# CAE 464/517 HVAC Systems Design Spring 2023

### February 16, 2023

# Air distribution systems: principles and air jet patterns

Built Environment Research @ IIT ] 🗫 🕣 🍂 🛹

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### ANNOUNCEMENTS

#### Announcements

• Do not forget about the Q&A file:

https://docs.google.com/document/d/1m6ezSI6Bi9wGQcjnaYj iAXY2kzRICPYWkKfayNp5WE

#### Announcements

• Group project: Add your name to the list

https://docs.google.com/spreadsheets/d/1WwM6L1i8SmpTWU 3xYQeNypfnqbloQRglBgCt5E7Kb2s/

#### Announcements

• Adding interesting and daily application of HVAC systems:

https://docs.google.com/presentation/d/15bvvZ0VVm9SgonCzZ 5N07MBvI0YdVRYaph6Z3evveJA/edit#slide=id.p

# AIR DISTRIBUTION (CHAPTERS 20-21 OF THE HANDBOOK)

 How much is the total airflow requirements? Can we use ASHRAE 62.1 to calculate the entire airflow requirement for a building and design the AHUs?

#### **CHAPTER 20**

#### **SPACE AIR DIFFUSION**

Indoor Air Quality and Sustainability	20.2
Terminology	
Principles of Jet Behavior	
Symbols	
Dymoorb	20.0

**R**OOM air distribution systems are intended to provide thermal comfort and ventilation for space occupants and processes.

Local temperature and carbon dioxide  $(CO_2)$  concentration have similar stratification profiles.

#### **CHAPTER 21**

#### **DUCT DESIGN**

BERNOULLI EQUATION	21.1
Head and Pressure	21.2
SYSTEM ANALYSIS	21.2
Pressure Changes in System	21.5
FLUID RESISTANCE	21.6
Friction Losses	21.6
Dynamic Losses	21.8
Ductwork Sectional Losses	21.13

FAN/SYSTEM INTERFACE	21.13
MECHANICAL EQUIPMENT	
ROOMS	21.15
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Design Methods	21.22
Industrial Exhaust Systems	

**CHAPTER 9** 

#### INTRODUCTION TO MIXING VENTILATION

#### **Price Industries Handbook**

https://www.priceindustries.com/edu cation/engineershandbook

#### **5.0 PROFESSIONAL DEVELOPMENT HOURS**

This chapter provides a concise approach to the proper selection of air distribution outlets with an emphasis on occupant comfort, air quality, and energy conservation.

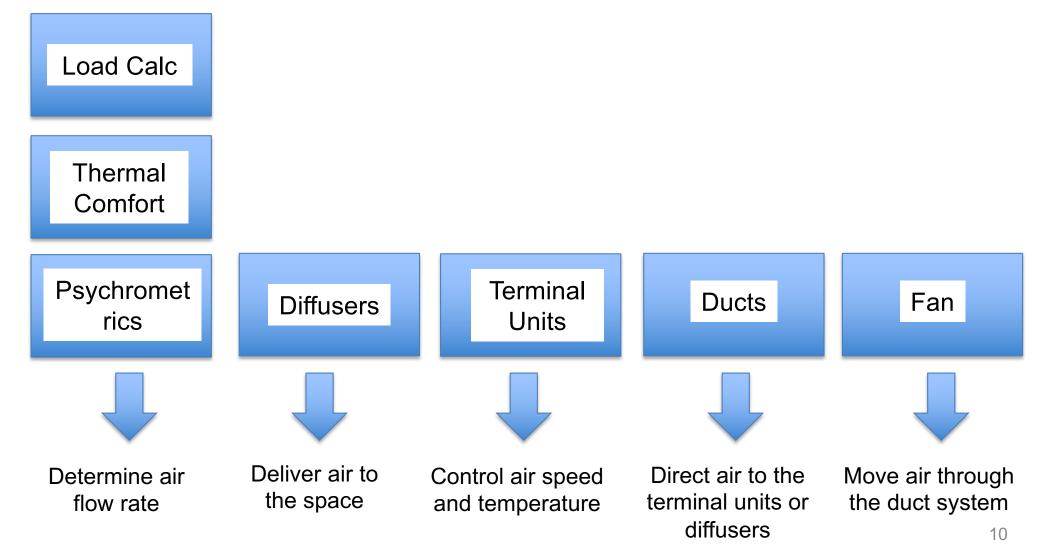


The design and selection of air distribution equipment in today's buildings presents one of the more unique challenges for mechanical designers. Unlike other mechanical equipment required for these environmental systems, the air distribution equipment selection must combine a proper choice of engineered products, efficiently providing conditioned air to the space while adding architectural features that complement the interior design.

Air outlet selection and application is no less important than any other facet of the HVAC system. Much time and money can be spent on the design and purchase of mechanical equipment, controls, piping, and ductwork, but if the air outlets are improperly applied or selected, final system performance will be compromised.

With today's emphasis on occupant comfort, air quality and energy conservation, the proper selection of air outlets is critical. It is the intent of this Mixing Ventilation chapter to provide a concise approach to the proper selection of air distribution outlets.

• There are a couple of components required for the design of an air distribution



- Aim to provide a proper room temperature and humidity throughout room by creating air motion
- Avoid drafts and unwanted secondary air flow and noises
- Consider appropriate:
  - □ Selection type and location of diffusers
  - □ Selection and location of return air grilles
  - □ Size of air distribution equipment
  - □ Calculation of pressure losses
  - Noise level

• For "occupied spaces" the goal is to satisfy thermal comfort requirements with a proper combination of:

□ Temperature

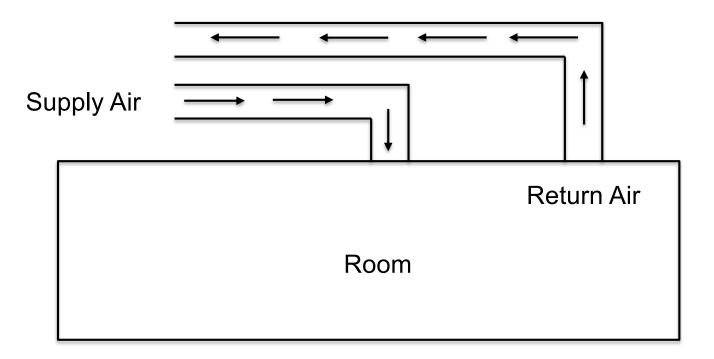
□ Humidity

□ Air motion

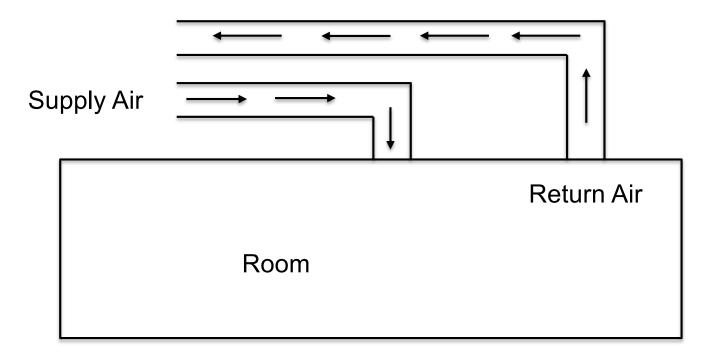
- For "unoccupied spaces" the goal is to satisfy the equipment environmental conditions with a proper air circulations with a proper combination of:
  - □ Temperature
  - □ Humidity

# **AIR DISTRIBUTION TERMINOLOGY**

 Supply Air (SA): Air entering a space that has been directly delivered from a fan, blower, or air handler. They get distributed through air delivery devices

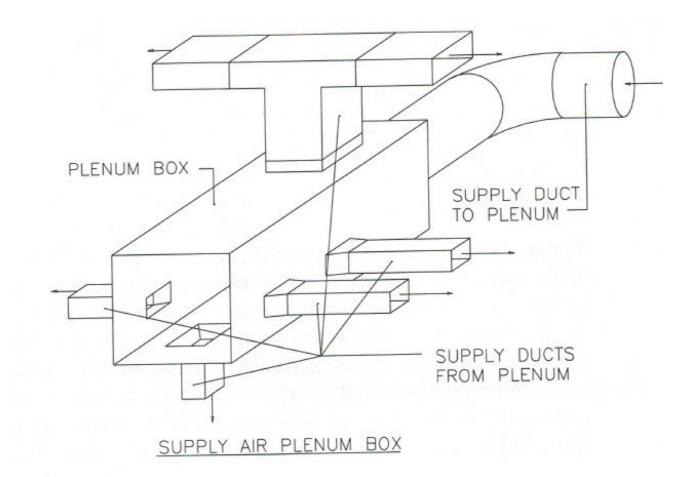


 Return Air (RA): Air removed from a space with the intent or partial or total recirculation. Typically escape through openings or air devices named "register"

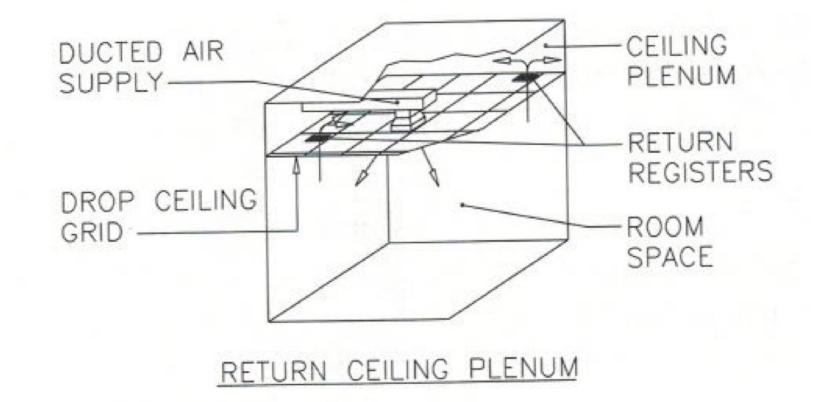


- RA removal options:
  - Hard Ducting: A closed system of ducts to direct the air back to the air handler
    - Return air in constant motion
    - No option to store the return air
    - Expensive air return option
    - Code requires that

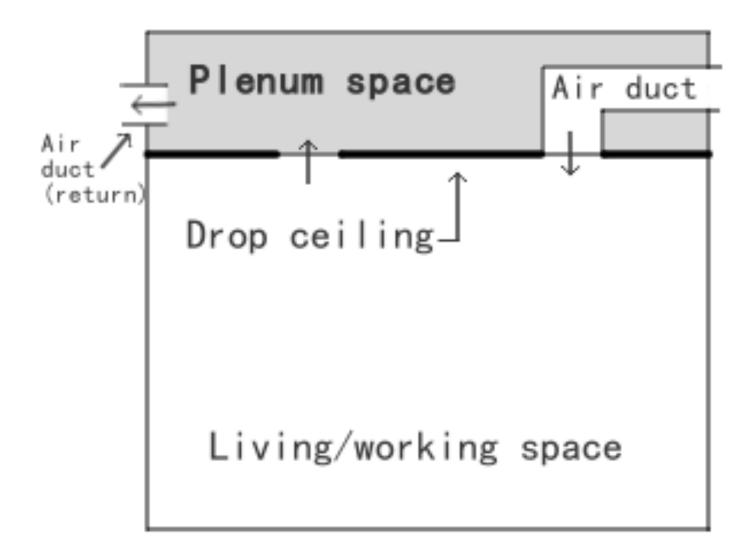
- RA removal options
  - Duct Plenum or Plenum Box: A plenum is a large enclosure that is used to store air until is needed



- RA removal options:
  - Ceiling Plenum: Similar to the duct plenum except the space located above the ceiling and below the above floor is used to store air



• Ceiling plenum options:

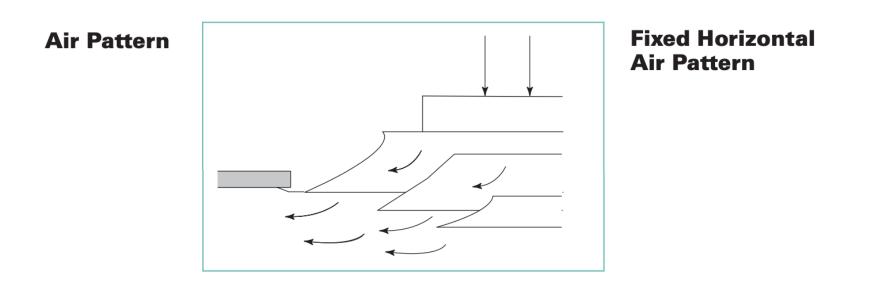


• Ceiling plenum options:



- Let's look at three definitions in addition to supply air (SA):
  - Exhaust Air (EA): Any air that is removed directly from a space and derived to the outside of the building. This air is not circulated
  - Make-up Air (MA): Any air that acts as an air supply from a source outside the space which is not forced into the space. It is not filtered, forced, or treated
  - Transfer Air (TA): Any air that acts as a supply to a space that comes from adjacent spaces

- Diffusers: Direct air after the ductwork:
  - □ Volumetric adjustment: Use dampers
  - Deflection: Use vanes
  - □ Diffusion: Spread air as passes through the diffusers



- Air register vs. air grille
  - Grille is used mostly for deflections
  - □ Register is a grille plus damper

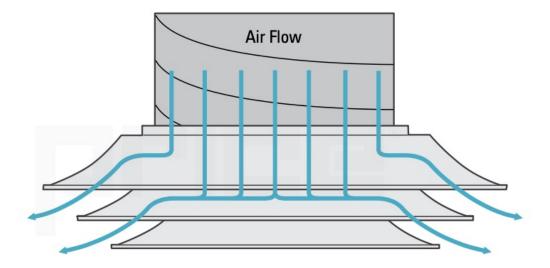


30 in. x 6 in. Return Air Vent Grille, White with Fixed Blades

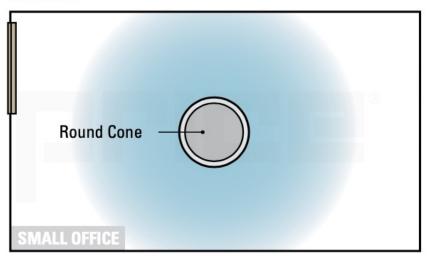


12 in. x 6 in. 3-Way Wall/Ceiling Register

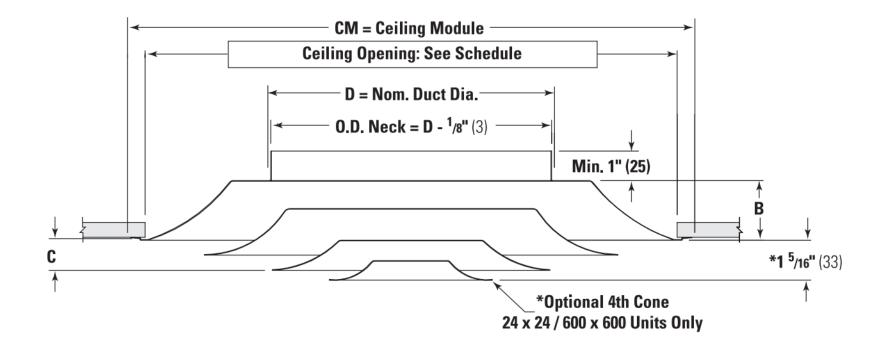
• Celling diffusers cross section:



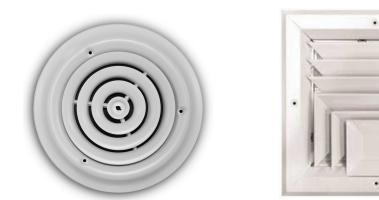
**Plan View** 



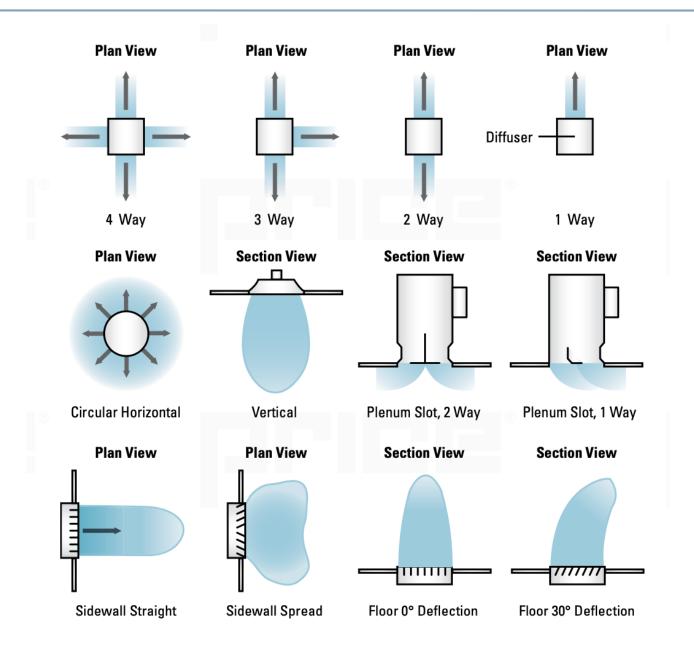
• A few common definitions for the diffuser selection:



- Celling Diffusers can spread air out through:
  - □ 2-way
  - □ 3-way
  - 4-way
  - □ All-way







 Sometimes we need to use Long, Slender Diffusers (LSD) to deliver air over a greater horizontal distance than a ceiling diffusers



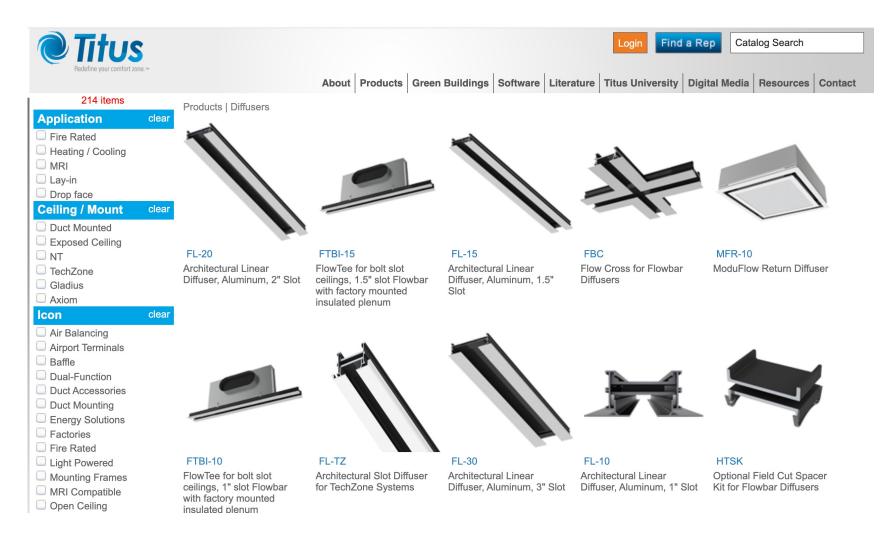


<u>http://www.acutherm.com/product/tlc-linear/</u>

https://www.homedepot.com/p/Everbilt-8-in-x-8-in-4-Way-Square-Ceiling-Diffuser-EA504M-08X08/300539275

https://www.homeessentialsdepot.com/truaire-3-way-aluminum-square-ceiling-diffuser-8-in-x-8-in-3562230-3562230/?gclid=EAIaIQobChMIIJTpv7O24AIVBLnACh1SMwM9EAkYAiABEgKBqPD\_BwE http://grilletech.com.au/product/linear-slot-diffuser/

 Please review some diffuser, grille, and register options for the next lectures



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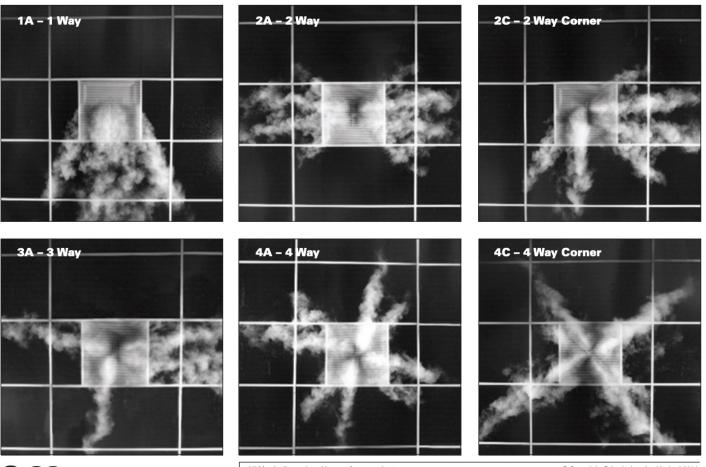


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### **AIRFLOW PATTERNS**

- Air mixing
- □ Leads to space heat transfer and resultant velocity reduction
- $\hfill\square$  Needs to occur outside of the occupied zone

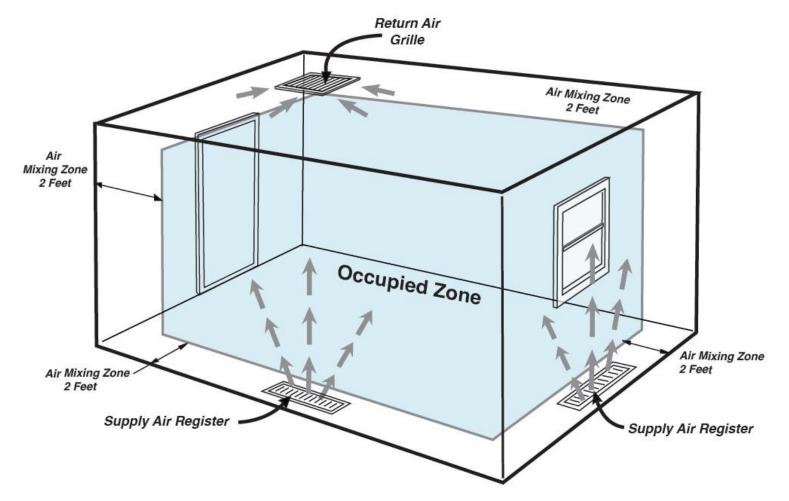


**C-38** 

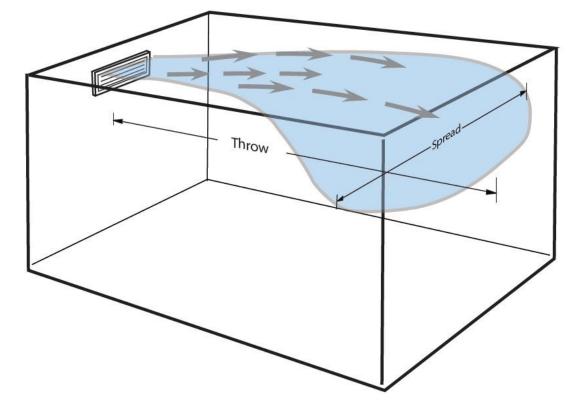
All Metric dimensions () are soft conversion. Imperial dimensions are converted to metric and rounded to the nearest millimeter.

- Occupant thermal comfort occurs through the secondary air flow in the space and not the direct air flow from the outlets
- The goal is to maximize the uniform air temperature and air speed distribution in the occupied zone
- A desirable space air speed is less than 50 fpm (How much is this in SI unit)

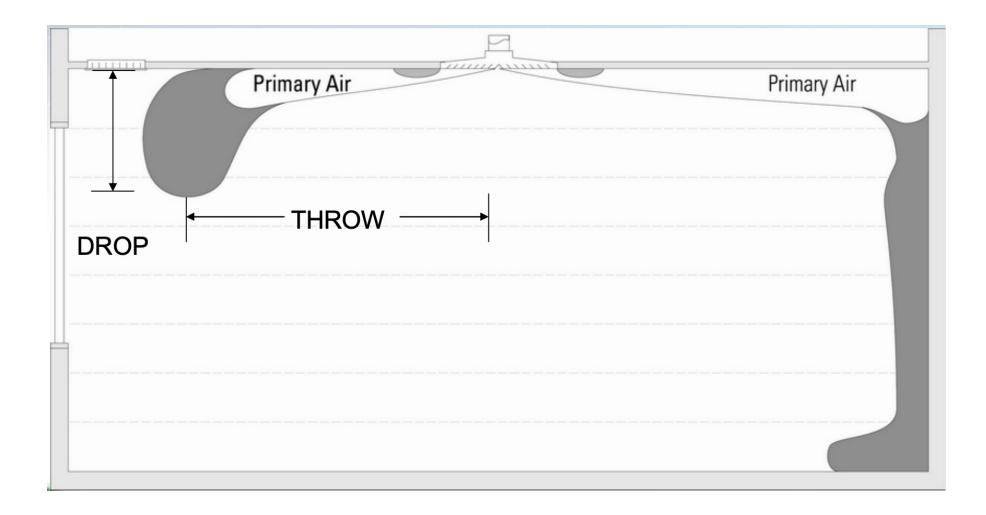
- We are interested in the occupied zone (or breathing zone):
  2 feet from any wall
  - □ 6 feet from the floor



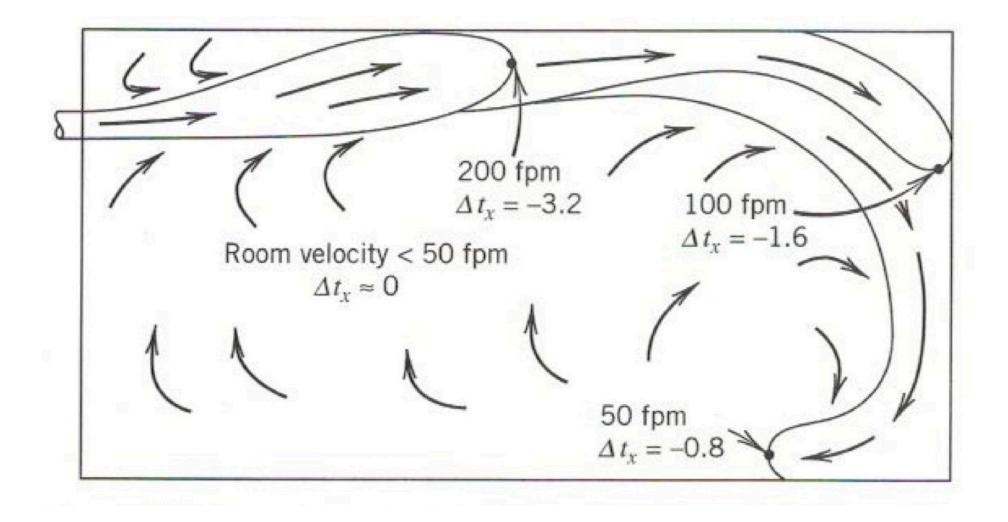
- Two important factors in the design of a register are:
  The distance from the register that the air will travel (Throw)
  - The maximum width the airstream will travel at a corresponding velocity (Spread)
    - Angle of the vanes (0°, 22° and 45°)



#### **Mixed Air Systems**



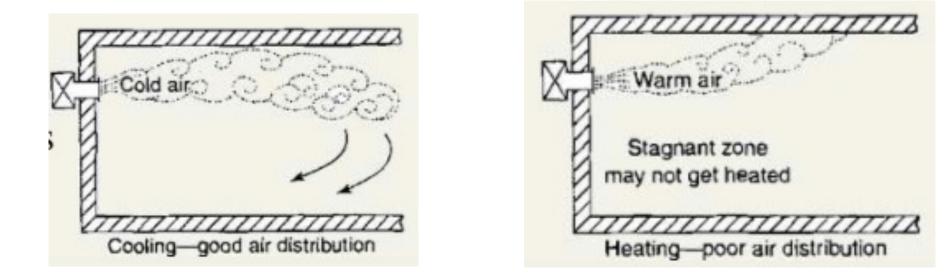
#### Primary, Secondary, and Total Air



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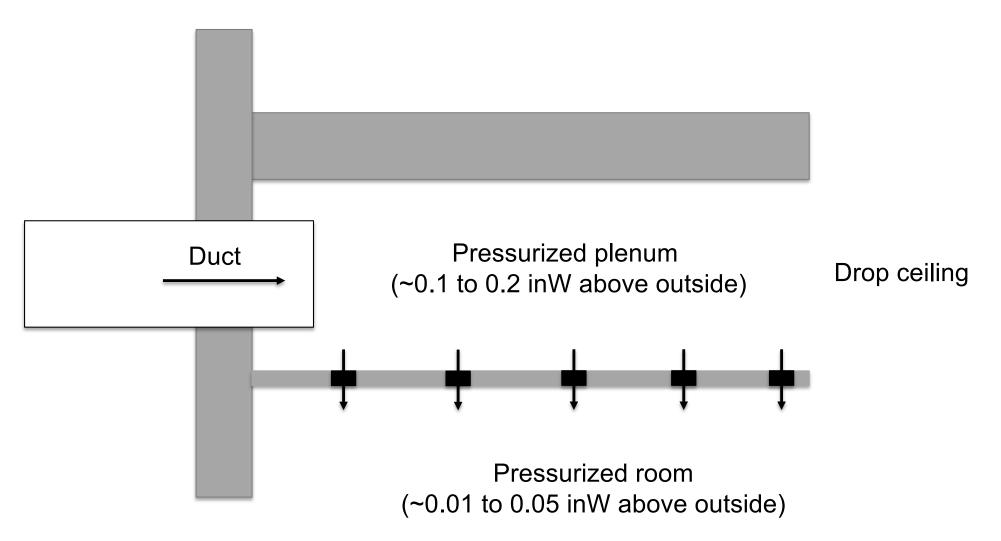
- It is desirable to design a fully mixed system to maintain a constant temperature gradient from the floor to the top of the occupied zone
- Displacement ventilation or underfloor systems not fully mixed

 Location of the air distribution devices has a significant impact on the room flow patterns

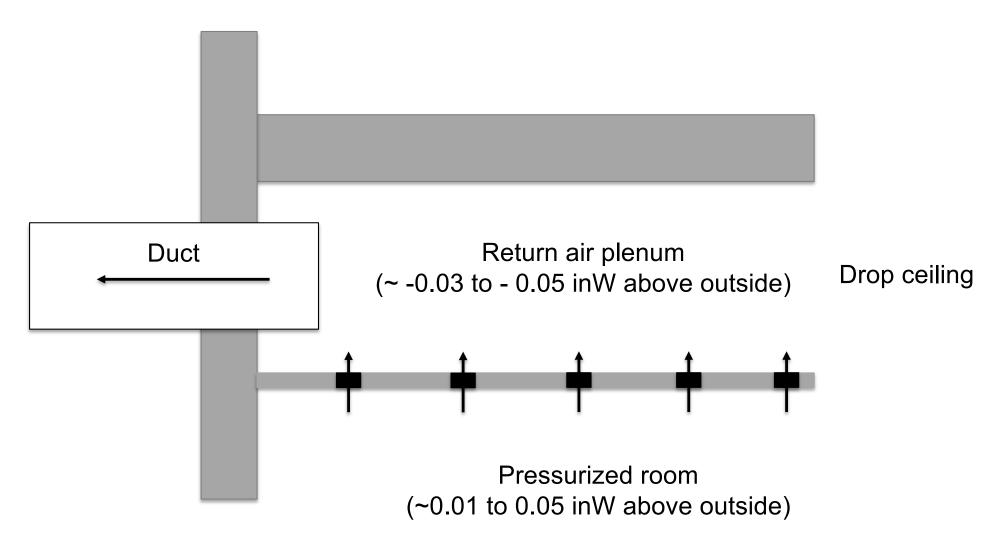


Good air distribution uses:
 Ceiling outlet location for cooling
 Floor or sill location under window for heating

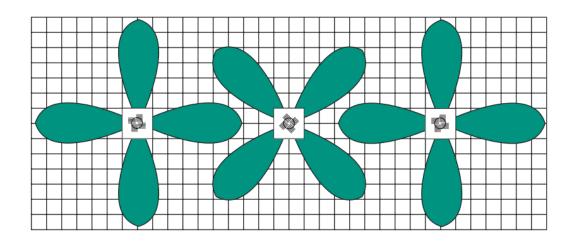
• Ceiling plenum (Supply air):



• Ceiling plenum (Return air):



- For room supply airflow, the major factors are:
  - □ Total room supply airflow quantity
  - □ Room supply air temperature
  - Diffuser type
  - Diffuser throw height (or outlet velocity) or mixing provided by a floor diffuser



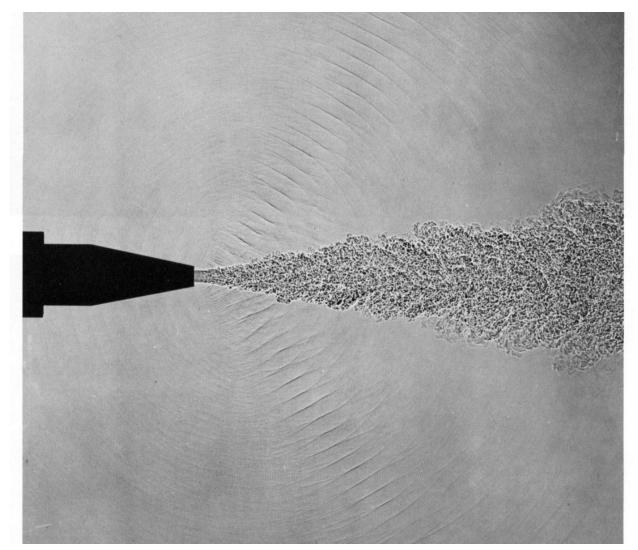
• For heat loads, the major factors are:

□ Magnitude and number of loads in space

- □ Load type (point or distributed sources)
- Elevation of load (e.g., overhead lighting, person standing on floor, glazing)
- □ Radiative/convective split
- Other heat sources

## **BEHAVIOR OF JETS**

 High speed supply air jets from the outlets mixes with room air to maintain indoor thermal comfort



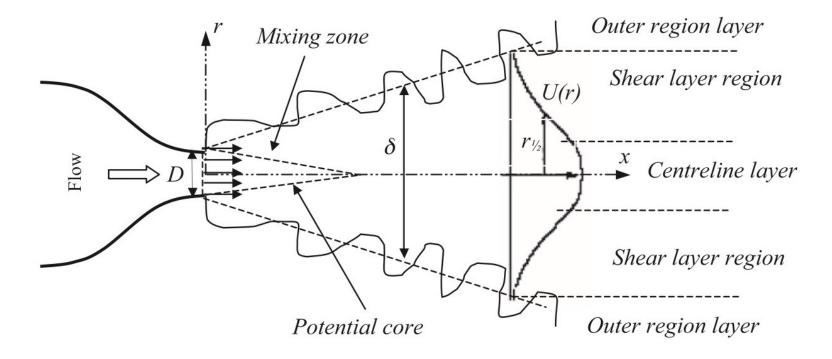
 $0 \le \frac{X}{D} \le 200$ 

$$0 \le \frac{X}{D} \le 35$$

Re = 10,000

Re = 2,500

 High speed supply air jets from the outlets mixes with room air to maintain indoor thermal comfort



- This air mixing
  - Leads to space heat transfer and resultant velocity reduction
    Needs to occur outside of the occupied zone

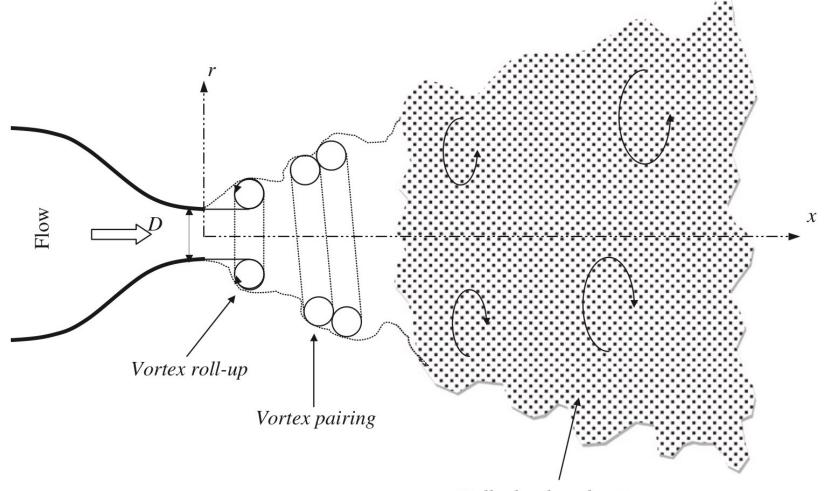
- If the supply air temperature is equal to the ambient room air temperature, the air jet is called an <u>isothermal</u>
- If the initial temperature of the air jet is different from the ambient air temperature, the air jet is called non-isothermal jet. The air temperature difference affects:
  - □ Jet trajectory
  - □ Location at which it attaches to and separates from the ceiling/floor
  - Throw

- If an air jet is not obstructed or affected by walls, ceiling, or other surfaces, it is considered as a <u>free jet</u>
- Usually, the outlet area is small compared to the dimensions of the space normal to air jet. We can consider it free jet as long as:

$$X \le 1.5 \sqrt{A_R}$$

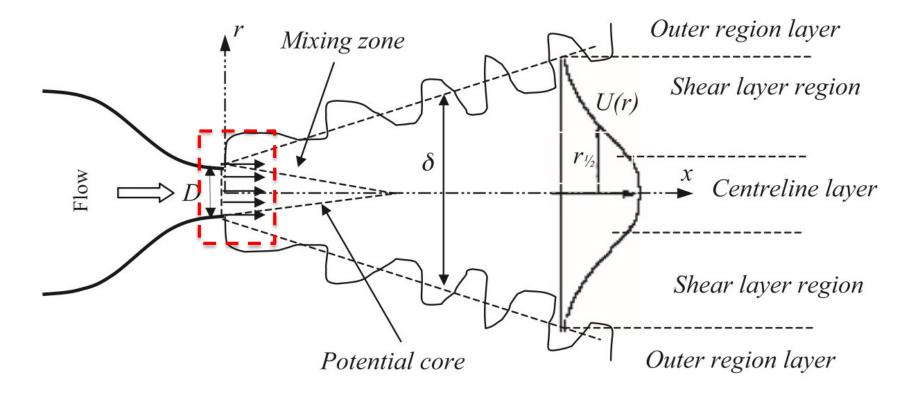
- $\Box$  X: Distance from face of the outlet (ft)
- $\Box$   $A_R$ : Cross-sectional area of confined space normal to jet (ft<sup>2</sup>)

• There are four distinct regions in an air jet.

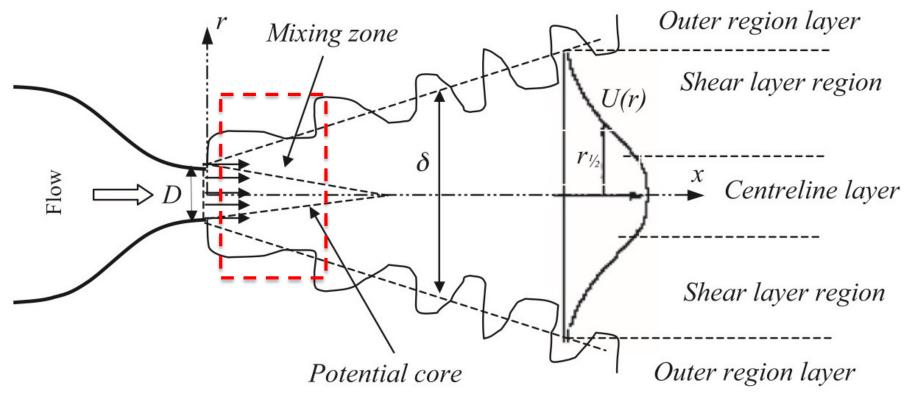


Fully developed region

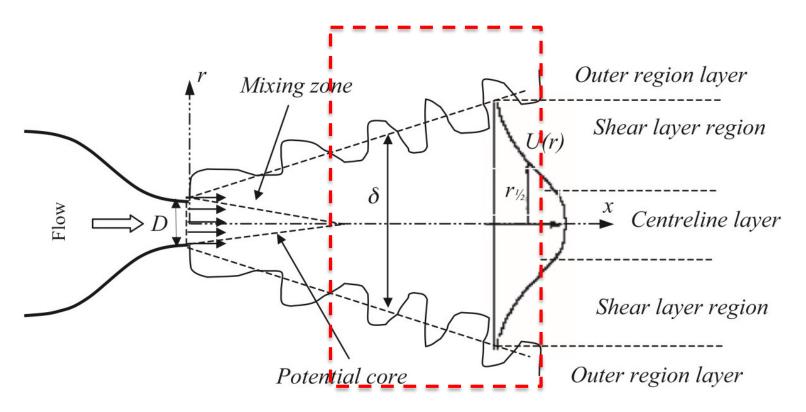
- Zone 1 (Core Zone):
  - Extend from the outlet face
  - □ Velocity and temperature of airstream remains relatively unchanged



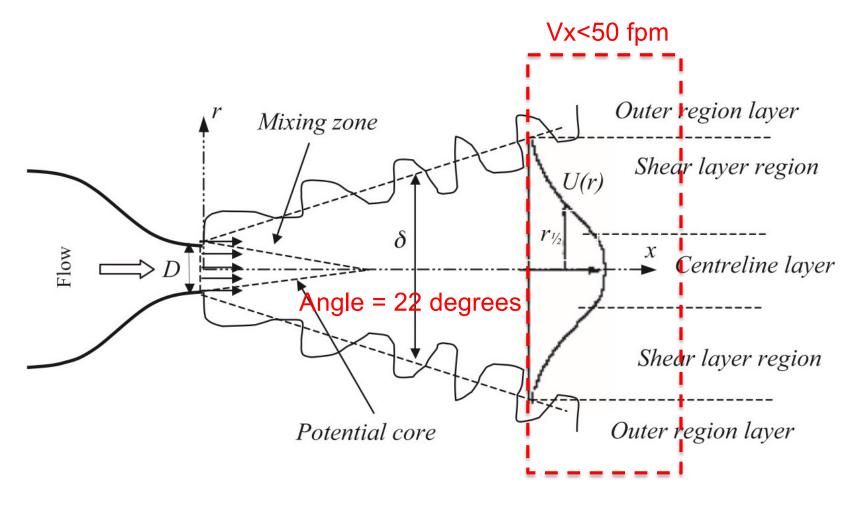
- Zone 2 (Transition Zone):
  - Transition zone
  - Length is determined by the type of outlet, aspect ratio of the outlet, initial airflow turbulence



- Zone 3 (Main Zone):
  - Zone of jet degradation where air velocity/temperature decrease rapidly
  - Turbulent flow is established and may be 25 to 100 equivalent of the air outlet diameters long
  - □ The angle is well-defined (20 degrees to 24 degrees)



- Zone 4 (Terminal Zone):
  - □ Most important since it enters the occupied area in this zone
  - □ The aim is to have 50 fpm velocity here



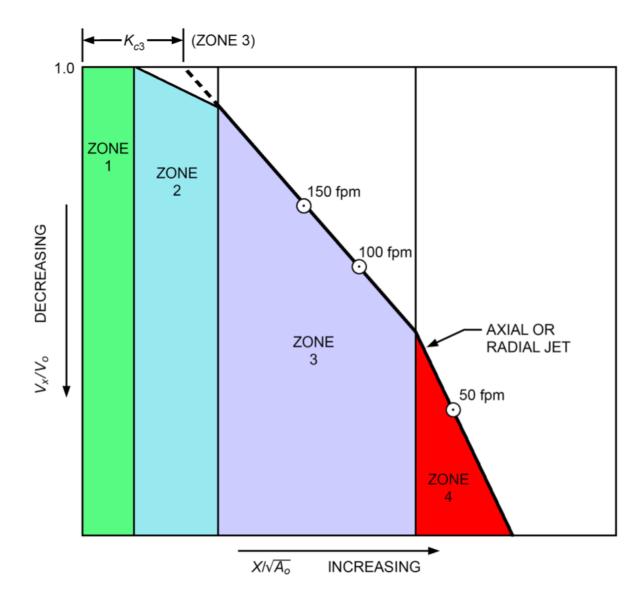


Fig. 11 Zones of Expansion for Axial or Radial Air Jets

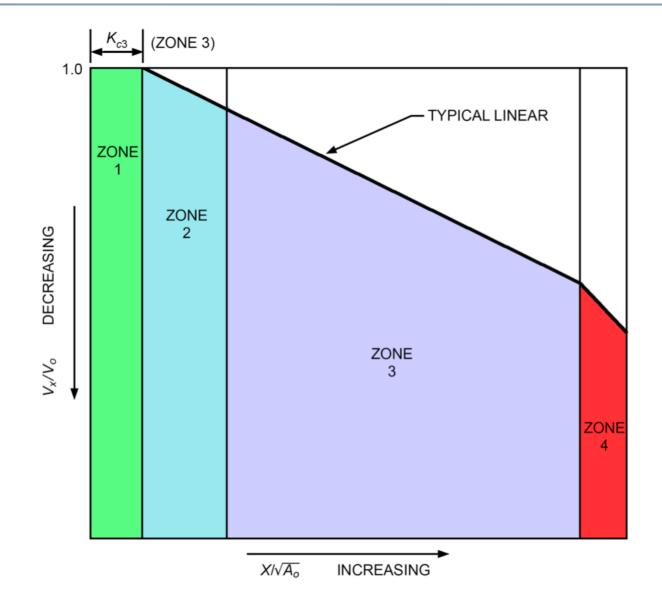
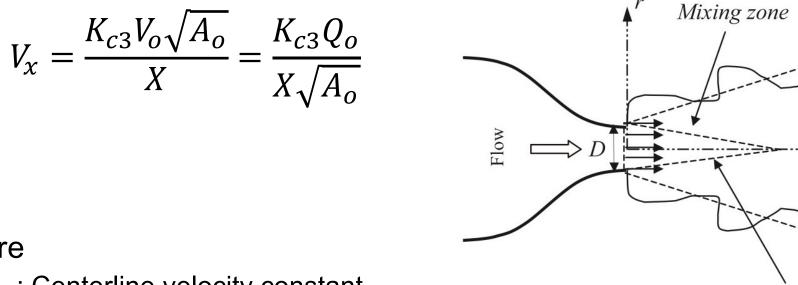


Fig. 12 Zones of Expansion for Linear Air Jets

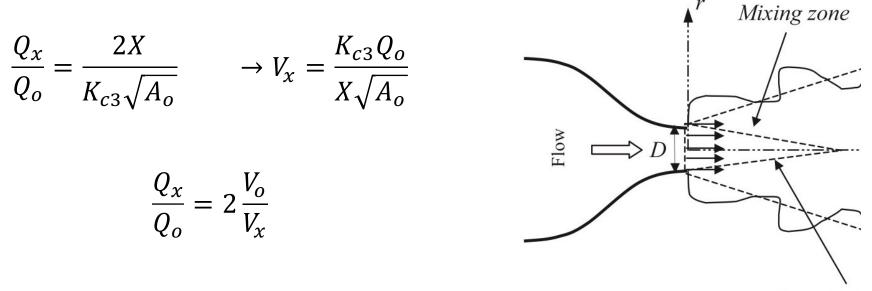
• Rectangular centerline jet velocity,  $V_x$  in Zone III is given by:



- Where
  - $\Box$   $K_{c3}$ : Centerline velocity constant
  - $\Box$   $Q_o$ : Discharge from outlet (cfm)
  - $\Box$  V<sub>o</sub>: Average initial discharge velocity (fpm)
  - $\Box$  X: Distance from outlet to the point of measurement (ft) = Throw
  - $\Box$   $A_o$ : Area corresponding to initial velocity (ft<sup>2</sup>)

### **Entrainment Ratio**

• Circular jets and jets from long slots



- Where
  - $\Box$   $Q_x$ : Total volumetric flow rate at distance X from face of outlet (cfm)
  - $\Box$   $Q_o$ : Discharge from outlet (cfm)
  - □ *X*: Distance from outlet to the point of measurement (ft)
  - $\Box$   $K_{c3}$ : Centerline velocity constant
  - $\Box$   $A_o$ : Area corresponding to initial velocity (ft^2)

# Table 1Generic Values for Centerline Velocity Constant $K_{c3}^{a}$ for Commercial Supply Outlets forFully and Partially Mixed Systems, Except UFAD

Outlet Type	Discharge Pattern	A <sub>o</sub>	$K_{c3}^{a}$
High sidewall grilles	0° deflection <sup>b</sup>	Free	5.7
(Figure 4)	Wide deflection	Free	4.2
High sidewall linear	Core less than 4 in. high <sup>c</sup>	Free	4.4
	Core more than 4 in. high	Free	5.0
Low sidewall	Up and on wall, no spread	Free	4.5
(Figure 7)	Wide spread <sup>c</sup>	Free	3.0
Baseboard	Up and on wall, no spread	Core	4.0
	Wide spread	Core	2.0
Floor grille (Figure 5)	No spread <sup>c</sup>	Free	4.7
22 24	Wide spread	Free	1.6
Ceiling (Figure 2)	360° horizontal <sup>d</sup>	Neck	1.1
	Four-way; little spread	Neck	3.8
Ceiling linear slot	Horizontal/vertical along surface <sup>c</sup>	Free	5.5
(Figure 3)	Horizontal/vertical free jetc	Free	3.9
	Free jet (air curtain units)	Free	6.0

<sup>a</sup>Generic values shown for example purposes <sup>b</sup>Free area is about 80% of core area. only. See manufacturer's data for specific <sup>c</sup>Free area is about 50% of core area.  $K_{c3}$  values. <sup>d</sup>Cone free area is greater than duct area.

# **CLASS ACTIVITY**

### **Class Activity**

- Example (Grille Throw): Consider a 12" by 18" high sidewall grille with zero deflection and 11.25" by 17.25" core area. The airflow rate is 600 cfm. Calculate the throw to:
  - □ 50 fpm
  - □ 100 fpm
  - □ 150 fpm

#### **Class Activity**

#### Solution:

#### Throw at 50 fpm:

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$$X = \frac{K_c Q_o}{V_x \sqrt{A_o}} \ (Zone \ III)$$

$$X = \frac{(5.7)(600 \ cfm)}{V_x \sqrt{\frac{11.25 \times 17.25}{144}}} = \frac{2,946}{V_x} \qquad X(at \ 50 \ fpm) = \frac{2,946}{50} = 59 \ ft$$

50 fpm is in zone IV. Zone IV is typically 20% to 30% less than the calculated value in Zone III

$$X = 59(0.8) = 47 ft$$

• Solution:

Throw at 100 fpm:

$$X(at\ 100\ fpm) = \frac{2,946}{100} = 29.5ft$$

Throw at 150 fpm:

$$X(at\ 150\ fpm) = \frac{2,946}{150} = 19.6ft$$