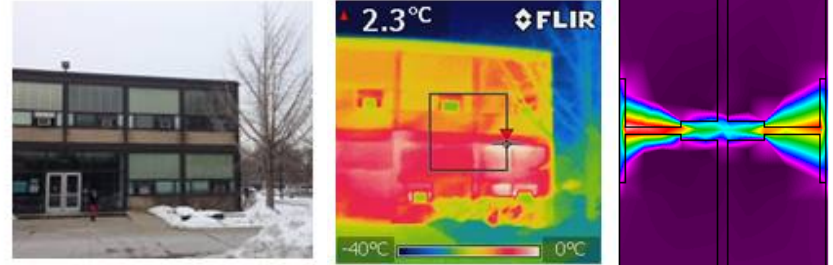


# CAE 331/513

## Building Science

### Fall 2018

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**September 11, 2018**

Combined modes of heat transfer and energy balances

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# Last time

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- Radiation
  - Long wave (surface to surface radiation)

$$Q_{1 \rightarrow 2} = \frac{A_1 \sigma (T_1^4 - T_2^4)}{\frac{1 - \varepsilon_1}{\varepsilon_1} + \frac{A_1}{A_2} \frac{1 - \varepsilon_2}{\varepsilon_2} + \frac{1}{F_{12}}}$$

$$Q_{1 \rightarrow 2} = \varepsilon_{surf} A_{surf} \sigma F_{12} (T_1^4 - T_2^4)$$

- Short wave (solar radiation)

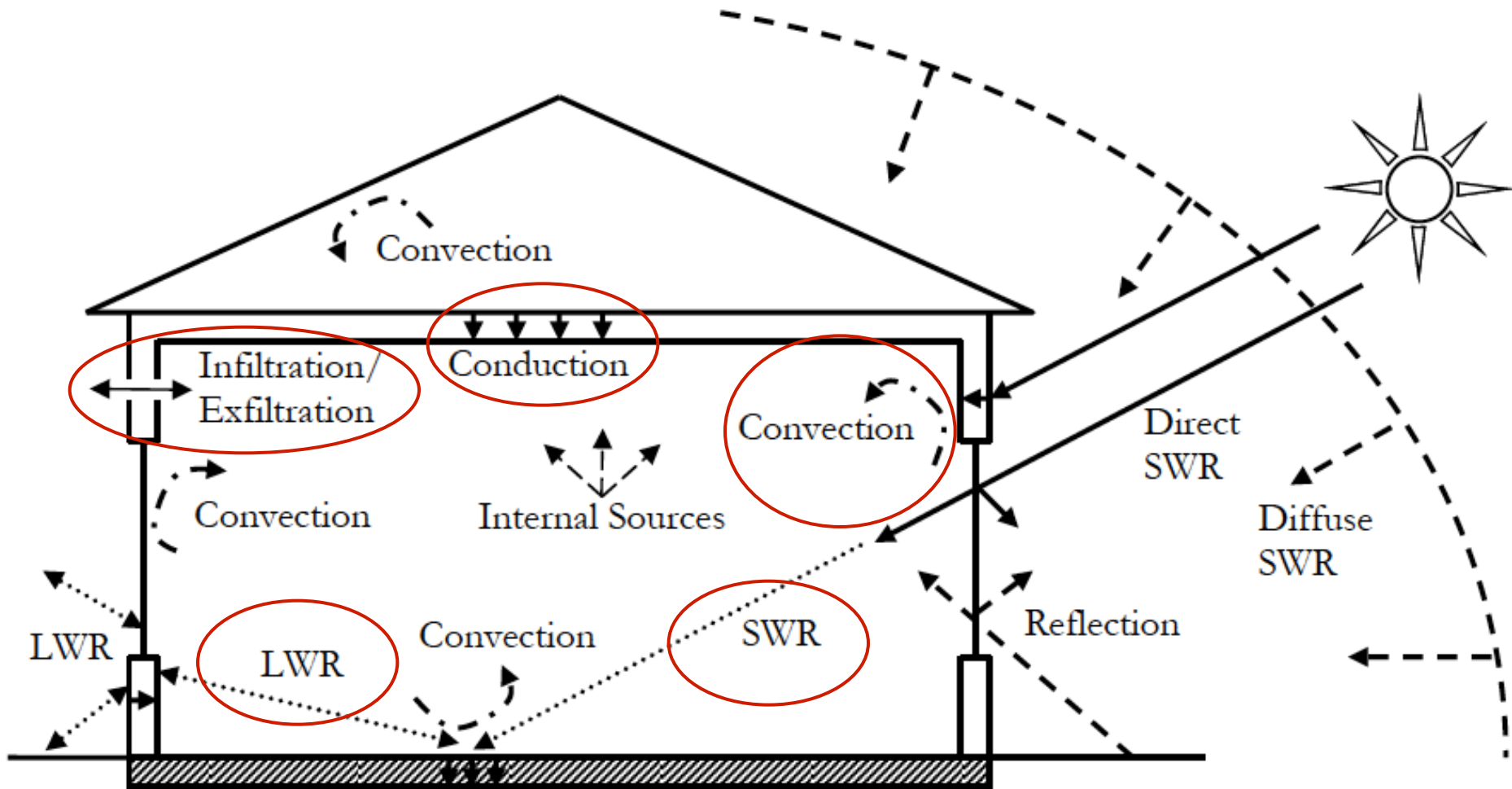
$$I_{solar} \left[ \frac{\text{W}}{\text{m}^2} \right]$$

Solar radiation:  
(opaque surface)  $q_{solar} = \alpha I_{solar}$

Transmitted solar radiation:  
(transparent surface)  $q_{solar} = \tau I_{solar}$

$$I_D = I_{DN} \cos \theta$$

# Summary: Modes of heat transfer in a building



# Summary to date: Modes of heat transfer in a building

## Conduction

$$q = \frac{k}{L} (T_{surf,1} - T_{surf,2})$$

$$\frac{k}{L} = U = \frac{1}{R}$$

$$R_{total} = \frac{1}{U_{total}}$$

$$R_{total} = R_1 + R_2 + R_3 + \dots$$

For thermal bridges and combined elements:

$$U_{total} = \frac{A_1}{A_{total}} U_1 + \frac{A_2}{A_{total}} U_2 + \dots$$

## Convection

$$q_{conv} = h_{conv} (T_{fluid} - T_{surf})$$

$$R_{conv} = \frac{1}{h_{conv}}$$

## Advection

$$Q_{bulk} = \dot{m} C_p \Delta T$$

## Radiation

### Long-wave

$$q_{1 \rightarrow 2} = \frac{\sigma (T_{surf,1}^4 - T_{surf,2}^4)}{\frac{1 - \epsilon_1}{\epsilon_1} + \frac{A_1}{A_2} \frac{1 - \epsilon_2}{\epsilon_2} + \frac{1}{F_{12}}}$$

$$q_{rad,1 \rightarrow 2} = h_{rad} (T_{surf,1} - T_{surf,2})$$

$$h_{rad} = \frac{4\sigma T_{avg}^3}{\frac{1}{\epsilon_1} + \frac{1}{\epsilon_2} - 1} \quad R_{rad} = \frac{1}{h_{rad}}$$

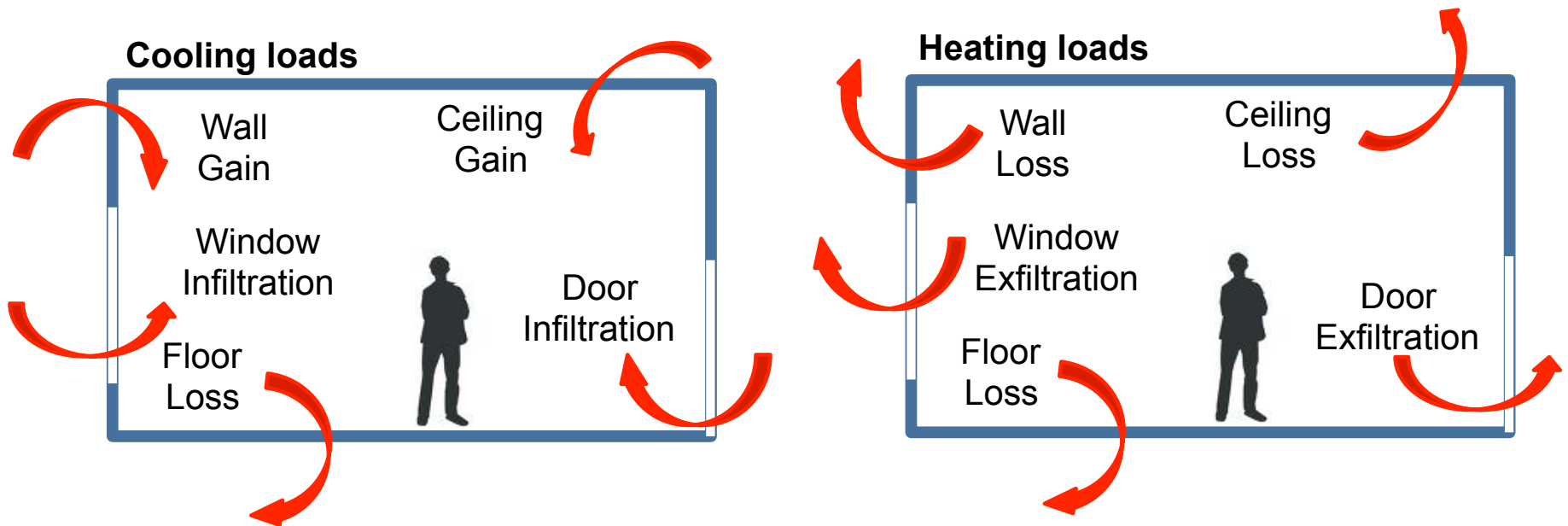
$$q_{1 \rightarrow 2} = \epsilon_{surf} \sigma F_{12} (T_{surf,1}^4 - T_{surf,2}^4)$$

Solar radiation:  $q_{solar} = \alpha I_{solar}$   
(opaque surface)

Transmitted solar radiation:  $q_{solar} = \tau I_{solar}$   
(transparent surface)

# Where are we going? Building energy balances

- Taken altogether, each of the heat transfer modes we've discussed can be combined with inputs for climate data, material properties, and geometry to make up a building's **energy balance**
  - We will also revisit this for **heating** and **cooling** load calculations



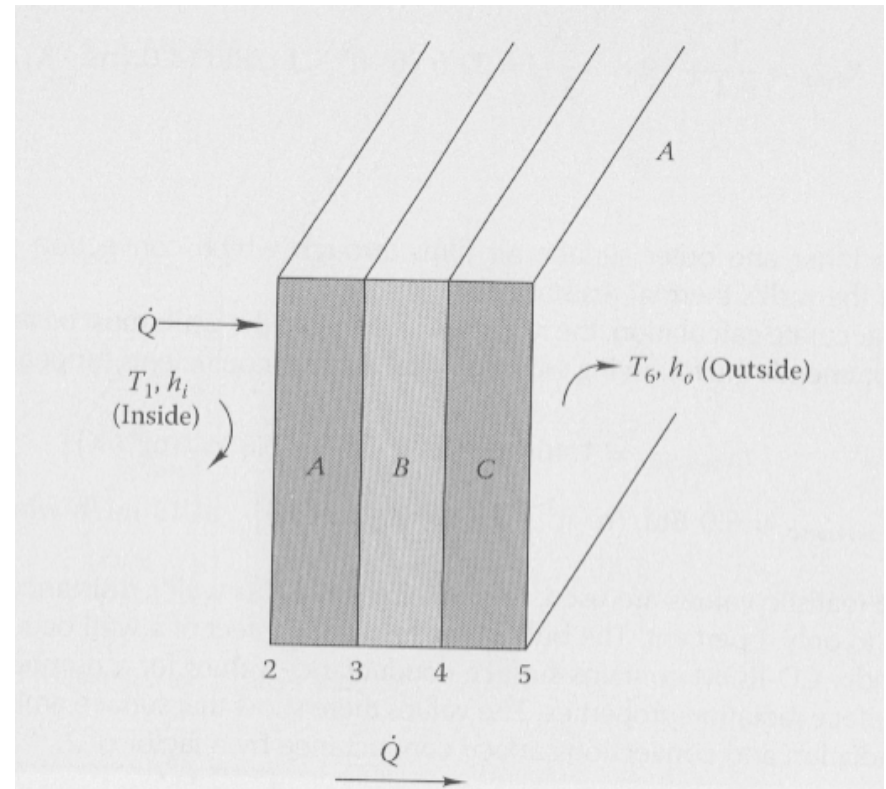
# **COMBINED MODE HEAT TRANSFER**

# Combined mode heat transfer: Series

- Nearly all heat transfer situations in buildings include more than one mode of heat transfer
- When more than one heat transfer mode is present, we can compute heat transfer (of all kinds) using resistances in series
  - **Sum resistances in series**

$$R_{tot} = R_A + R_B + R_C + R_{film,in} + R_{film,out}$$

$$q = \frac{1}{R_{tot}} (T_{air,in} - T_{air,out})$$



# Combined mode heat transfer: Series

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- Remember our wood stud wall?
  - Fiberglass batts in 2x6 inch stud cavities, with brick exterior and gypsum board interior ( $R_{\text{assembly}} = 18.7 \text{ IP}$ )



- Just need to add the “film resistances” to calculate heat transfer from indoor air to outdoor air



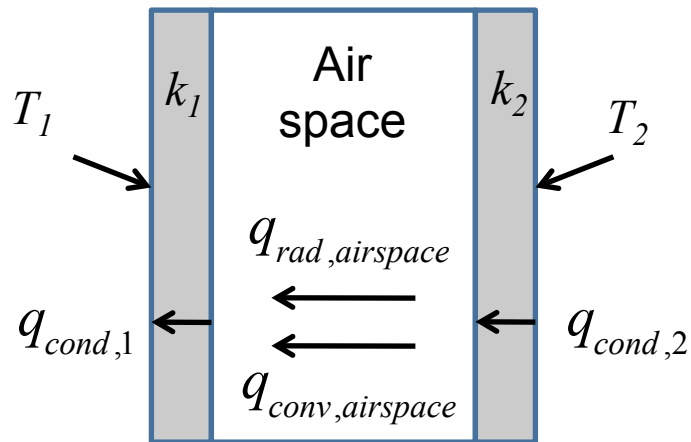
# Typical convective “film resistances”

- We often use the values given below for most conditions

Surface Conditions	Horizontal Heat Flow	Upwards Heat Flow	Downwards Heat Flow
Indoors: $R_{in}$	0.12 m <sup>2</sup> K/W (SI) 0.68 h·ft <sup>2</sup> ·°F/Btu (IP)	0.11 m <sup>2</sup> K/W (SI) 0.62 h·ft <sup>2</sup> ·°F/Btu (IP)	0.16 m <sup>2</sup> K/W (SI) 0.91 h·ft <sup>2</sup> ·°F/Btu (IP)
$R_{out}$ : 6.7 m/s wind (Winter)		0.030 m <sup>2</sup> K/W (SI) 0.17 h·ft <sup>2</sup> ·°F/Btu (IP)	
$R_{out}$ : 3.4 m/s wind (Summer)		0.044 m <sup>2</sup> K/W (SI) 0.25 h·ft <sup>2</sup> ·°F/Btu (IP)	

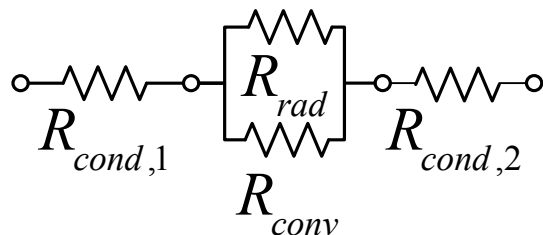
# Combined mode heat transfer: Parallel

- When more than one mode of heat transfer exists at a location (e.g., convection and radiation), resistances get placed **in parallel**
  - Example: Heat transfer in a building cavity



$$\frac{1}{R_{airspace}} = \frac{1}{R_{rad,airspace}} + \frac{1}{R_{conv,airspace}}$$

$$R_{tot} = R_1 + R_{airspace} + R_2$$



$$q = \frac{1}{R_{tot}} (T_2 - T_1)$$

# Combined modes of heat transfer: Parallel

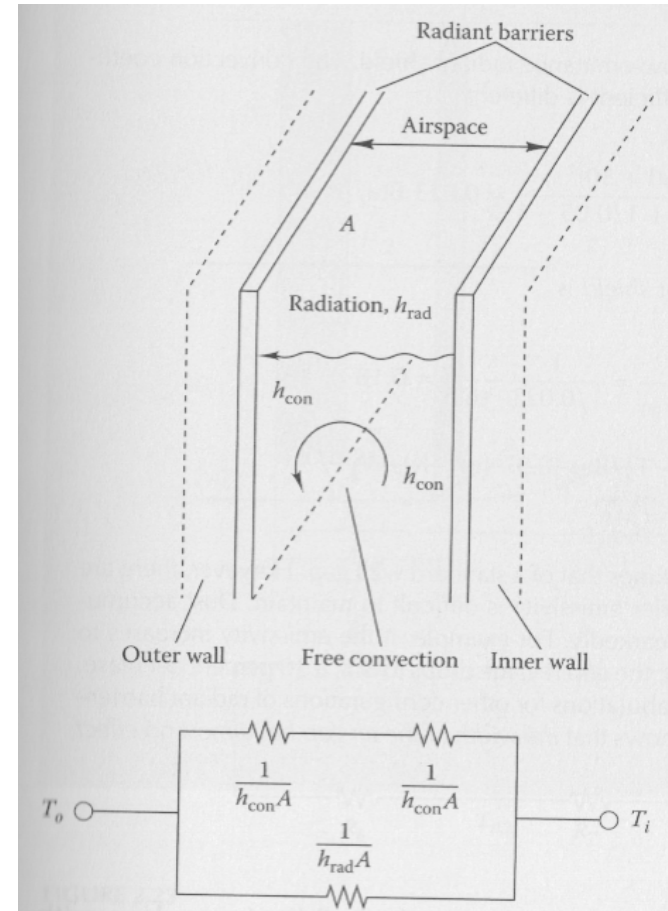
- Example problem: Radiant barrier in a residential wall

A building designer wishes to evaluate the R-value of a 1-inch wide *ventilated air gap* in a wall for its insulation effect

She finds the resistance to heat flow to be quite small, so she proposes lining the cavity's inner and outer surfaces with a highly reflecting aluminum foil film whose emissivity is 0.05

Find the R-value of this cavity with both emissivity conditions, including both radiation and convection effects in the cavity

Assume the surface temperatures facing the gap are  $7.2^{\circ}\text{C}$  and  $12.8^{\circ}\text{C}$



# Combined convection + conduction: **Heat exchangers**

- Heat exchangers are used widely in buildings
- Heat exchangers are devices in which two fluid streams, usually separated from each other by a solid wall, exchange thermal energy by both convection and conduction
  - One fluid is typically heated, one is typically cooled
    - Fluids may be gases, liquids, or vapors

$$k_{\text{aluminum}} = 205 \text{ W/mK}$$
$$k_{\text{copper}} = 385 \text{ W/mK}$$

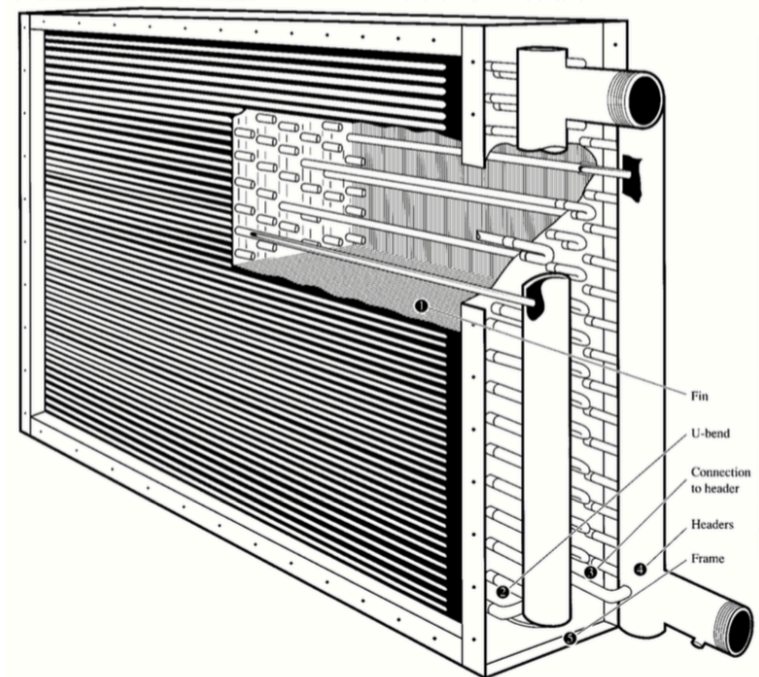
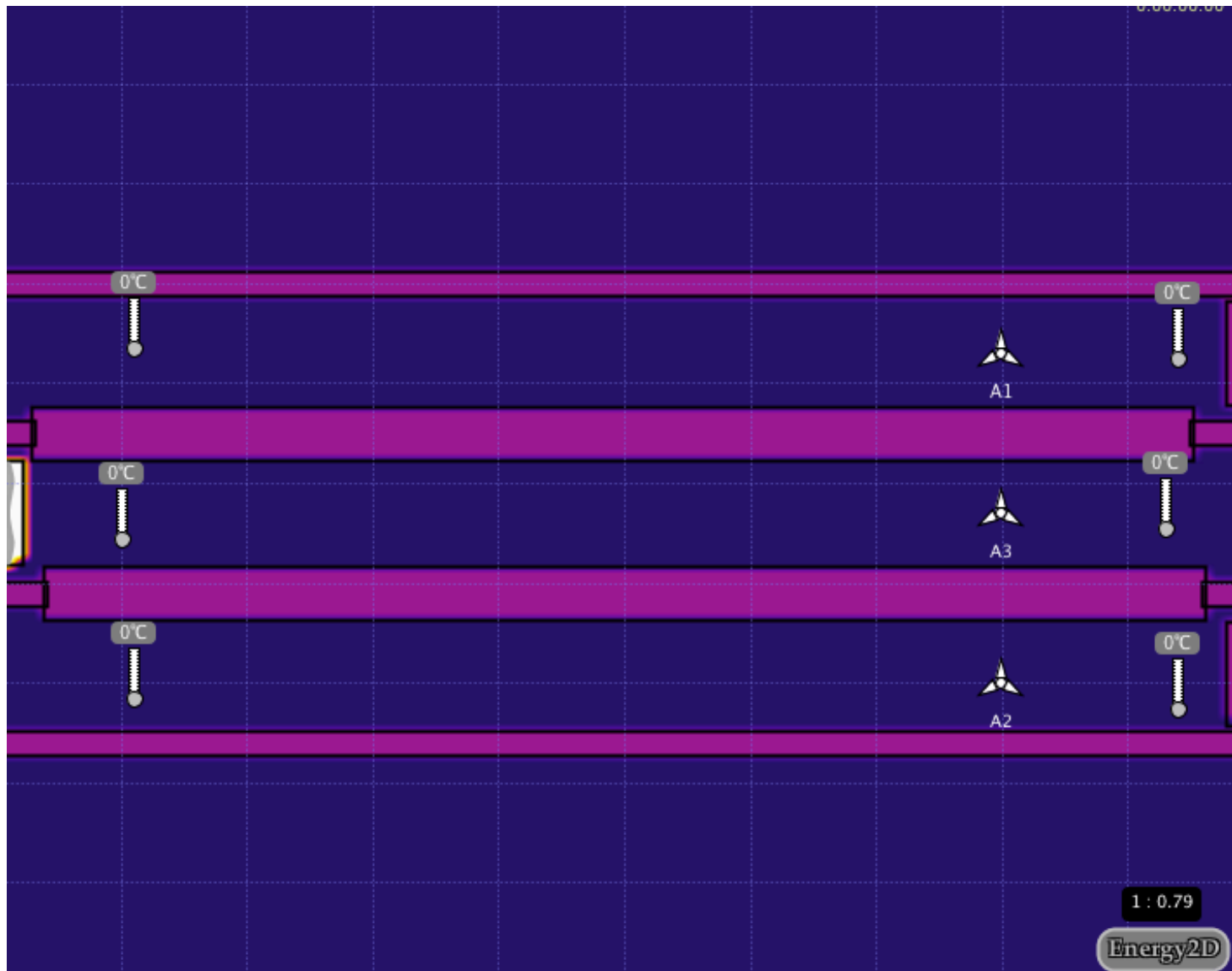


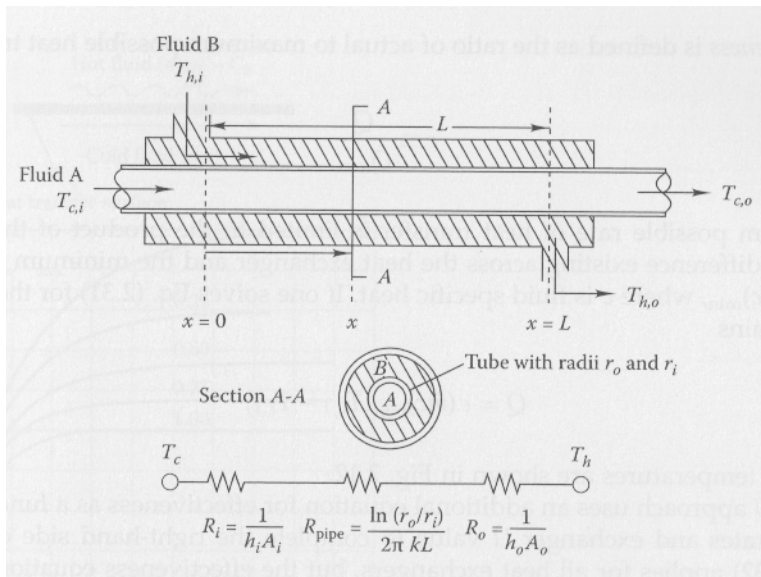
FIGURE 15.26a Structure of a water cooling coil. (Source: York International Corporation. Reprinted with permission.)

# Combined convection + conduction: **Heat exchangers**



# Combined convection + conduction: Heat exchangers

- The effectiveness of a heat exchanger depends on:
  - The flow rates of fluids in the heat exchanger
  - The overall UA-value of the heat exchanger
    - U is governed by convection and conduction resistance
    - A is governed by heat exchanger design (high surface A)



$$U_o A_o = \frac{1}{R'_{\text{conv},i} + R'_{\text{pipe}} + R'_{\text{conv},o}}$$

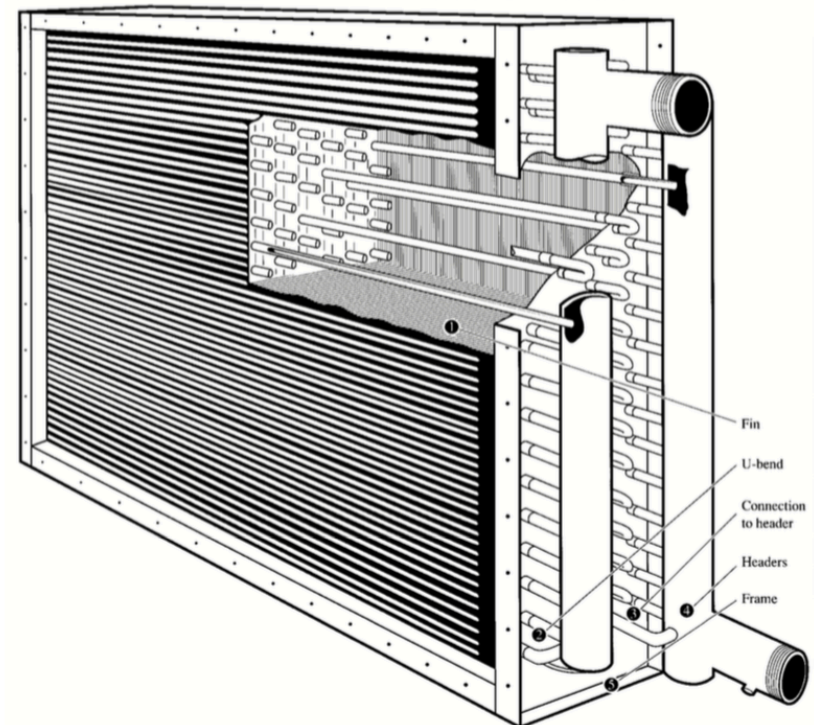
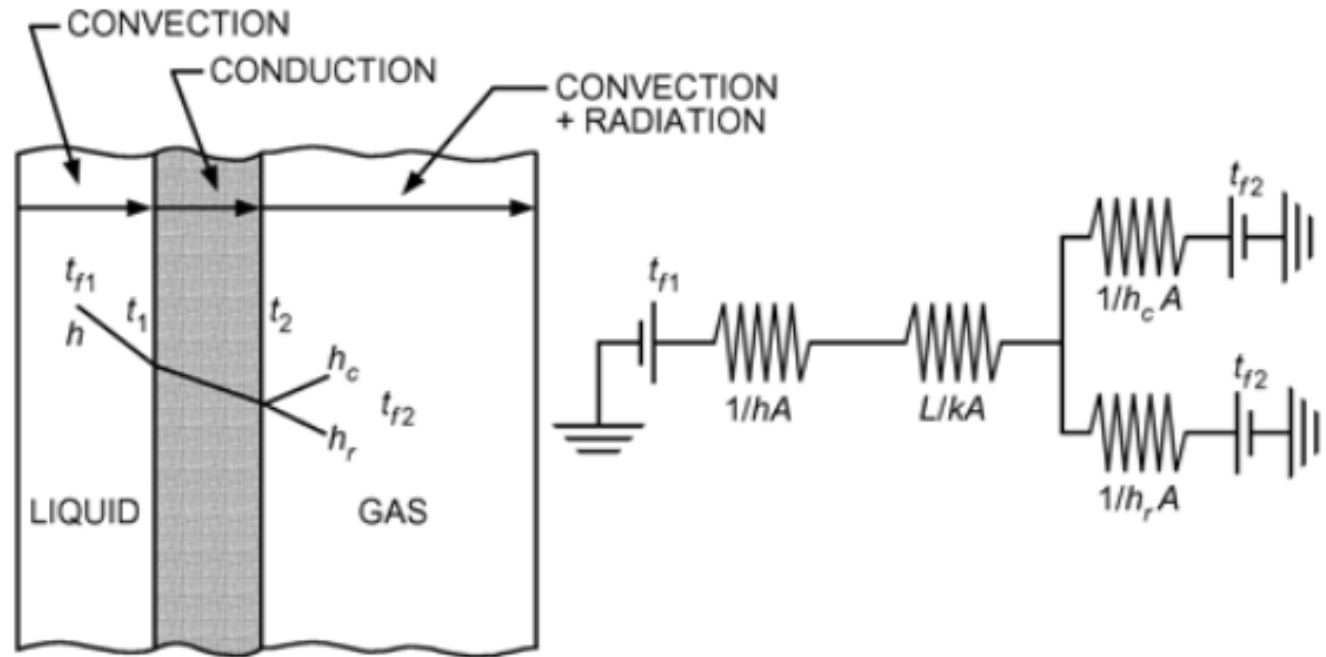


FIGURE 15.26a Structure of a water cooling coil. (Source: York International Corporation. Reprinted with permission.)

# Combined convection + conduction: Heat exchangers

- Example from ASHRAE Handbook of Fundamentals (Ch. 4)



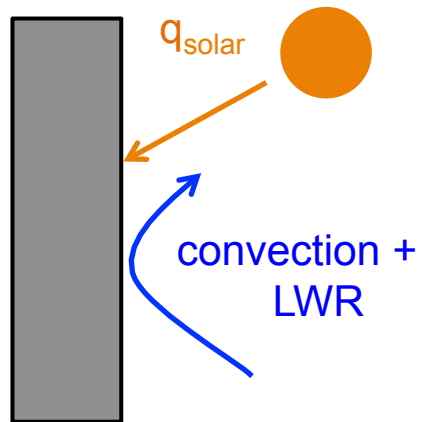
$$q = UA(T_{f,1} - T_{f,2})$$

$$\frac{1}{UA} = R'_{tot} = R'_1 + R'_{1-2} + R'_2$$

$$R'_1 = \frac{1}{h_{1,conv} A} \quad R'_{1-2} = \frac{L}{kA}$$

$$\frac{1}{R'_2} = \frac{1}{R'_{2,conv}} + \frac{1}{R'_{2,rad}}$$

$$R'_{2,conv} = \frac{1}{h_{2,conv} A} \quad R'_{2,rad} = \frac{1}{h_{2,rad} A}$$



# BUILDING ENERGY BALANCES



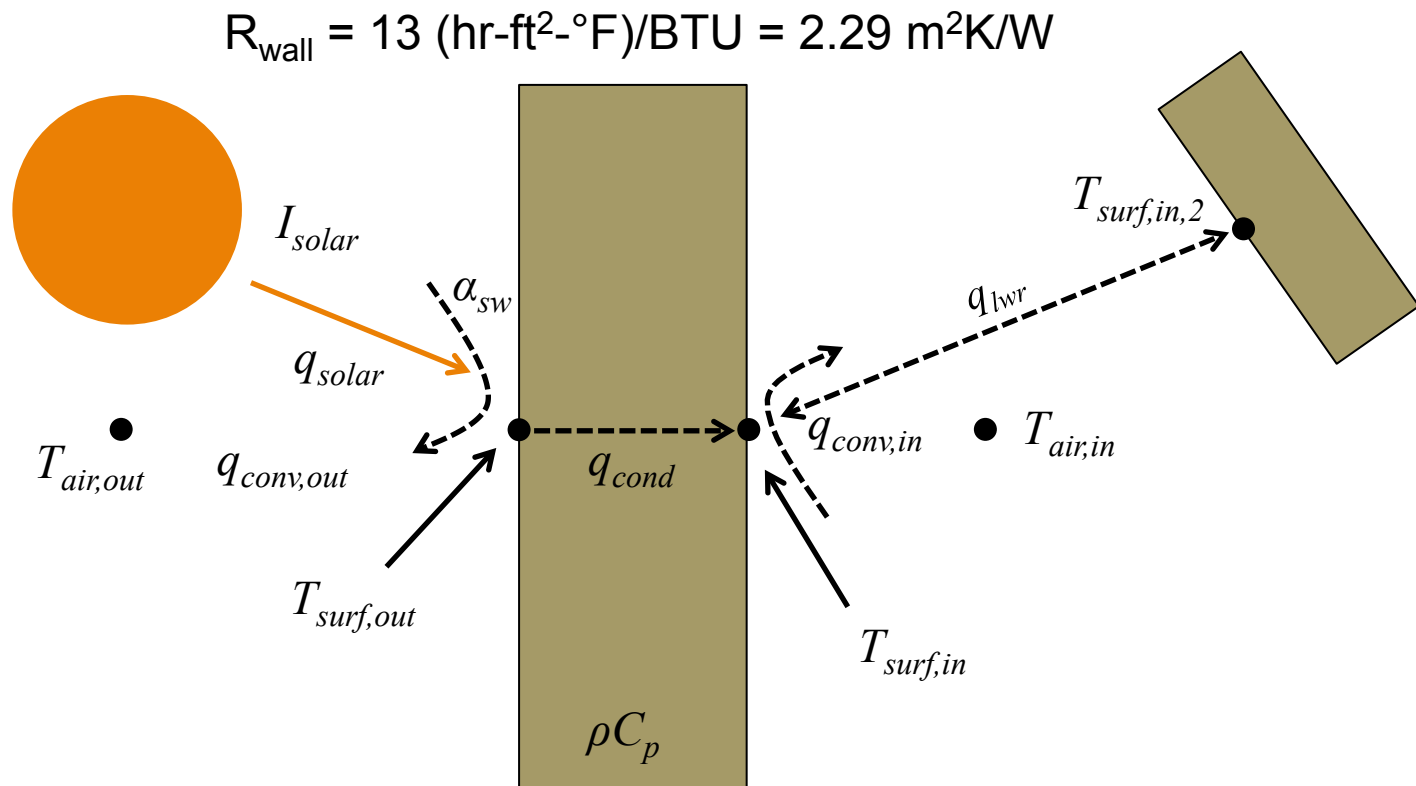
# Building energy balances

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- We know that multiple modes of heat transfer are typically acting at the same time at multiple points point within a building...
  - ... So we can also write expressions to quantify heat flow/flux to/from these various points simultaneously by accounting for all relevant modes of heat transfer
    - Writing “building energy balances”
    - Solving systems of equations

# Building energy balances: Simplified

Imagine an external wall of a building:



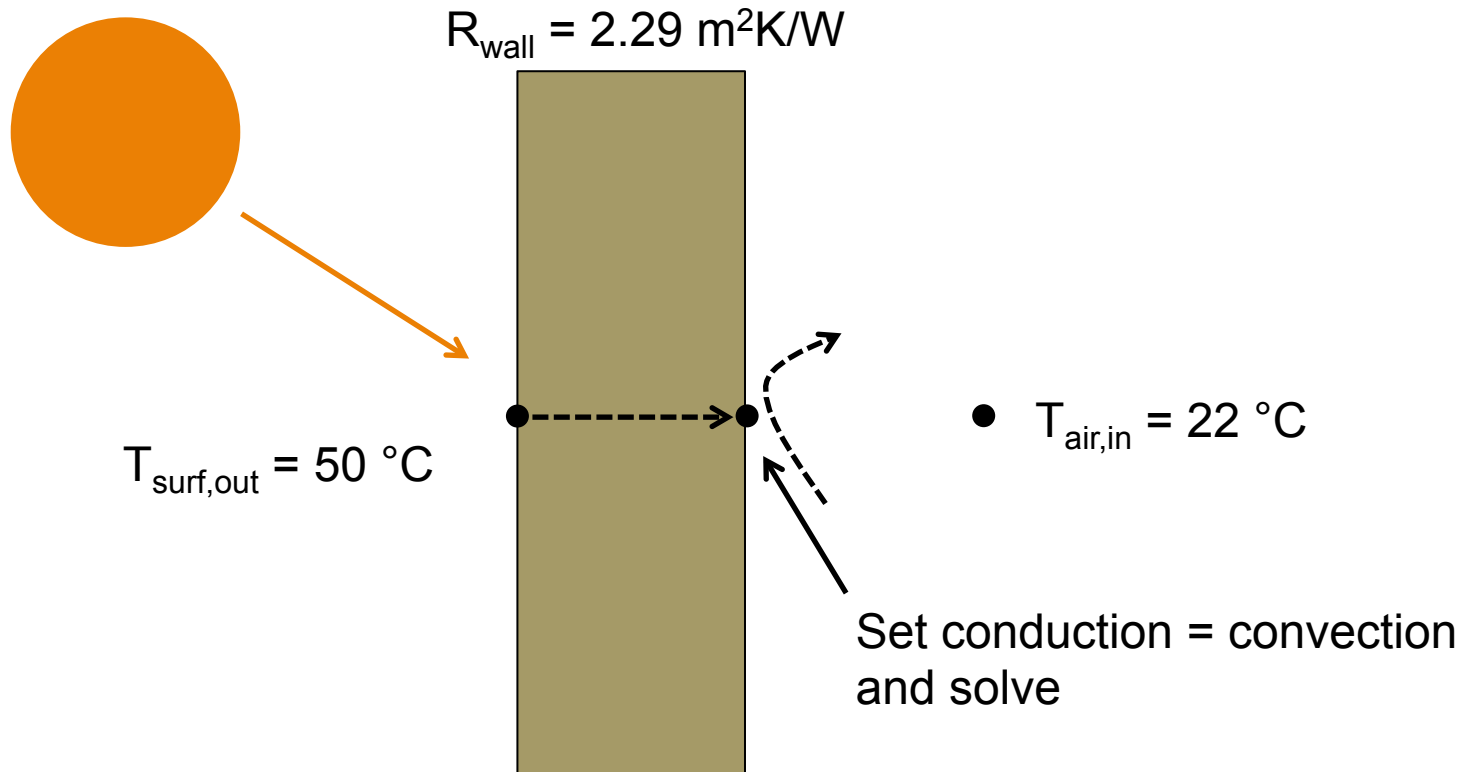
# How is this helpful to us?

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- Imagine the classroom wall behind me is being heated by the sun on the other side
- The exterior surface temperature is 122°F (50°C)
- The interior air temperature is 72°F (22°C)
- The R-value of the wall is R-13 (IP) (2.29 m<sup>2</sup>K/W)
- What is the interior surface temperature of the wall?
- This interior surface temperature impacts the heat flux to indoor air, as well as the surrounding surface temperatures (via radiation), which all impact the building's energy balance

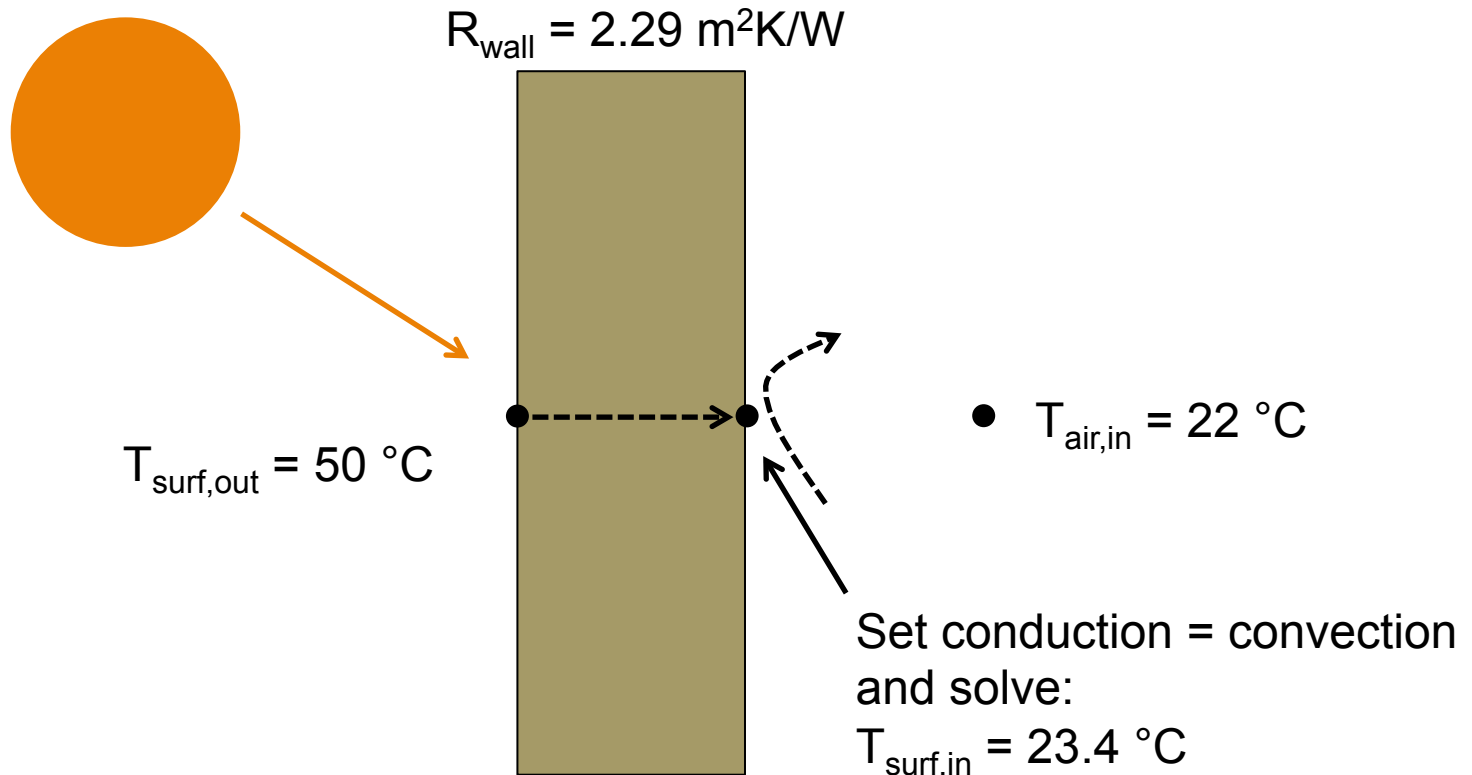
# Building energy balance example

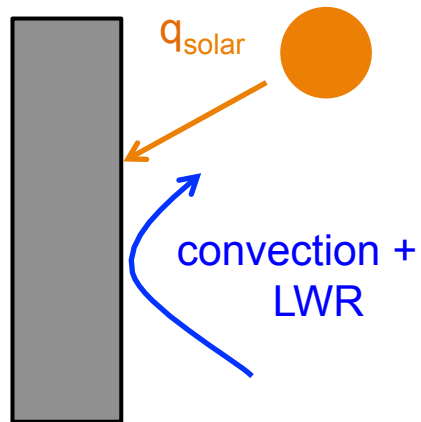
- Estimate the surface temperature of an interior wall whose exterior side is being warmed by the sun
  - Assume that LWR can be ignored and assume steady-state



# Building energy balance example

- Estimate the surface temperature of an interior wall whose exterior side is being warmed by the sun
  - Assume that LWR can be ignored and assume steady-state

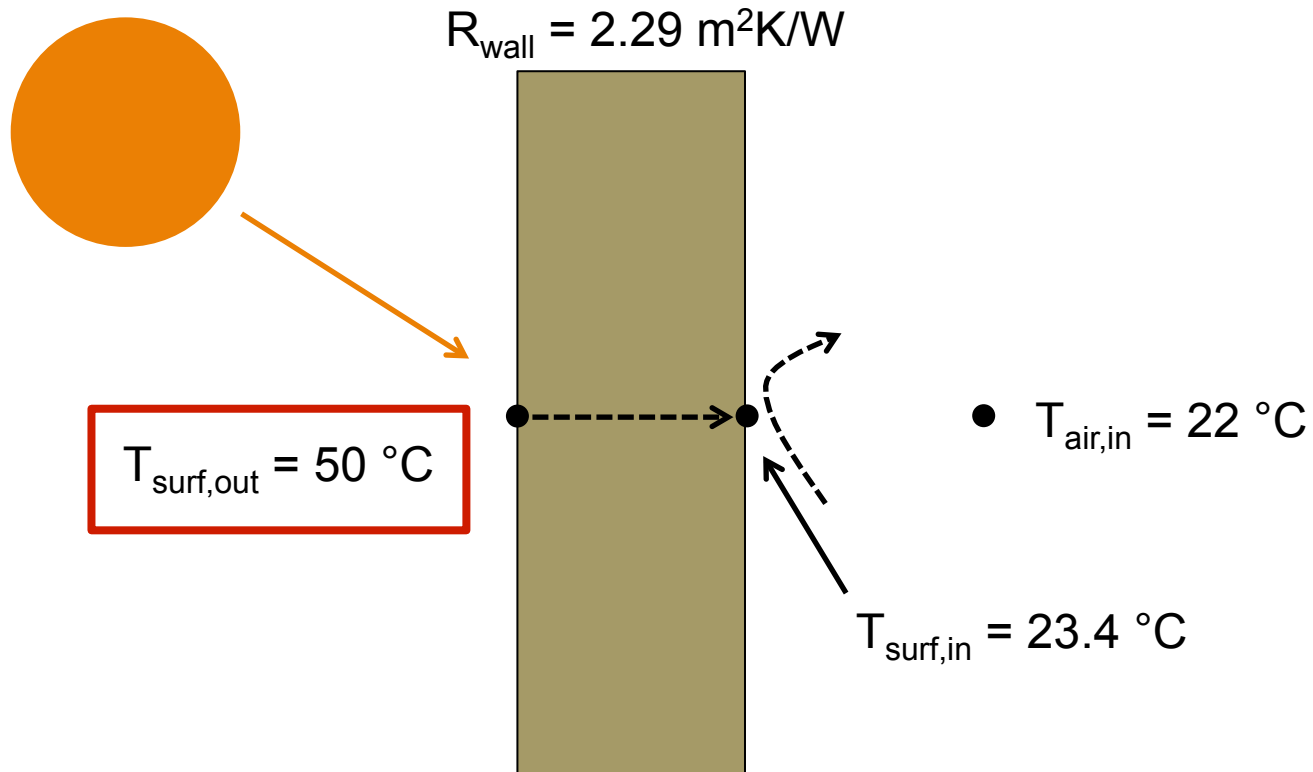




## “SOL-AIR” TEMPERATURES

# Sol-air temperatures

- In the last example, we were given that the exterior surface temperature was 122°F (50°C)
  - How did we know that?



# Sol-air temperatures

- If we take an external surface with a combined convective and radiative heat transfer coefficient,  $h_{conv+rad}$

$$q_{conv+rad} = h_{conv+rad} (T_{air} - T_{surf})$$

- If that surface now absorbs solar radiation ( $\alpha I_{solar}$ ), the total heat flow at the exterior surface becomes:

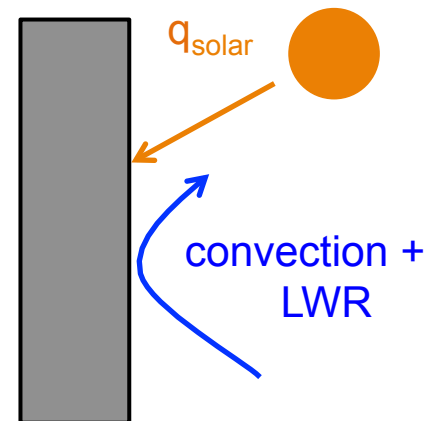
$$q_{conv+rad} = h_{conv+rad} (T_{air} - T_{surf}) + \alpha I_{solar}$$

- To simplify our calculations, we can define a “**sol-air**” temperature that accounts for all of these impacts:

$$T_{sol-air} = T_{air} + \frac{\alpha I_{solar}}{h_{conv+rad}}$$

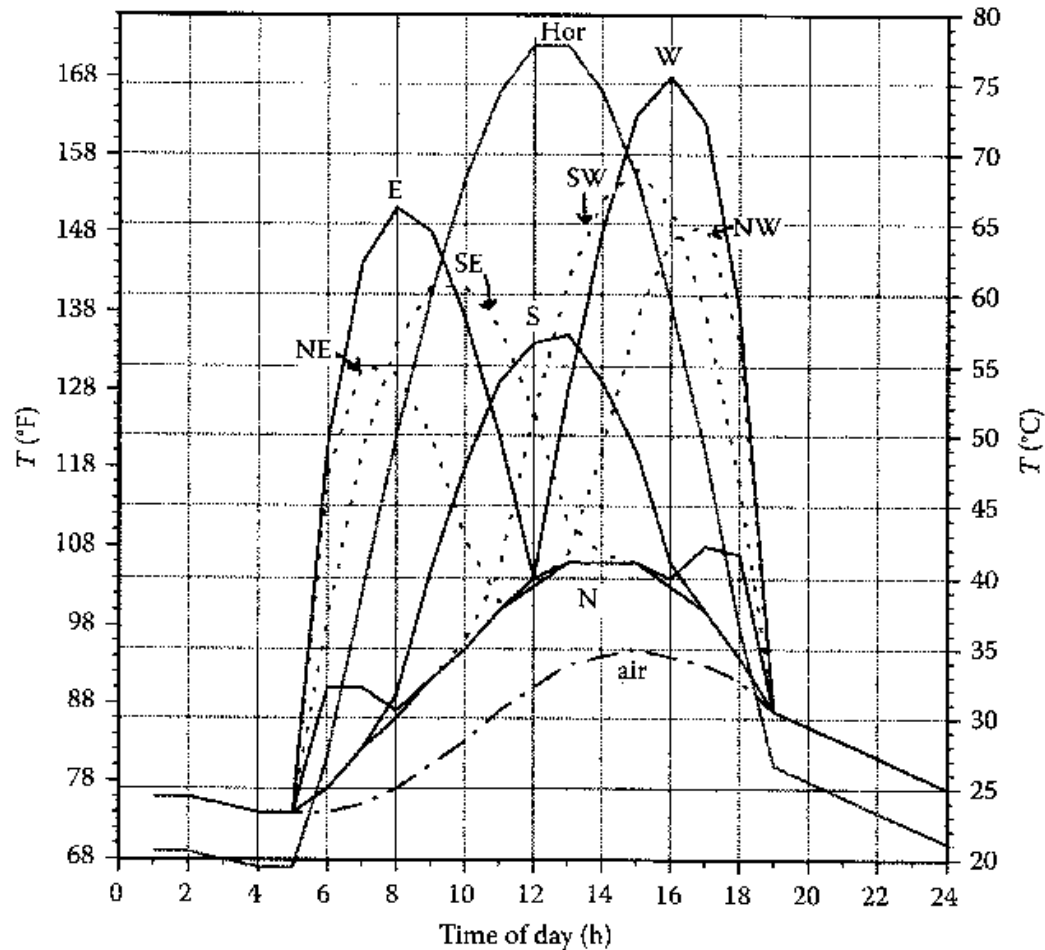
- Now we can describe heat transfer at that surface as:

$$q_{total} = h_{conv+rad} (T_{sol-air} - T_{surf})$$





# Example **sol-air** temperatures



**FIGURE 6.17**

Sol-air temperature for horizontal and vertical surfaces as a function of time of day for summer design conditions, July 21 at  $40^\circ$  latitude, assuming  $\alpha/h_o = 0.30$  ( $\text{h} \cdot \text{ft}^2 \cdot ^\circ\text{F}$ )/Btu [ $0.052$  ( $\text{m}^2 \cdot \text{K}$ )/W]. The curves overlap when there is  $\infty$  direct radiation on a surface. (Courtesy of ASHRAE, *Handbook of Fundamentals*, American Society of Heating, Refrigerating and Air-Conditioning Engineers, Atlanta, GA, 1989, Table 26.1.)

# Solar radiation and external surface temperatures

- We can also use air temperatures and material properties (absorptivity and emissivity) to estimate exterior surface temperatures that are exposed to radiation
  - These are not extremely accurate but provide a reasonable estimate

<b>Situation</b>	<b>Thermally massive</b>	<b>Thermally lightweight</b>
Roofs: direct sun	$t_a + 42 \alpha$	$t_a + 55 \alpha$
Roof: sun + reflected /emitted radiation	$t_a + 55 \alpha$	$t_a + 72 \alpha$
Roof exposed to night sky	$t_a - 5 \varepsilon$	$t_a - 10 \varepsilon$
Walls: winter sun	$t_a + 35 \alpha$	$t_a + 48 \alpha$
Walls: summer sun	$t_a + 28 \alpha$	$t_a + 40 \alpha$
Walls exposed to night sky	$t_a - 2 \varepsilon$	$t_a - 4 \varepsilon$

Source: Straube and Burnett

# Next time

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- HW #2 is due
- Fenestration (doors and windows) – applications of combined mode heat transfer