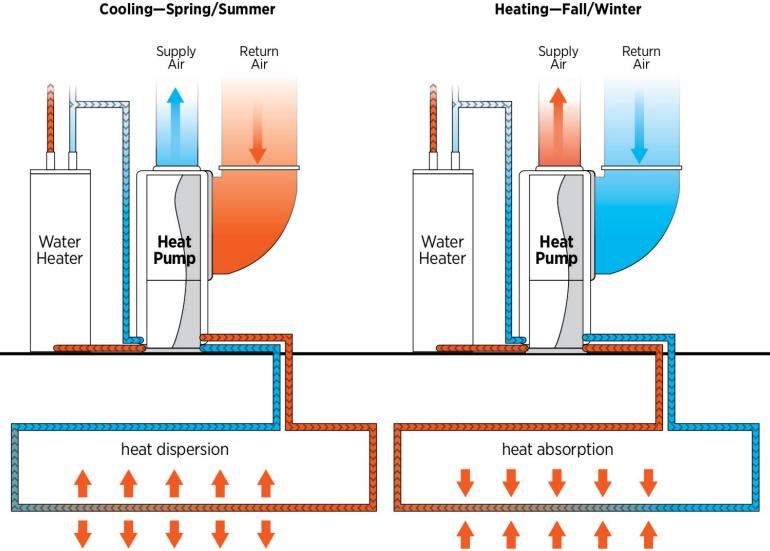


Installation of a horizontal closed-loop ground heat exchanger for a geothermal heat-pump system. Photo credit: Ed Lohrenz/ International Ground Source Heat Pump Association



https://www.oldhouseonline.com/repairs-and-how-to/geothermal-options-old-homes https://www.homebuilding.co.uk/ground-source-heat-pumps-need-know/ GeoVision Report 2019

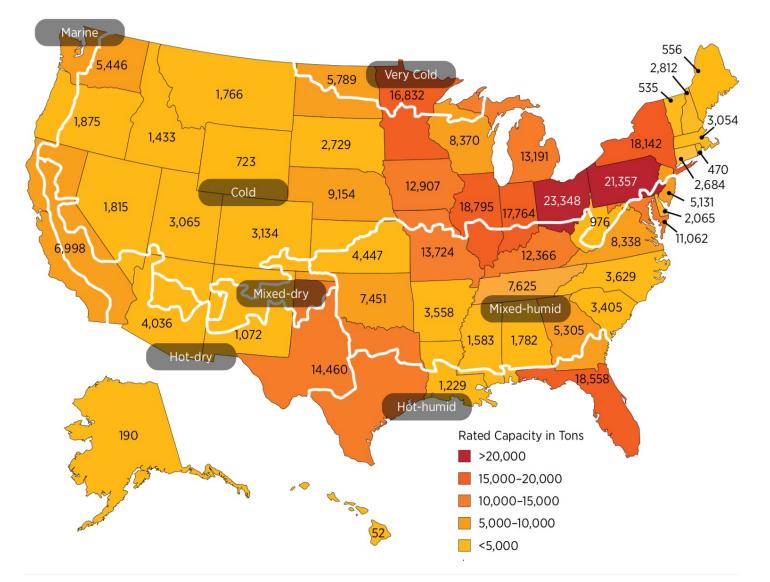




Heating—Fall/Winter

GeoVision Report 2019

- Installed capacity in the U.S. was 16,800 MW (or 4.8 million cooling tons) as of 2016
- GHP is being used more in residential buildings than commercial buildings
- In residential buildings:
 - □ 75% in new construction
 - □ 25% in retrofitted homes
- GHPs account for 1% of the U.S. HVAC market



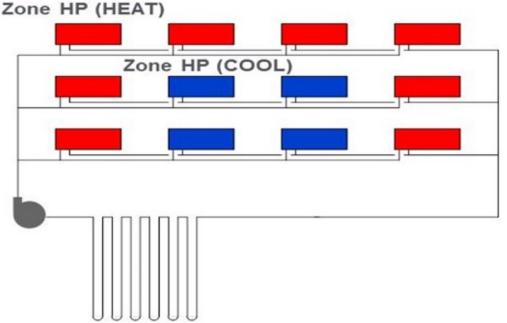
• 10 states account for 52% of the GHP shipments:

Florida, Illinois, Indiana, Michigan, Minnesota, Missouri, New York, Ohio, Pennsylvania, and Texas

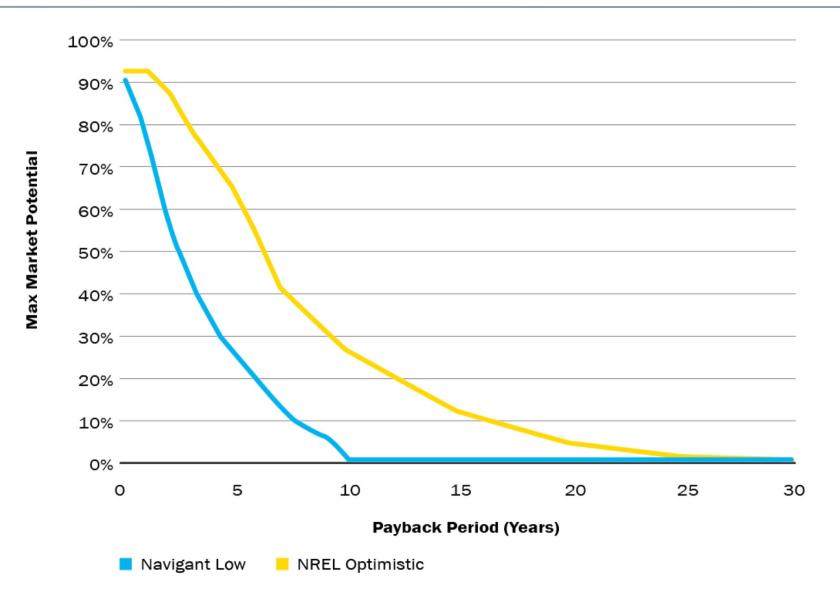
Concentrated in areas with cold climate and high population density

- The most common GHP system configuration in the US homes are:
 - □ A packaged heat pump
 - Split water-to-air heat pump (WAHP) with a centrally ducted forced-air distribution system that conditions:
 - One floor of a multistory home
 - The entire house

- For commercial buildings distributed GHP systems are the most common:
 - □ Each zone of the building is conditioned with an individual WAHP
 - □ Multiple WAHPs are connected to a common water loop
 - □ Use a two-pipe water loop with a variable speed central pumping
 - The cooling capacities of the WAHP units usually range from 0.5 to 20 ton (1.74–70 kW)



- Central GHP systems use large heat pumps or modular water-to-water heat pumps to:
 - Generate hot and/or chilled water for delivery to the conditioned space (a good option for retrofitting existing central chiller and boiler systems)
 - To satisfy the simultaneous demands for heating and cooling in different zones of a building (what's the pipe configuration?)

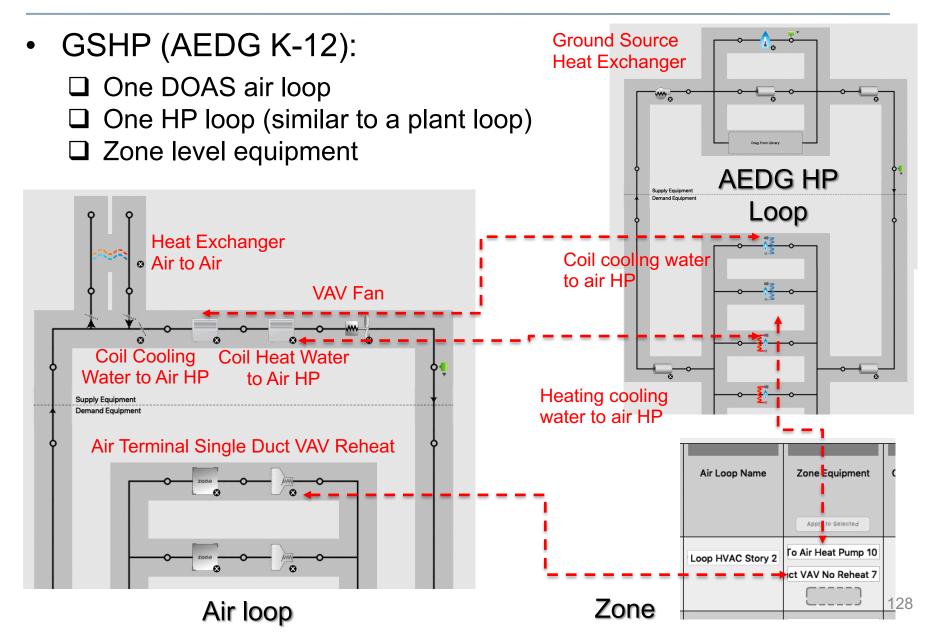


- Cost depends on the geological conditions, building loads, system designs, and the HP equipment. For a typical:
 - □ Home varies from \$3,000 to \$5,000
 - □ Large-scale housing retrofit is about \$4,600 per ton (in 2006)
 - □ For commercial buildings, the cost ranges are (in 2012):

Item	Cost	
	(\$)	
Water-source heat pump	$2.5-4/ft^2$	
Closed-loop vertical GHX	6.76–15/ft	
Total system	$13-26/ft^2$	

• A simple payback is about 8-14 years for a retrofit project and shorter for a new construction.

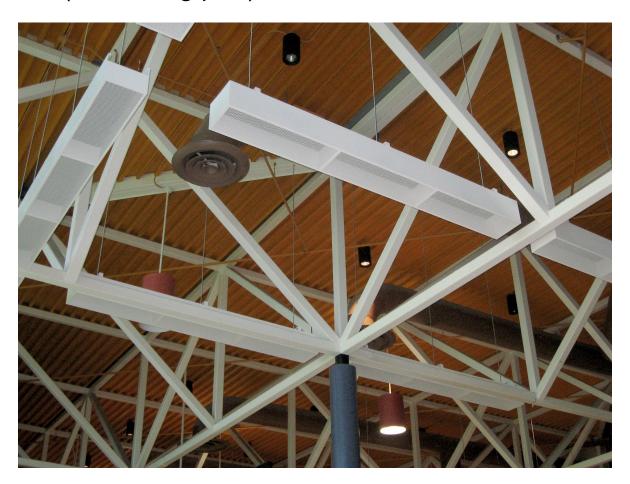
Ground Source Heat Pump (AEDG K-12)



PASSIVE CHILLED BEAMS (EXTRA)

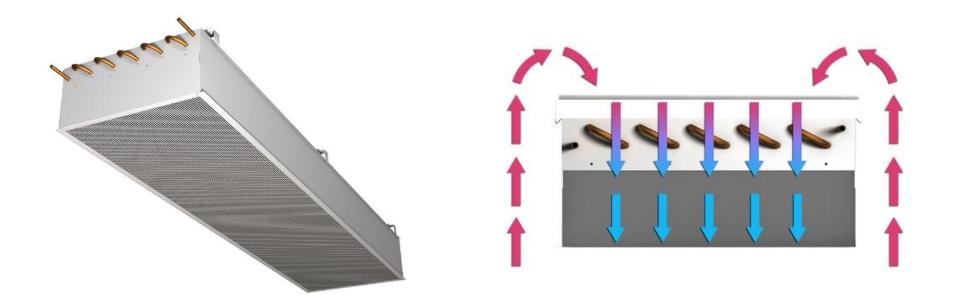
Passive Chilled Beams

There are two types of chilled beams
Active (air comes from the AHU and is pressurized)
Passive (no moving part)



Passive Chilled Beams

It works based on the natural convection
 Water temperature is about 57 F (15-20 F colder than the air temperature)



Passive Chilled Beams

- Since moving parts are involved, it is energy efficient
- It is usually used with other air systems such as
 - Displacement ventilation
 - Under floor
 - □ Active chilled beams

CHILLED CHILLED BEAMS (EXTRA)

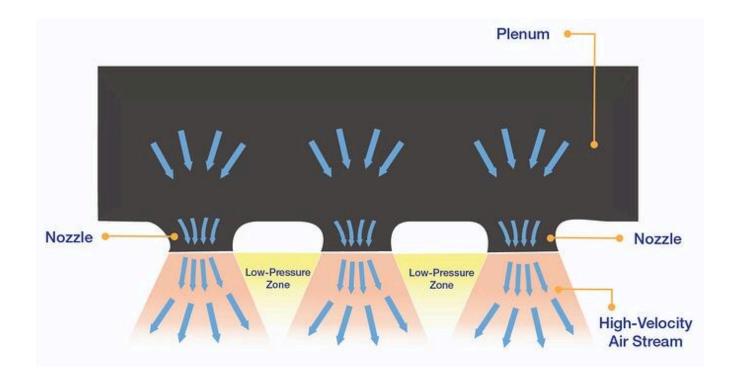
Active Chilled Beams

 Relatively energy efficient over time, reduce the size of the ductwork, and lower the noise level



Active Chilled Beams

 From AHU to plenum – and then pressurized and move to the space



Active Chilled Beams

The ratio of air through nozzles versus the induced is important

