CAE 464/517 HVAC Systems Design Spring 2023

February 21, 2023

Air distribution systems: Classification of air diffusion

Built Environment Research @ IIT] 🗫 🕣 🍂 🛹

Advancing energy, environmental, and sustainability research within the built environment www.built-envi.com Dr. Mohammad Heidarinejad, Ph.D., P.E.

Civil, Architectural and Environmental Engineering Illinois Institute of Technology

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ANNOUNCEMENTS

Announcements



Mechanical Engineering Tips

in Commercial Buildings

SPEAKER

Mechanical Engineer Danielle Passaglia

WHEN

February 23rd, 2023 12:40 pm – 1:40 pm

WHERE

John T. Rettaliata Engineering Center, RE 242

TALK ABOUT

 ✓ Work Experiences
 ✓ Careers in Mechanical Engineering
 ✓ Skill in Engineering Simulation Tools

For more information, feel free to contact ASHRAE official email ashrae_iit@iit.edu



Interested in Joining

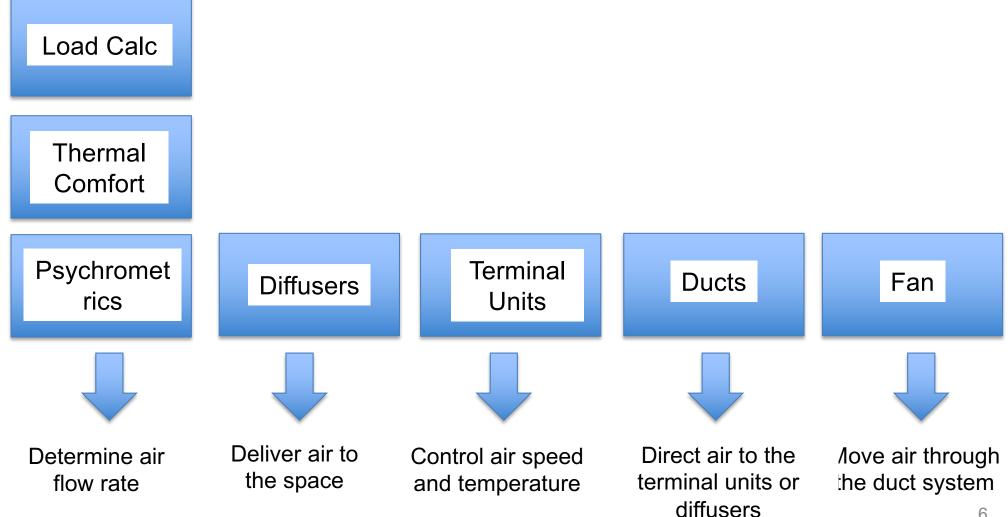
Lunch will be provided!

HOMEWORK / PROJECT / EXAM

RECAP

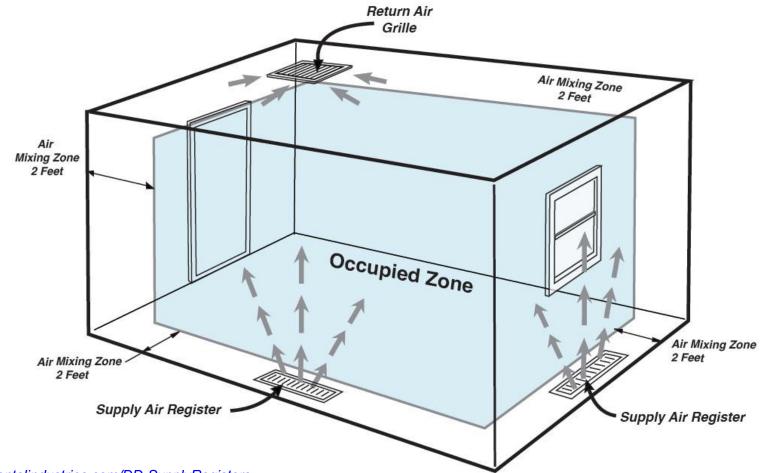
Recap

 There are a couple of components required for the design of an air distribution

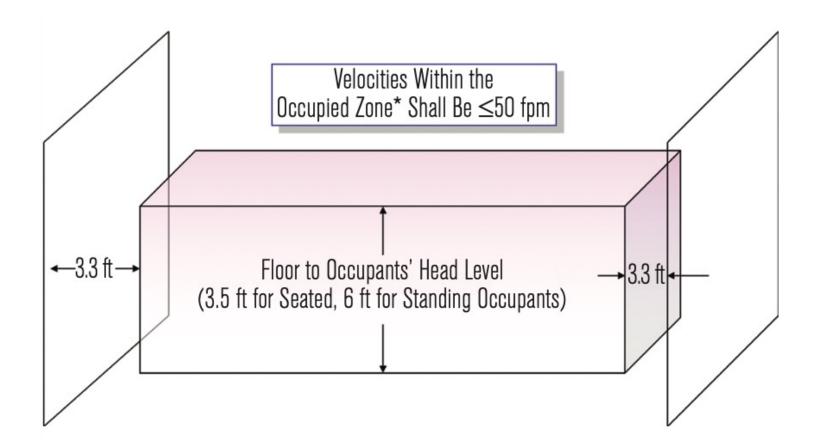


Recap

- We are interested in the occupied zone (or breathing zone):
 2 feet from any wall
 - □ 6 feet from the floor

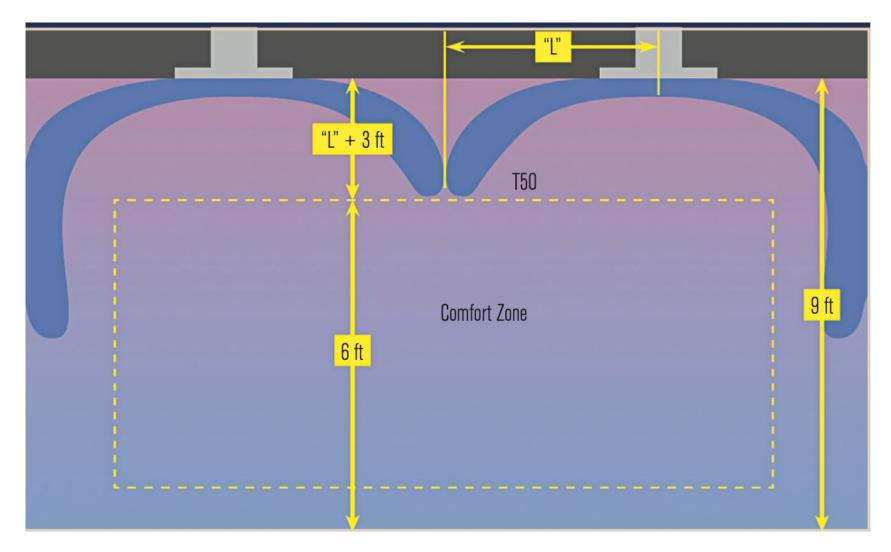


• The definition of the occupied zone can vary:

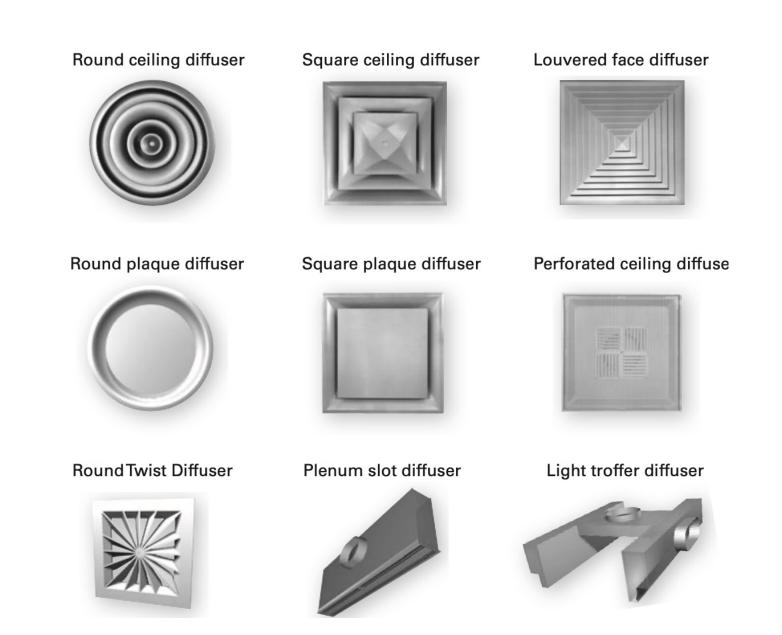


Recap

• The definition of the occupied zone can vary:

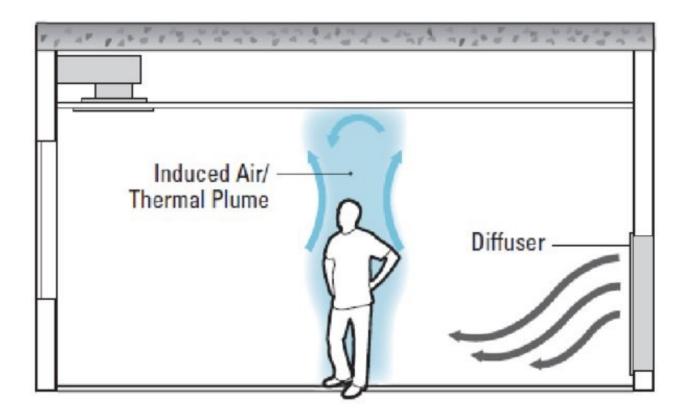


AIR DIFFUSION



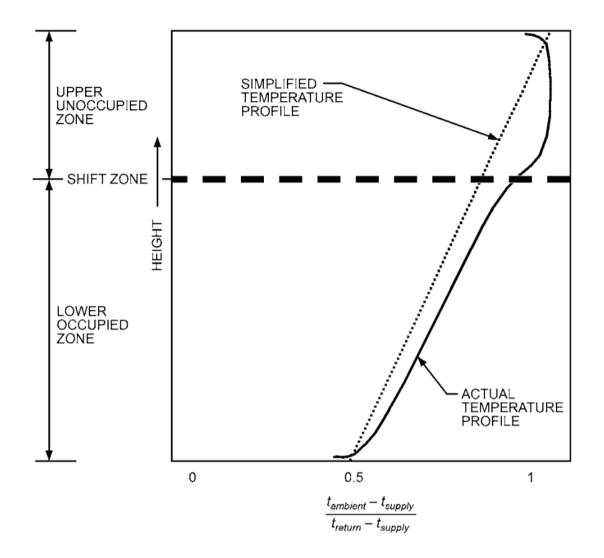


- Thermal Displacement Ventilation (TDV):
 - □ Introduce cool air into a space at low face velocities
 - Let convection and stratification move warm contaminated air up to returns



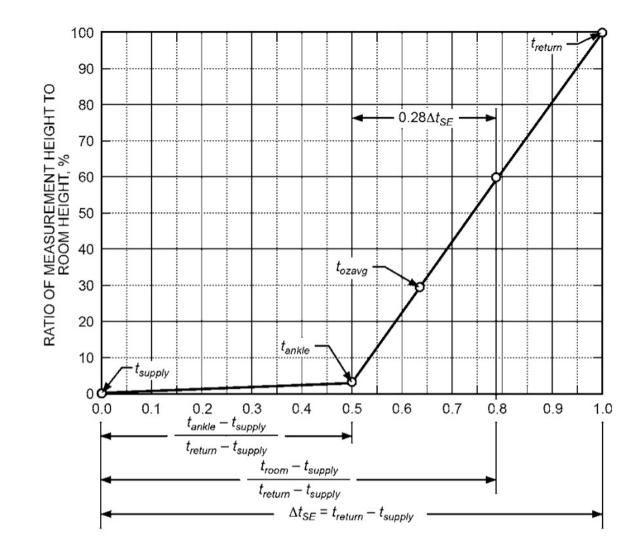
• Thermal Displacement Ventilation (TDV):

□ Temperature profile is usually as follow:



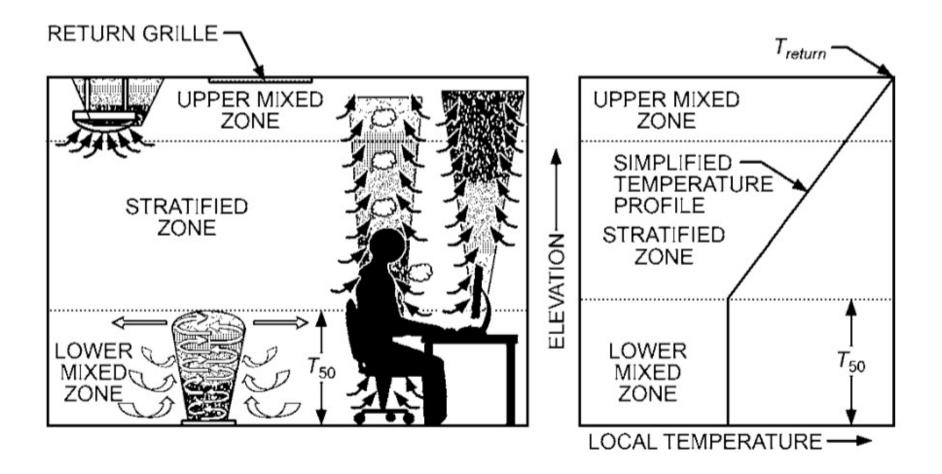
• Thermal Displacement Ventilation (TDV):

□ An example temperature profile for a ceiling height of 10 ft

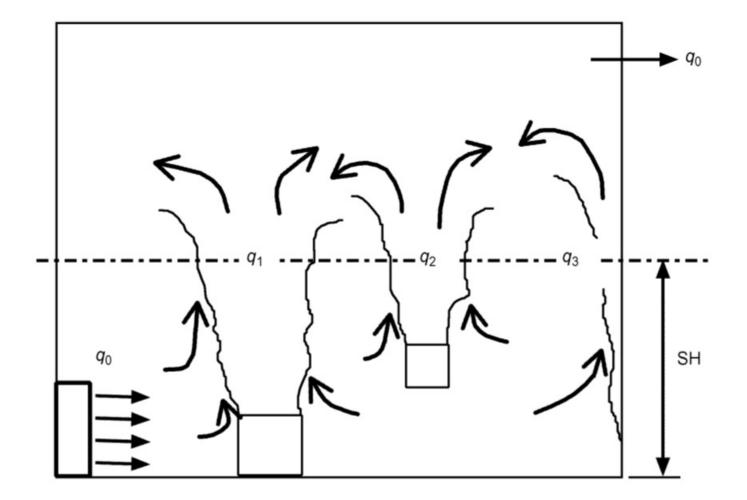


Underfloor Air Distribution (UFAD):

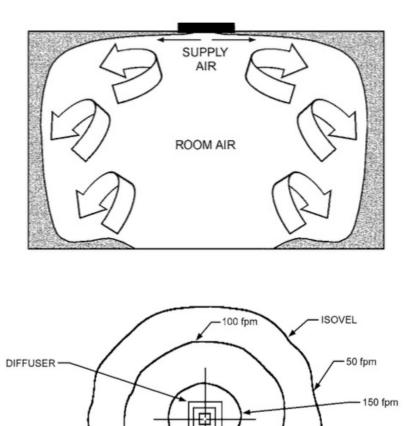
□ Supply airflow is usually vertically at a relatively high velocity



• We need to make sure to consider internal objects

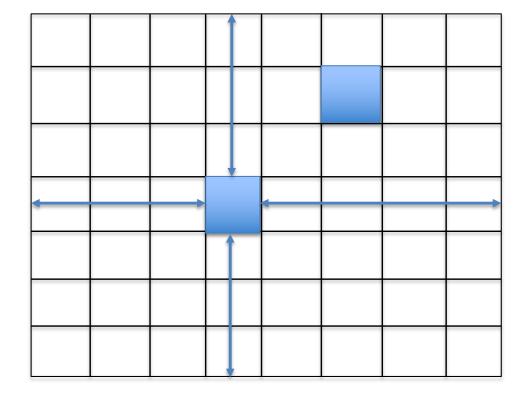


• Remember the secondary flow is important



THROW

- To identify the characteristics length, we need to identify the location of the air outlets
- Identify the maximum achievable throw:



High Sidewall Grille



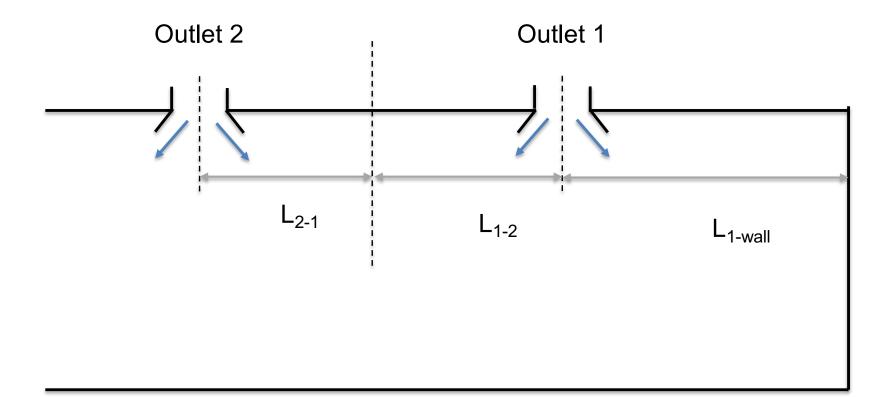
Distance to wall perpendicular to jet

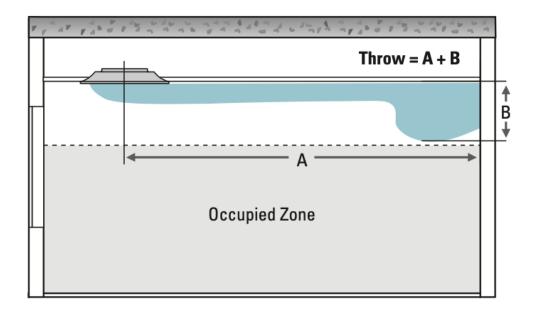
Circular Ceiling Diffuser

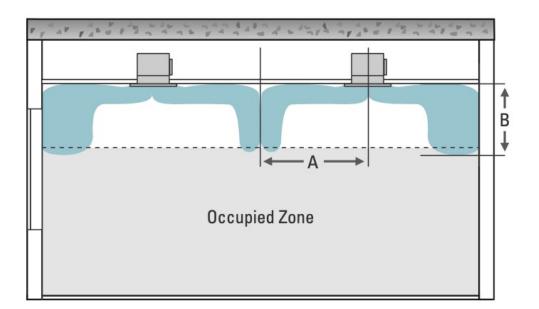


Distance to closet wall or intersecting air jet

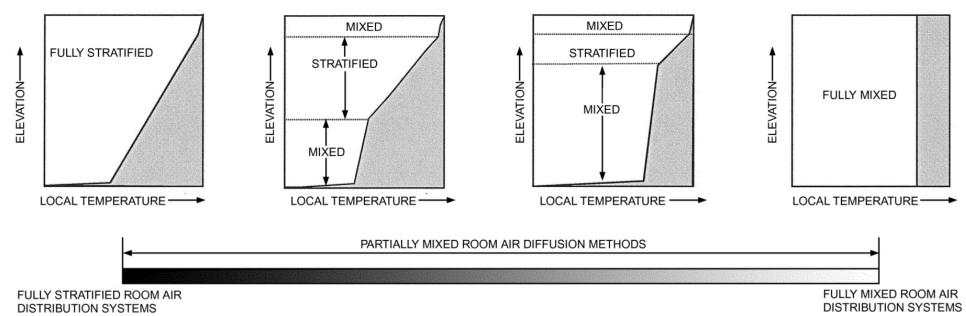
- The available length (L) for outlet #1 is the lesser of L_{1-wall} and L_{1-2} for this case







VERTICAL TEMPERATURE STRATIFICATION PROFILES



EXAMPLES:

- Thermal displacement using low-velocity cool air
- Natural ventilation

EXAMPLES:

- Underfloor air distribution (using room air induction) in cooling operation
- Underseat air distribution (using room air induction) in cooling operation
 Track (seek least least
- Task/ambient cooling (using furniture-based outlets)
- Task/ambient (spot) cooling or heating (industrial applications)

EXAMPLES:

- · Overhead mixed air supply in cooling operation
- · Fan-coil units and unit ventilators
- High-velocity floor-based supply in heating operation

- Fully mixed systems: Room air is fully mixed
 - □ Little or no thermal/pollutant stratification vertically
 - Achieved by supplying large amounts of conditioned air into the room either from overhead or underfloor
 - Conditioning of the space achieved by diluting space air with the supply air
 - Overhead air distribution is an example of this distribution

- Fully stratified systems: Space is fully stratified vertically
 - □ There is a distinct thermal gradient
 - □ Happens through the displacement ventilation systems

- Partially mixed systems:
 - This arises when the occupied space is fully mixed and maintained at a condition distinctly different from the unoccupied zone
 - The unoccupied volume is usually stratified into three zones whose relative lengths may vary
 - Most underfloor air distribution systems (UFAD) are examples of this type

CLASSIFICATION OF AIR DIFFUSION METHODS

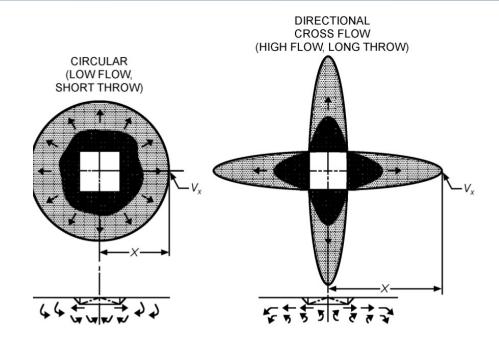
Outlet Classification

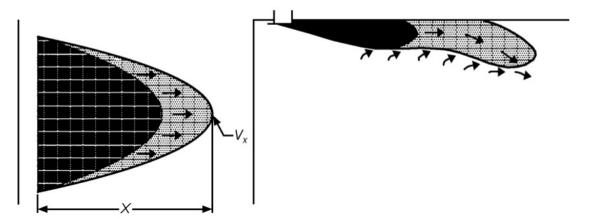
- Group A
 - Outlets mounted in or near the ceiling that discharge air horizontally
 - Outlets discharge horizontally the air that is not influenced by an adjacent surface
- Group B
 - Outlets mounted in or near the floor that discharge air vertically in a nonspreading jet
- Group C
 - Outlets mounted in or near the floor that discharge air vertically in a spreading jet
- Group D
 - □ Outlets mounted in or near the floor that discharge air horizontally
- Group E
 - Outlets that project supply air vertically downward

Group A1 (Ceiling: Horizontal Discharge)

- High sidewall grilles
- Sidewall diffusers
- Ceiling diffusers
- Linear ceiling diffusers

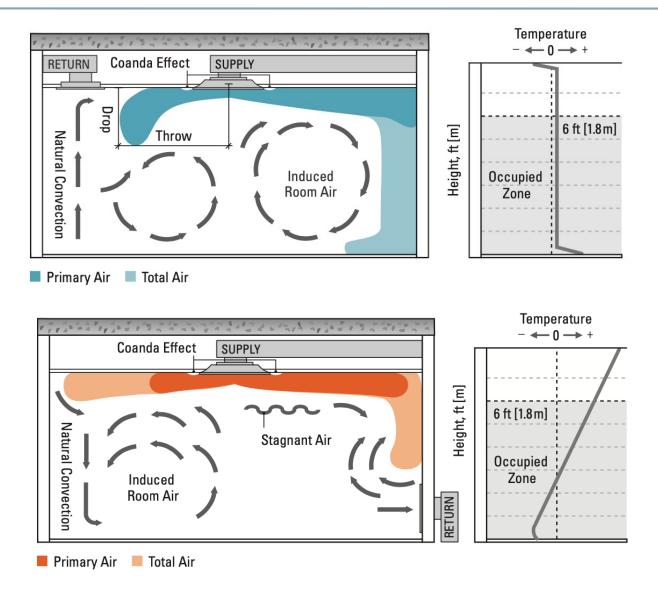
Group A1 (Ceiling: Horizontal Discharge)





Non-Isothermal

Group A1 (Ceiling: Horizontal Discharge)

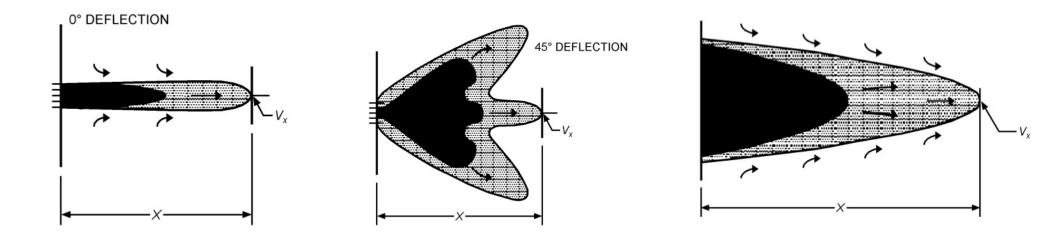


What do you think any of these configurations are good for heating or cooling?

Engineering Guide Air Distribution, Price

Group A2 (Horizontal Discharge Not Surface)

 Similar to A2 with higher stratification for the cooling close to the ceiling



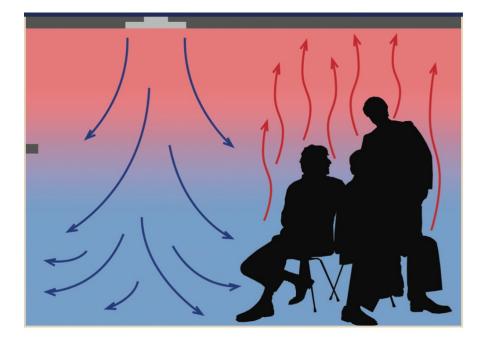
Group A (Ceiling: Horizontal Discharge)

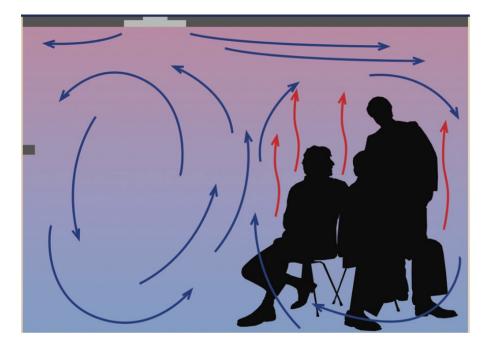
- Often used in mild climates and on the second and succeeding floors of multistory buildings
- Not recommended for cold climates or with unheated floors
- Because the primary air project radially in all directions, the rate of entrainment is large, causing the high-momentum jet to diffuse quickly
- Ceiling diffuser allow handling larger quantities of air at higher velocities than other types

Group A (Ceiling: Horizontal Discharge)

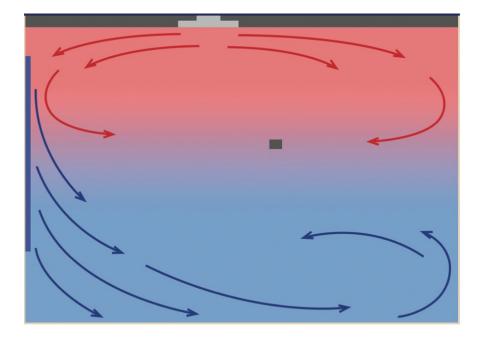
- Quite effective for cooling
- Not recommended for heating unless the floor is above a heated space
- Linear diffusers fall into this group
 - Good for cooling
 - □ Require a supplemental heating system

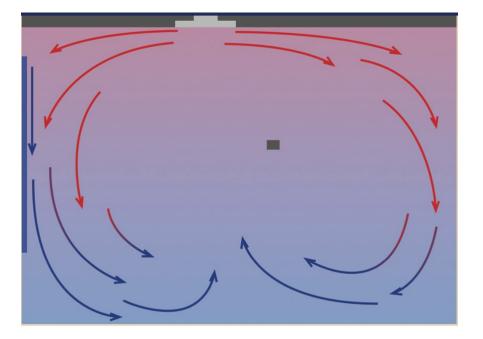
Group A (Ceiling: Horizontal Discharge)





Group A (Ceiling: Horizontal Discharge)





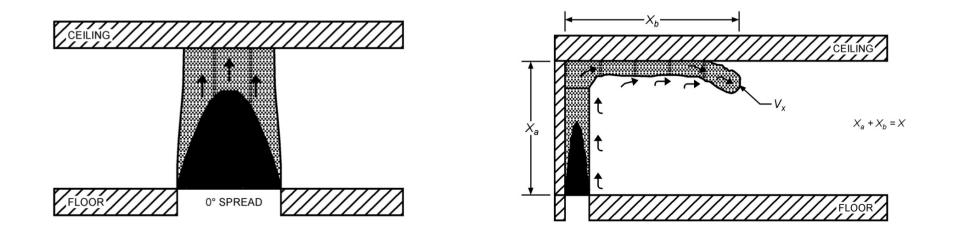
$$\Delta T > 15 \text{ °F} (8.3 \text{ °C})$$

 $V \sim 150 fpm (0.75 \frac{m}{s})$

$$\Delta T < 15 \,^{\circ}\text{F} \,(8.3 \,^{\circ}\text{C})$$

- Floor registers
- Baseboard units
- Low sidewall units
- Linear type grilles

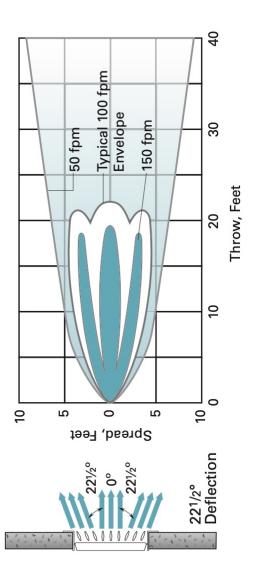
- Satisfactory performance for cooling
- Less desirable for heating because a larger stratified zone will usually occur

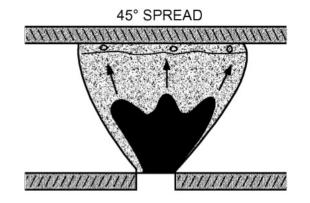


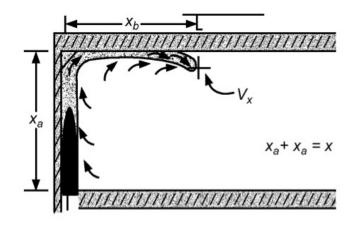
Outlet in or near floor, non spreading linear jets

4°F (2°C) Acceptable to 85% of occupants

- Floor diffusers
- Sidewall diffusers
- Linear type diffusers









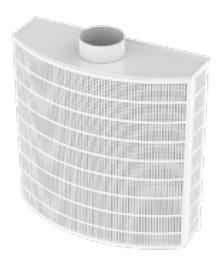


- Usually assume superior for heating applications, especially:
 The floor is located over an unheated space or a slab
 Considerable glass area exists in the wall
- Diffusers with a wide spread are usually best for heating because buoyancy tends to increase the throw
- Similarly, these diffusers are not good for cooling application because the throw may not be adequate for mixing
- Sometimes diffusers are available to change depending on the season

Group D (Horizontal Discharge)

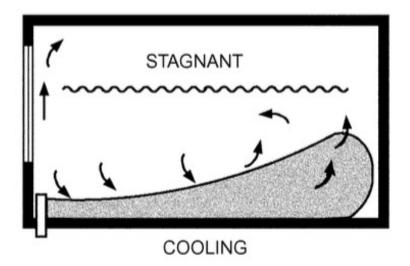
 They are good for displacement ventilation to remove large amount of contaminants from a space

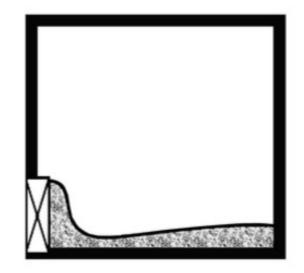




Group D (Horizontal Discharge)

- Baseboard registers
- Low-sidewall registers





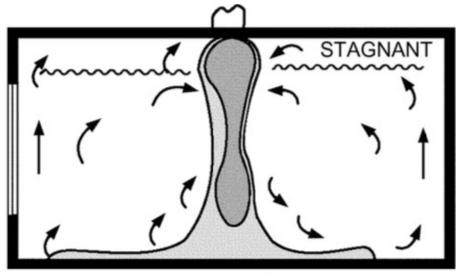
Outlet Group D (High Velocity) - (Nonisothermal)

Outlet Group D (Low Velocity) -Nonisothermal

Group E (Vertical Discharge)

- These outlets are used in partially stratified systems
 - □ Ceiling diffusers
 - □ Linear grilles
 - □ Sidewall diffusers
- These outlets use low discharge velocities
- In mixed systems, they use higher discharge velocities
- Are used for a special applications such as cooling large glasses

Group E (Vertical Discharge)



COOLING

Outlet Group E (High Velocity) -(Nonisothermal)

Outlet Group E (Low Velocity) -Nonisothermal

Return Air & Exhaust Grilles

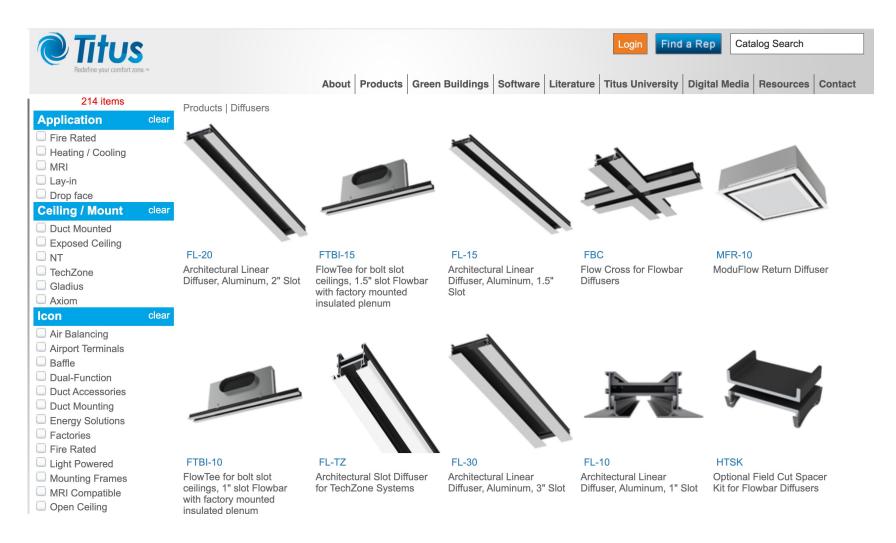
- Less important than diffusers since velocity decreases significantly as the area increases
- Their location has little effect on room air motion (Supplies create the air flow patterns)
- Both supplies and returns should not be blocked by any objects
- The location of returns might be important when there is any pressure imbalance

Return Air & Exhaust Grilles

- From energy standpoint, it is economical to return the coolest air to the heating coil and warmest air to the cooling coil (Stagnation region)
- However, careful consideration is required for high ceiling, atrium, and large vertical glass surfaces
- Always, make sure to avoid short circuiting of supply air

Supply and Return Options

 Please review some diffuser, grille, and register options for the next lectures



Supply and Return Options

 Please review some diffuser, grille, and register options for the next lectures



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REVISITING ASHRAE 62.1

Revisiting ASHRAE 62.1

- ASHRAE 62.1 also provides:
 - Criterion for reduction in the outdoor delivery rate when recirculated air is treated
 - Criterion for variable ventilation rates when the air volume in the space can be used to dilute contaminants
- Many municipals have their own prescribed rates that can differ from ASHRAE
- ASHRAE 62.1 notation uses airflow rate as V

Revisiting ASHRAE 62.1

• Air change rate:

$$ACH = \frac{Volumetric\ flow\ rate}{Volume}\ (or\ \frac{60Q}{V}\ using\ cfm)$$

• Time Constant:

$$\tau = \frac{V}{Q} = \frac{m}{\dot{m}}$$

- □ *ACH*: Air change rate (air changes/hour)
- □ Q: Volumetric flow rate ft³/hr
- \Box V: Room volume ft³
- \Box τ : Time constant
- \square m: Mass of air in the space
- \Box \dot{m} : mass flow rate of outdoor air

CLASS ACTIVITY

Class Activity

- Calculate the volumetric flow rate (in CFM and $\frac{ft^3}{hr}$) and time constant for the following condition:
 - □ Office area: 800 ft²
 - □ Office height: 10 ft

Class Activity

• Solution:

□ Office areas usually have an ACH between 2 to 6

$$\tau = \frac{1}{2 \, [1/hr]} = 0.5 \ hour$$

$$2 = \frac{Q}{(800 \times 10)}$$

$$Q = 16,000 \frac{ft^3}{hr} = 16,000 \times \frac{ft^3}{hr} \times \frac{1 hr}{60 minutes} = 266 CFM$$

Air Change Terms

 Ventilation effectiveness provides measure of an air distribution system's ability to remove an internally generated pollutant

 Age-of-air (θ): Time in which some quantity of outside air has been in a particular building or space (hr)

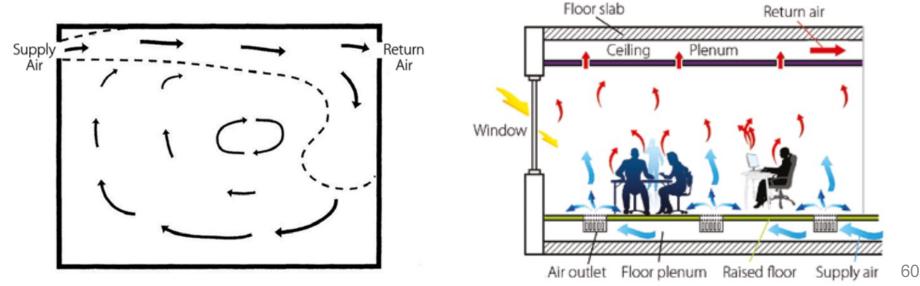
Outdoor Air Requirement

- The per person rate is to dilute the contaminants the occupants create
- The per space rate is related to dilute the contaminants created by furnishings and non-occupant activities
- Calculate the zone outdoor air flow rate from the breathing zone air flow rate

$$V_{oz} = \frac{V_{bz}}{E_z}$$

Ventilation Effectiveness

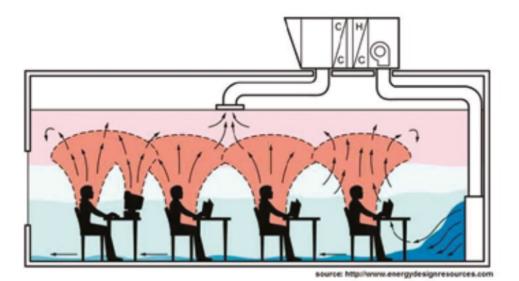
- The zone effectiveness depends upon
 - □ The location of the supply air diffusers
 - □ The location of he return air vents
 - □ Heating or cooling mode
- All three of these factors go into determining how much mixing takes place



Ventilation Effectiveness

• It is better to use the ASHRAE 62.1 table, but common ventilation effectiveness are calculated as:

Heating/Cooling	Supply	Return	Ez
Cooling	Overhead	Overhead	1.0
Heating	Underfloor	Opposite	1.0
Heating	Overhead	Overhead	0.8
Heating	Displacement	Overhead	0.7



MECHANICAL DRAWINGS

 Please, see Chapter 38 of ASHRAE Fundamentals: Abbreviations and Symbols. This chapter entails:

□ Abbreviations

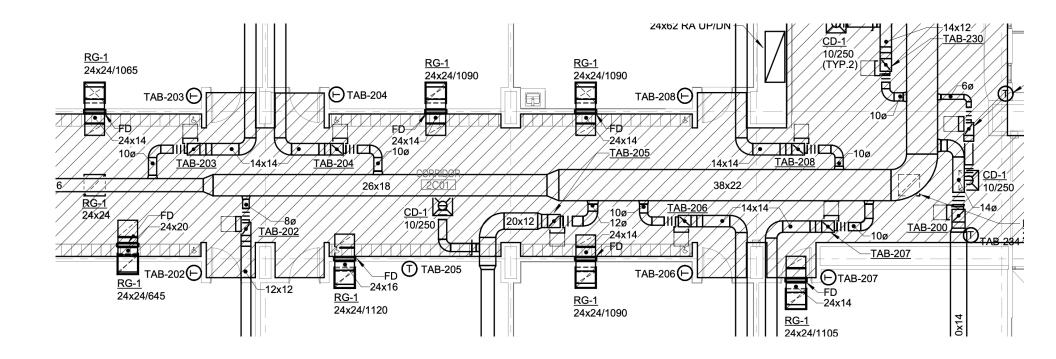
- □ Letter symbols
- □ Graphical symbols

CHAPTER 38

ABBREVIATIONS AND SYMBOLS

Abbreviations for Text, Drawings, and Computer Programs	37.1
Letter Symbols	
Dimensionless Numbers	
Mathematical Symbols	
Subscripts	
Graphical Symbols for Drawings	
Piping System Identification	

 A good approach starts drawing on the existing architectural drawings:



 Look for the Mechanical Symbols, Notes, and Abbreviations page (usually M.000, M.00.00, …)

·
<u>LOCATION:</u> 3101 South Wabash Ave. Chicago, IL 60616 <u>PROJECT:</u> IIT BAILEY HALL
DRAWING TITLE MECHANICAL SYMBOLS, NOTES & ABBREVIATIONS
DRAWING NUMBER: M.000
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VENTILATION SYM	ION SYMBOLS		MBOLS
SYMBOL	DESCRIPTION	SYMBOL	DESCRIPTION
	NEW DUCTWORK		PRESSURE GAGE & COCK
\square	DUCT SECTION - SUPPLY UP	+>+	STRAINER
$\mathbb{Z} \mathbb{Z} \mathbb{N} \mathbb{Z} \mathbb{Z}$	DUCT SECTION - SUPPLY DOWN		STRAINER WITH BLOW OFF VALVE
	DUCT SECTION - RETURN UP		THERMOMETER
	DUCT SECTION - RETURN DOWN	Р/Т	PRESSURE/TEMPERATURE SENSOR
	DUCT SECTION - EXHAUST UP	[CAP
	DUCT SECTION - EXHAUST DOWN		UNION
	DUCT SECTION - OUTSIDE AIR UP		
$\left \right>$	DUCT SECTION - OUTSIDE AIR DOWN		FLEXIBLE CONNECTION
$R \longrightarrow f$	INCLINED RISE WITH RESPECT TO AIRFLOW	₩	PRESSURE REDUCING VALVE
	INCLINED DROP WITH RESPECT TO AIRFLOW	\$*	RELIEF VALVE
	FLEXIBLE CONNECTION TO EQUIPMENT		GATE VALVE
	LOUVER & SCREEN WXD GROSS OPENING	6	BUTTERFLY WHEN VALVE IS 4" OR MORE BALL VALVE WHEN VALUE IS 3" OR LESS
	FLEXIBLE DUCT	—á—	GAS COCK
	VOLUME DAMPER WITH QUADRANT LOCKING	——ö——	PRESSURE/TEMPERATURE TAP
	MOTORIZED DAMPER	0 T	UNIT HEATER VERTICAL
	SPLITTER DAMPER	۵ <u> </u> د	UNIT HEATER HORIZONTAL PIPE DOWN
BDD	BACKDRAFT DAMPER (GRAVITY)	·	PIPE UP
▲ FD	FIRE DAMPER, SLEEVE & ACCESS DOOR		NEW PIPING
	AIR EXTRACTING VANES		PIPING ASSEMBLY (SEE DETAIL)
XXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXX	TURNING VANES, DOUBLE THICKNESS AIRFOIL TYPE	v	AIR VENT
SYSTEM RISER NO.	RISER MARK		EQUIPMENT (SPECIFIED BY TAG BELOW)
Ū	THERMOSTAT (G) W/ GUARD		EQUIPMENT TAG
s	SENSOR	SD	SMOKE DETECTOR
Э	HUMIDISTAT	FSD	FIRE / SMOKE DETECTOR

 Look for the Mechanical Symbols, Notes, and Abbreviations page (usually M.000)

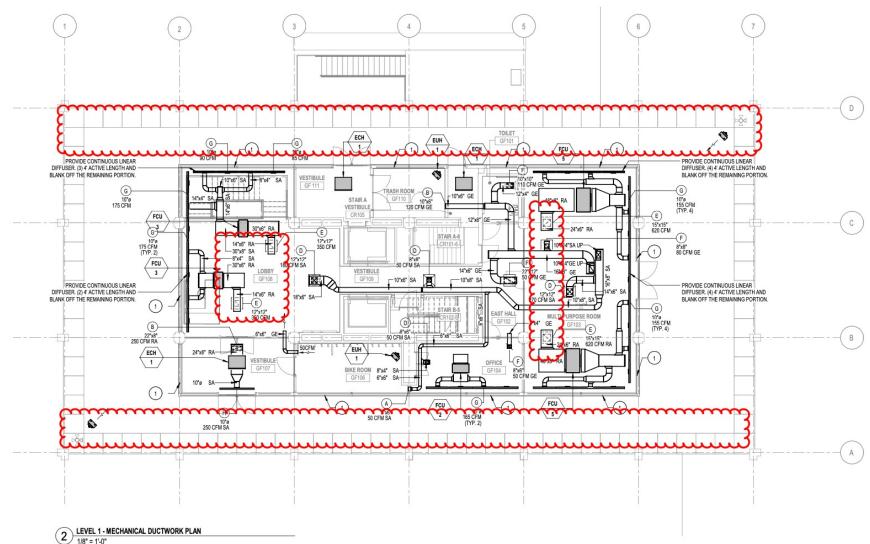
UCTWORK		MECHANICAL PIPING	MECHANICAL PIPING				
ABBREVIATION	SYSTEM NAME	ABBREVIATION	SYSTEM NAME				
FLUE	BOLIER FLUE	CD	CONDENSATE DRAIN				
CAI	COMBUSTION AIR INTAKE	NGMP	NATURAL GAS MEDIUM PRESSURE				
DE	DRYER EXHAUST	RFGG	REFRIGERATION GAS				
GE	EXHAUST AIR	RFGL	REFRIGERATION LIQUID				
OA	OUTSIDE AIR						
RA	RETURN AIR						
SA	SUPPLY AIR						
TE	TOILET EXHAUST						

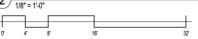
ABBREVIATION DESCRIPTION AIR CONDITIONER AC: ACH AIR CHANGES PER HOUR ADS AIR DIRT SEPARATOR AFF ABOVE FINISH FLOOR AFG ABOVE FINISHED GRADE AFMS AIR FLOW MONTIORING STATION AFU ANNUAL FUEL EFFICIENCY RATIO AHU AIR HANDLER AMP AMPERAGE APD AIR PRESSURE DROP AS AIR SEPARATOR ATU AIR TERMINAL UNIT AUX. AUXILLARY в BOILER BAS BUILDING AUTOMATION SYSTEM BLDG BUILDING BTU BRITSH THERMAL UNITS BTU/H BRITISH THERMAL UNITS PER HOUR CAI COMBUSTION AIR INTAKE CAP CAPACITY CAV CONSTANT AIR VOLUME CAV CABINET UNIT HEATER CC COOING COIL CEH CUBIC FEET PER HOUR CFM CUBIC FEET PER MINUTE CFM GAUGE CHWP CHILLED WATER PUMP CEILING CLG CO2 CARBON DIOXIDE CONN CONNECTION COEFFICIENT OF PERFORMANCE COP CP STEAM CONDENSATE PUMP CRAC COMPUTER ROOM AIR CONDITIONER CT COOLING TOWER CUH CABINET UNIT HEATERS CWP CONDESER WATER PUMP DB DRY-BULB TEMPERATURE DE DRYER EXHAUST DEF DRYER EXHAUST FAN DIA. Ø DIAMETER DN DOWN DOA DEDICATED OUTSIDE AIR FAD EXHAUST AIR DAMPER EAT ENTERING AIR TEMPERATURE EER ENERGY EFFICIENCY RATIO EE EXHAUST FAN EFF EFFICIENCY ESP EXTERNAL STATIC PRESSURE ΕT EXPANSION TANK EUH ELECTRIC UNIT HEATERS EWT ENTERING WATER TEMPERATURE ΕX EXISTING

EXHAUST

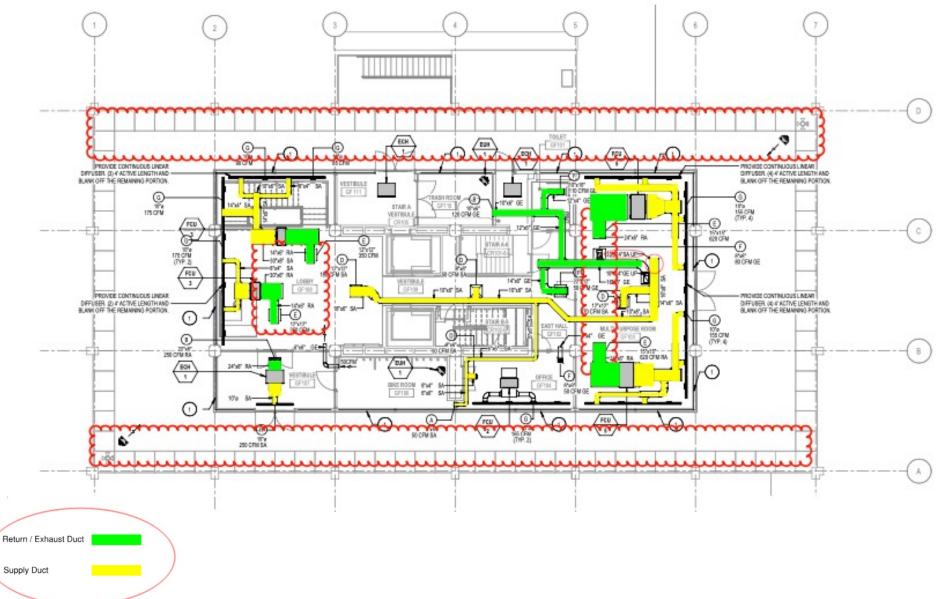
EXH

Mechanical Drawings: Air distribution





Mechanical Drawings: Air distribution



Mechanical Drawings: Air distribution

						VENTIL	ATION SCHEDUL	E							
ROOM			CLASSIFICATION		CODE REQUIREMENTS				ACTUAL PROVIDED				FAN SYSTEM		
	ROOM NO.	OM NO. ROOM NAME		S.F. AREA	MECHANICAL VENTILATION (CFM)		NATURAL LIGHT & VENT (SF)		MECHANICAL VENTILATION (CFM)		NATURAL LIGHT & VENT (SF)				REMARKS
					SUPPLY CFM	EXHAUST CFM	GLASS AREA	VENT AREA	SUPPLY CFM	EXHAUST CFM	GLASS AREA	VENT AREA	SUPPLY	EXHAUST	
LOWER LEVEL	BA101	TOILET ROOM	TOILET ROOMS	63	0	130	-	-	0	130	-	-	-	ERV-1	
	CR101	STAIR A	STAIR	77	-	-	-	-	0	0	-	-	-	-	
	CR103	ELEVATOR PIT	STORAGE INACTIVE	49	0	0	-	-	0	0	-	-	-	-	
	CR104	ELEVATOR	ELEVATOR	66	-	-	-	-	0	0	-	-	-	-	
	HA101	EAST HALL	CORRIDORS	125	NR	NR	-	-	50	50	-	-	ERV-1	ERV-1	
	HA102	VESTIBULE	CORRIDORS	171	NR	NR	-	-	50	50	-	-	ERV-1	ERV-1	
	HA103	LOUNGE	LOUNGE	1406	1410	2110	-	-	1420	2110	NONE	NONE	ERV-1	ERV-1	
	LL101	TRANSFORMER ROOM	STORAGE INACTIVE	173	0	0	-	-	0	0	-	-	-	-	
	LL102	CONDENSER ROOM	STORAGE INACTIVE	460	0	0	-	-	0	0	-	-	-	-	
	LL103	ATS ROOM	STORAGE INACTIVE	39	0	0	-	-	0	0	-	-	-	-	
	LL104	SWITCH ROOM	STORAGE INACTIVE	112	0	0	-	-	0	0	-	-	-	-	
	LL105	BUILDING STORAGE	STORAGE INACTIVE	1915	0	0	-	-	50	50	-	-	ERV-1	ERV-1	
	LL106	JANITOR'S CLOSET	JANITOR'S CLOSET	45	0	90	-	-	0	90	-	-	-	ERV-1	
	LL107	IT CLOSET	STORAGE INACTIVE	29	0	0	-	-	0	0	-	-	-	-	
	LL109	MULTI-PURPOSE ROOM	OFFICE	278	170	85	-	-	240	85	NONE	NONE	ERV-1	ERV-1	
	LL110	LAUNDRY	LAUNDRIES (PUBLIC)	274	415	415	-	-	415	420	-	-	ERV-1	ERV-1	
	LL111	LIGHT WELL	EXTERIOR	70	-	-	-	-	0	0	-	-	-	-	
	LL112	FITNESS CENTER	GYMNASIUM	378	760	570	-	-	795	570	12.83	NONE	ERV-1	ERV-1	
	LL113	PUMP ROOM	STORAGE INACTIVE	809	0	0	-	-	0	0	-	-	-	-	
	LL115	STORAGE	STORAGE INACTIVE	172	0	0	-	-	0	0	-	-	-	-	
	ST101	STAIR C	STAIR	171	-	-	-	-	0	0	-	-	-	-	
	ST102	ACCESSORY STAIR	STAIR	164	-	-	-	-	0	0	-	-	-	-	
LEVEL 1	CR101	STAIR A	STAIR	77	-	-	-	-	0	0	-	-	-	-	
	CR102	STAIR B	STAIR	85	-	-	-	-	о	0	-	-	-	-	

VENTILATION SCHEDULE

ROOM NAME		MIN. VENTILATION RATE CFM/SQ. FT.		FLOOR	ORDINANCE REQUIREMENTS		ACTUAL PROVIDED		SYSTEM SERVED BY		
	ORDINANCE CLASSIFICATION			AREA	SUPPLY	SUPPLY EXHAUST		EXHAUST	AIR HANDLING	EXHAUST FAN	REMARKS
		SUPPLY	EXHAUST	SQ. FT.	CFM	CFM	CFM	CFM	UNIT	EAHAUST FAN	
FIRST FLOOR											
COLLEGE & CAREER CENTER 1015	CLASSROOM	1.5	0.75	994	1,491	746	1,500	1,500	AHU-1	AHU-1	
CORRIDOR N112	CORRIDOR	0.0	0.0	301	0	0	50	50	AHU-1	AHU-1	