

# CAE 208 / MMAE 320: Thermodynamics

## Fall 2023

---

**October 17, 2023**

Mass & energy analysis of control volumes (1)

Built  
Environment  
Research

@ IIT



*Advancing energy, environmental, and  
sustainability research within the built environment*

[www.built-envi.com](http://www.built-envi.com)

**Dr. Mohammad Heidarinejad, Ph.D., P.E.**  
Civil, Architectural and Environmental Engineering  
Illinois Institute of Technology

[muh182@iit.edu](mailto:muh182@iit.edu)

# **ANNOUNCEMENTS**

# Announcement

Career Services  
Illinois Tech

**ASCE**  
ILLINOIS INSTITUTE OF TECHNOLOGY

## 9TH ANNUAL CAEE CAREER FAIR

Hosted by CAEE orgs  
and Career Services

OCTOBER 17, 2023  
2:00PM - 5:00PM  
HERMAN HALL

A digital resume book will be made available to registered companies a week before the fair begins.

For more information scan the QR code below!

**Scan Here  
To Register!**



# Announcements

---

- Assignment 5 due Thursday

# Announcements

---

- Midterm exam 1:
  - The solutions files are uploaded
  - The exams were graded

# Announcements

- Midterm exam 1:
  - ❑ Watch the recordings if you need additional review time

The screenshot displays the Panopto Content interface. On the left is a dark sidebar menu with the following items: CAE\_208\_MMAE\_320.2024 10 (Thermodynamics) (selected), Home, Syllabus, Content, Assignments, Discussions, Collaborate Ultra, Class Recording | Panopto, My Grades, Email, Galvin Library, and Tools. The main content area is titled "Panopto Content" and features a search bar with the text "Search in folder 'CAE\_208\_MMAE\_320.202410: Thermodynamics'...", a "+ Create" button, and a folder icon labeled "CAE\_208\_MMAE\_320.202410: Thermodynamics". Below the folder name is a "Sort by:" menu with options for Name, Duration, Date, and Rating. A dashed box highlights a "+ Add folder" button. At the bottom, a video thumbnail is shown with the title "10/05/2023 Lecture14\_Energy analysis of closed systems (2)".

# Announcements

---

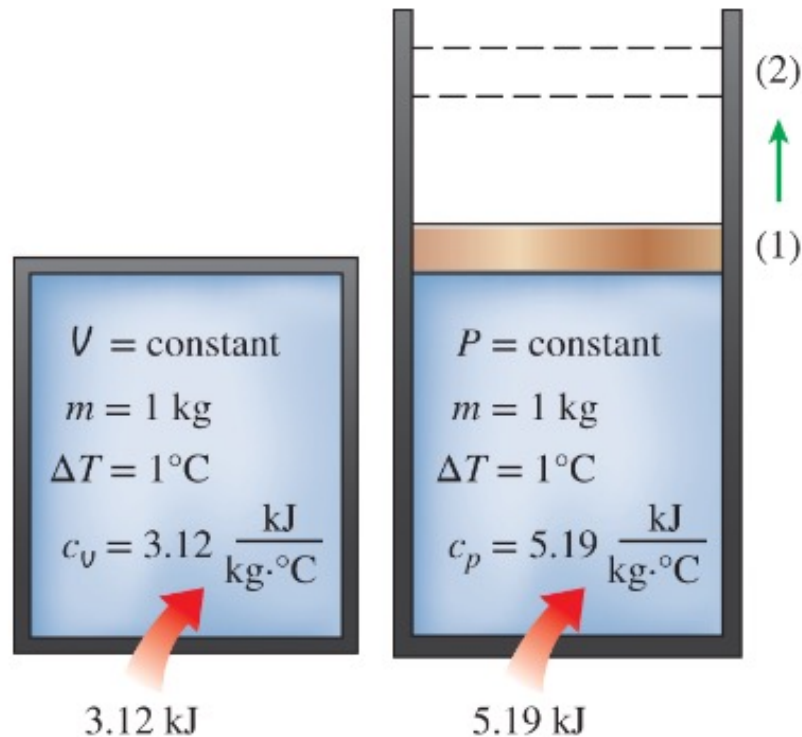
- Midterm exam 2:
  - ❑ November 07 (If we need to change it, please let me know as soon as possible)
  - ❑ All students are responsible for their open book exam part
    - No book / tables sharing will be allowed
    - I will not bring any extra books

**RECAP**



# Recap

- Specific heat is defined as the energy required to raise temperature of a unit mass of substance by one degree
  - ❑ Specific heat at constant volume ( $c_v$ )
  - ❑ Specific heat at constant pressure ( $c_p$ )



# Recap

---

$$c_v = \left( \frac{\partial u}{\partial T} \right)_v$$

= the change in internal energy  
with temperature at  
constant volume

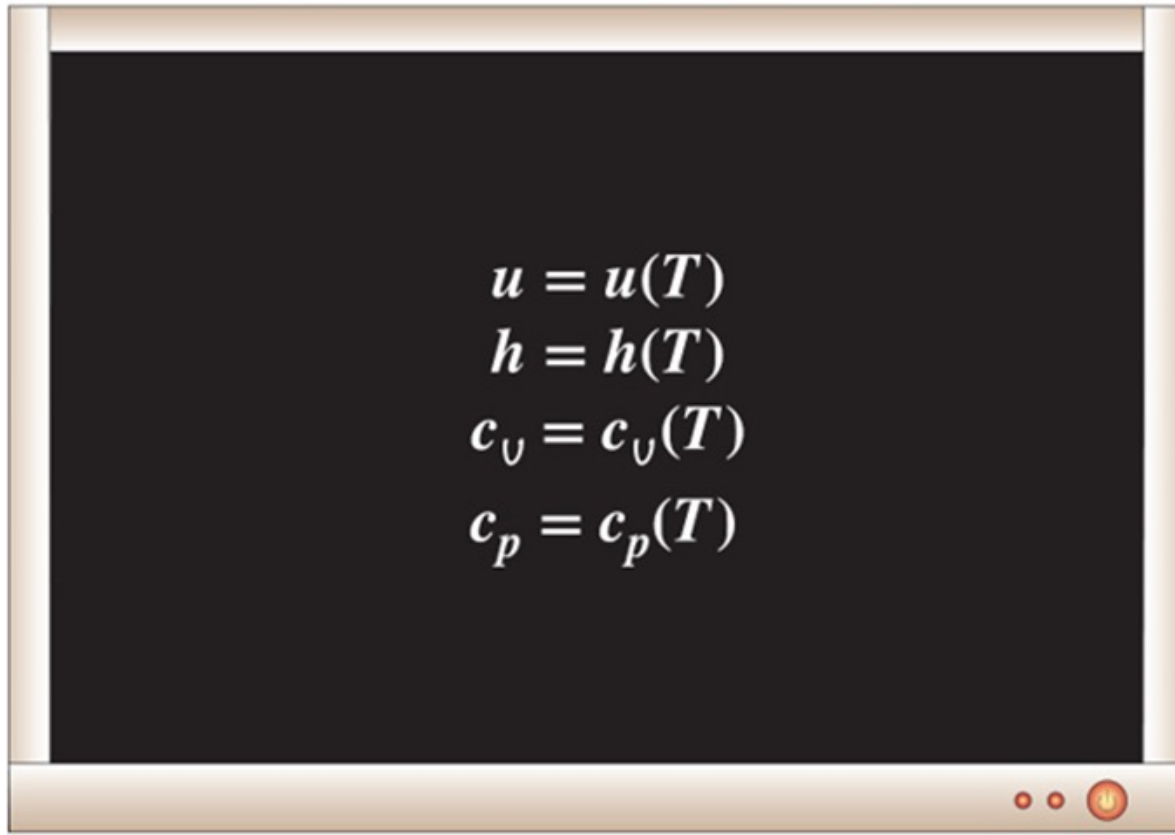
$$c_p = \left( \frac{\partial h}{\partial T} \right)_p$$

= the change in enthalpy with  
temperature at constant  
pressure

# Recap

---

- We have:



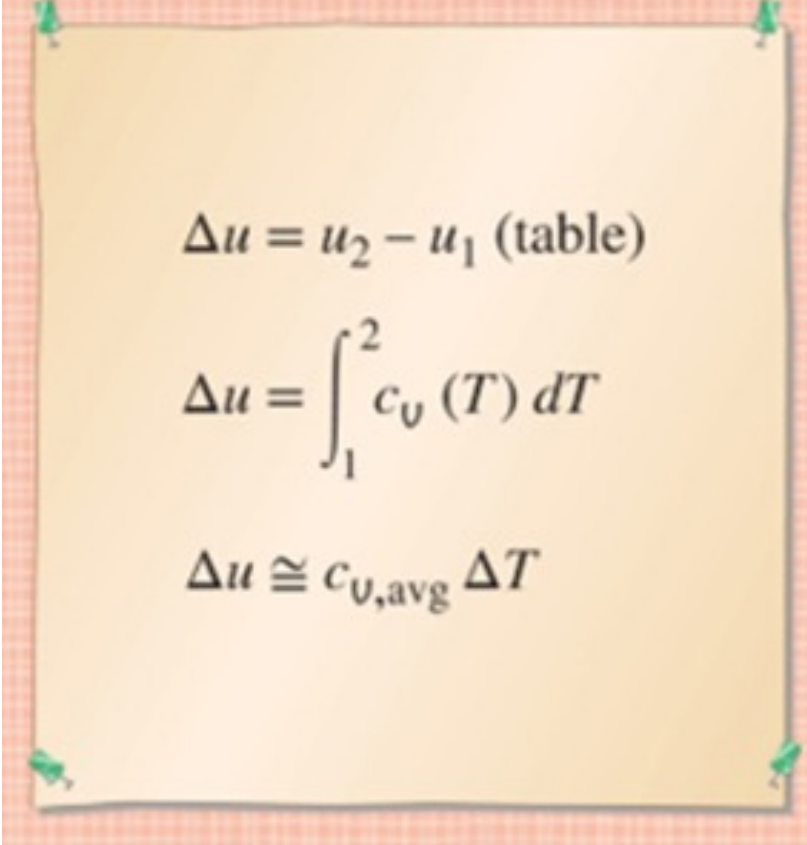
A blackboard with a light-colored frame and a small window-like control bar at the bottom right. The blackboard contains four equations written in white text:

$$u = u(T)$$
$$h = h(T)$$
$$c_v = c_v(T)$$
$$c_p = c_p(T)$$

# Recap

---

- Three ways to calculate  $\Delta h$  and  $\Delta u$ :


$$\Delta u = u_2 - u_1 \text{ (table)}$$

$$\Delta u = \int_1^2 c_v(T) dT$$

$$\Delta u \cong c_{v,\text{avg}} \Delta T$$

# **OBJECTIVES**

# Objectives

---

- Develop the conservation of mass principle
- Apply the conservation of mass principle to various systems including steady- and unsteady-flow control volumes
- Apply the first law of thermodynamics as the statement of the conservation of energy principle to control volumes

# Objectives

---

- Identify the energy carried by a fluid stream crossing a control surface as the sum of internal energy, flow work, kinetic energy, and potential energy of the fluid and to relate the combination of the internal energy and the flow work to the property enthalpy

# Objectives

---

- Solve energy balance problems for common steady-flow devices such as nozzles, compressors, turbines, throttling valves, mixers, heaters, and heat exchangers
- Apply the energy balance to general unsteady-flow processes with particular emphasis on the uniform-flow process as the model for commonly encountered charging and discharging processes

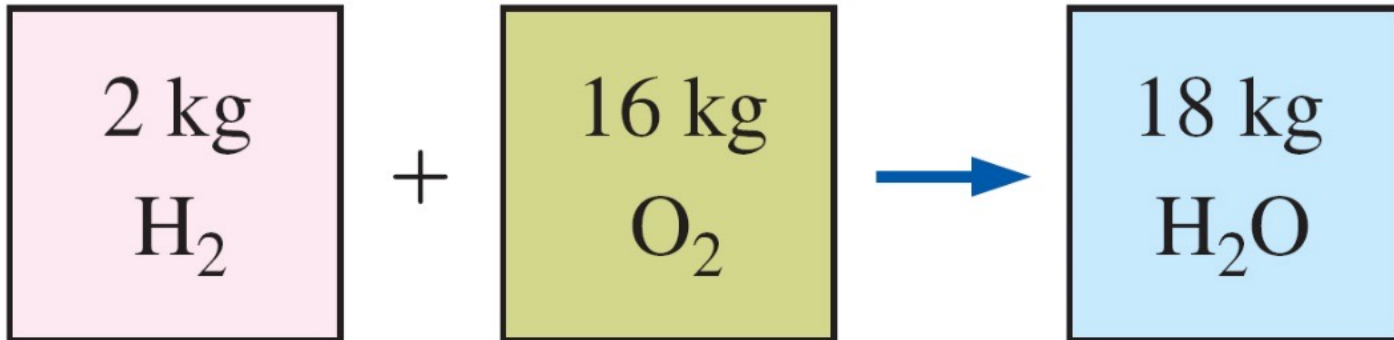


# **CONSERVATION OF MASS**

# Conservation of Mass

---

- Conservation of mass: Mass, like energy, is a conserved property, and it cannot be created or destroyed during a process



# Conservation of Mass

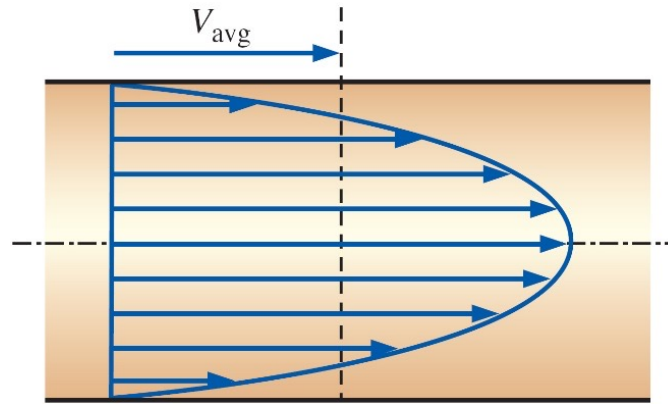
---

- In using control volumes, mass can cross the boundaries, and so we must keep track of the amount of mass entering and leaving the control volume

# Conservation of Mass

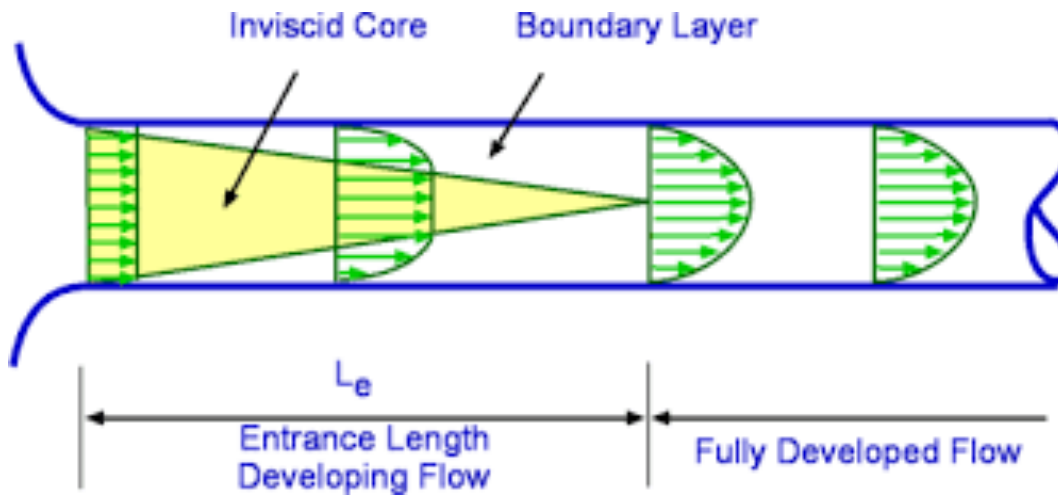
---

- Let's look at a flow in a pipe:



# Conservation of Mass

- Let's look at a flow in a pipe:

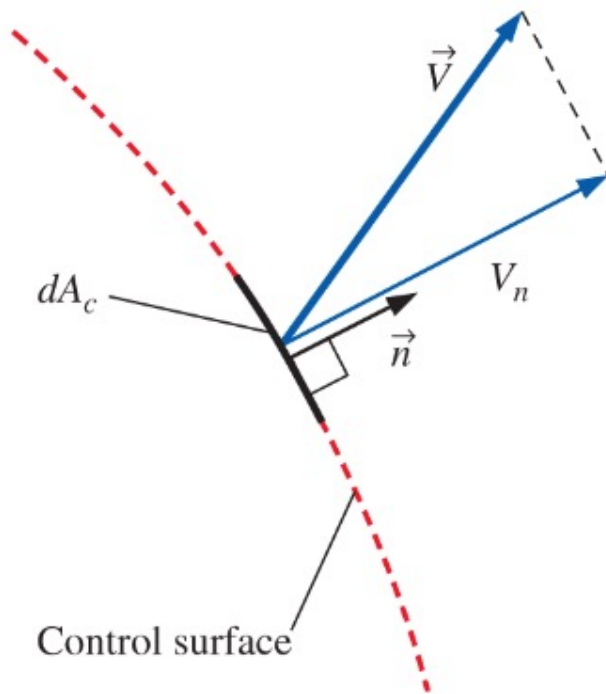


# Conservation of Mass

---

- Mass and volume flow rates

$$\delta \dot{m} = \rho V_n dA_c$$



# Conservation of Mass

---

- Mass and volume flow rates

$$\delta \dot{m} = \rho V_n dA_c$$

$$\dot{m} = \int \delta \dot{m} = \int \rho V_n dA_c$$

$$\dot{m} = \rho V_{avg} A_c$$

$$\dot{m} = \rho \dot{V} = \frac{\dot{V}}{v}$$

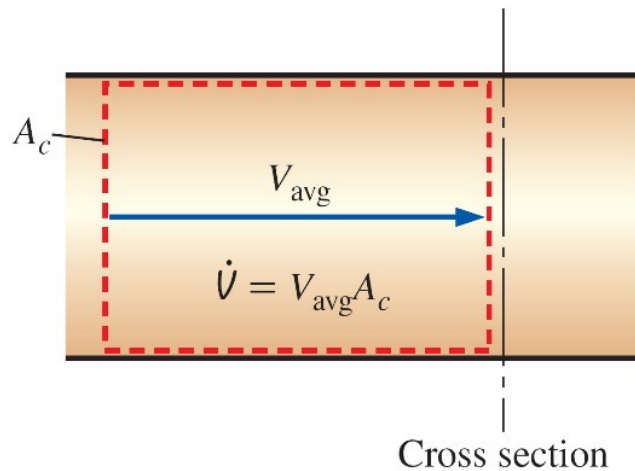
# Conservation of Mass

---

- Mass and volume flow rates

$$V_{avg} = \frac{1}{A_c} \int V_n dA_c$$

$$\dot{V} = \int V_n dA_c = V_{avg} A_c$$

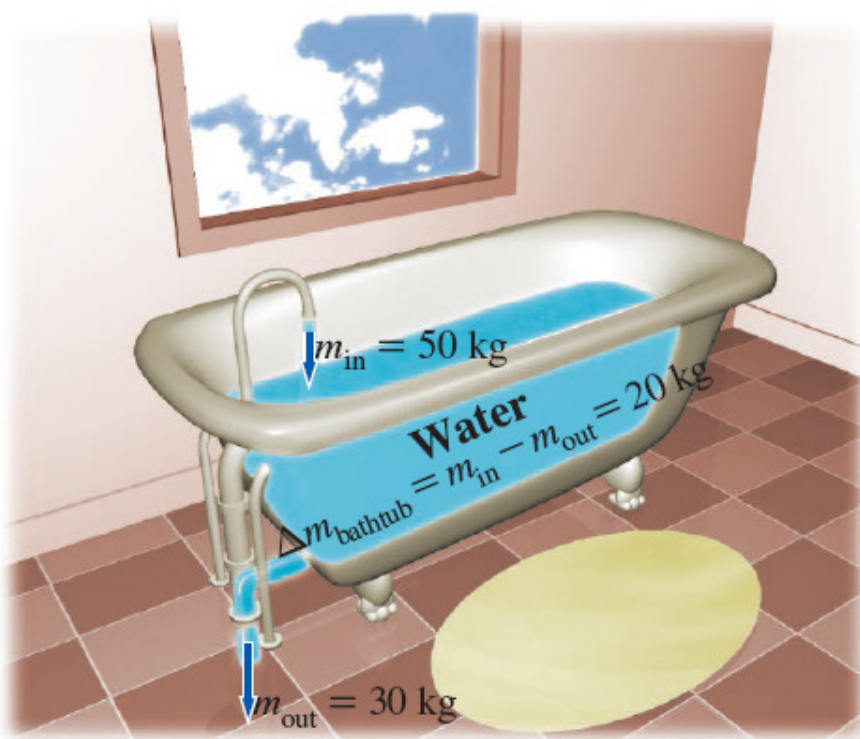




# Conservation of Mass

- The conservation of mass for a control volume:

$$\begin{aligned} & (\text{Total mass entering the CV during } \Delta t) - (\text{Total mass leaving the CV during } \Delta t) \\ & = (\text{Net change of mass within the CV during } \Delta t) \end{aligned}$$



# Conservation of Mass

---

- The conservation of mass for a control volume:

$$m_{in} - m_{out} = \Delta m_{CV}$$

$$\Delta m_{CV} = m_{final} - m_{initial}$$

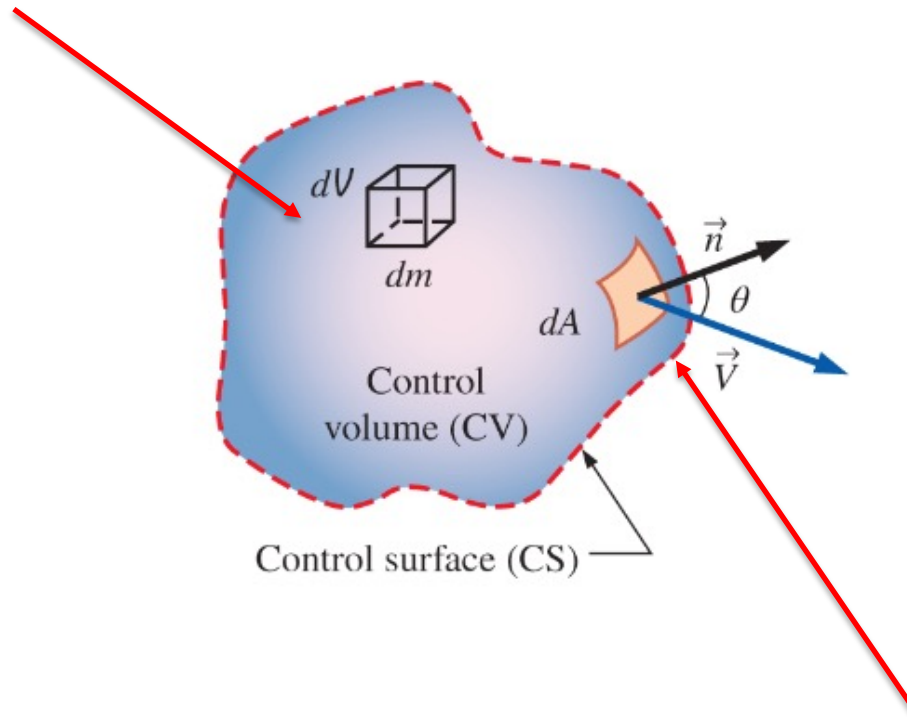
$$\dot{m}_{in} - \dot{m}_{out} = \frac{dm_{CV}}{dt}$$

*These equations are often referred to as the **mass balance** and are applicable to any control volume undergoing any kind of process.*

# Conservation of Mass

- We can write in the integral form:

Total mass within the CV



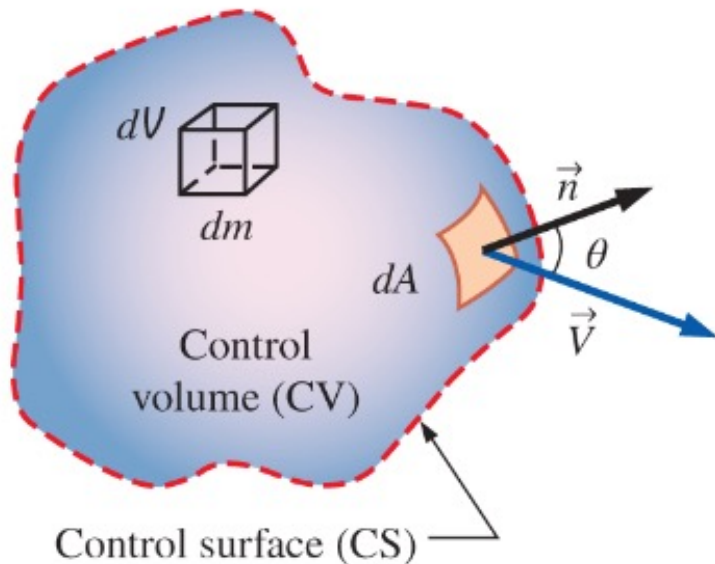
The mass flow rate through the boundaries

# Conservation of Mass

- Total mass within the CV:

Total mass within the CV

$$m_{CV} = \int_{CS} \rho dV$$

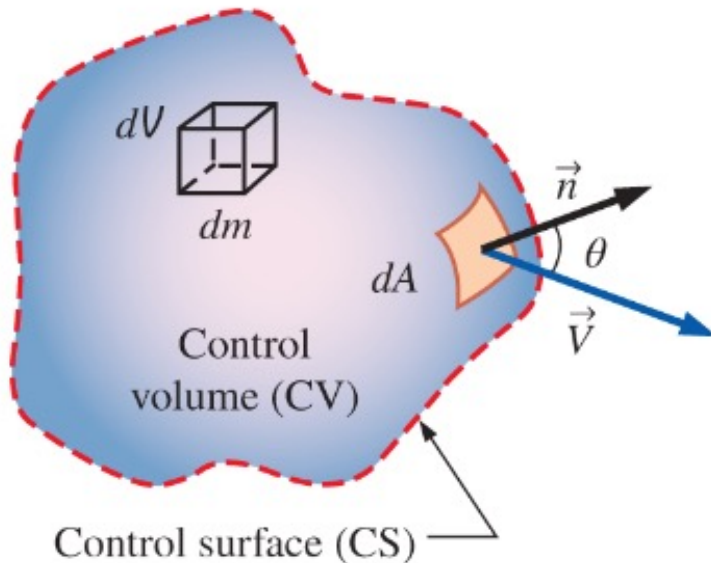


# Conservation of Mass

- The time rate of change of the amount of mass within the control volume is expressed as:

Rate of change of mass within the CV:

$$\frac{dm_{CV}}{dt} = \frac{d}{dt} \int_{CS} \rho dV$$



What does happen when  $\frac{dm_{CV}}{dt} = 0$ ?

# Conservation of Mass

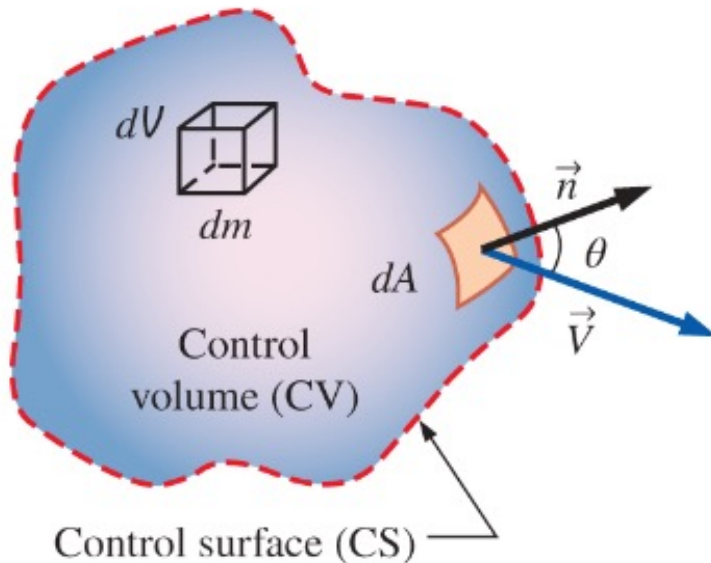
- The mass flow rate through  $dA$  is proportional to the fluid density and normal velocity ( $V_n$ )

*Normal component of velocity*

$$V_n = V \cos(\theta) = \vec{V} \cdot \vec{n}$$

*Differential mass flow rate*

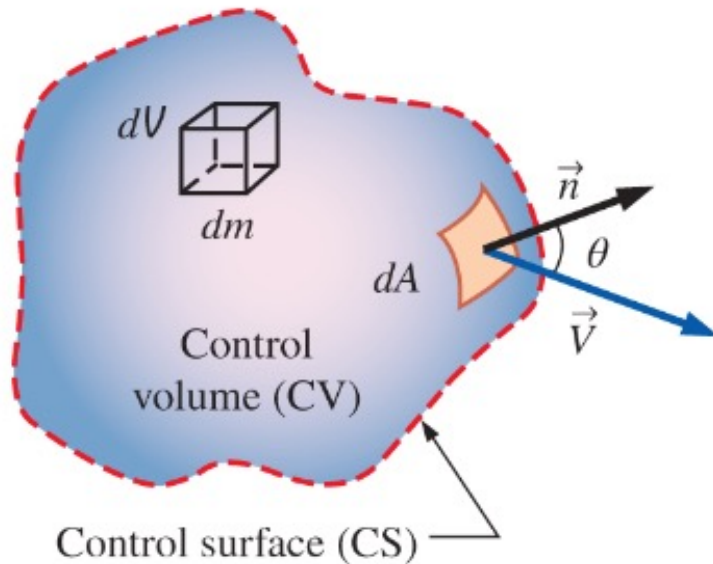
$$\begin{aligned} \delta \dot{m} &= \rho V_n dA = \rho (V \cos(\theta)) dA \\ &= \rho (\vec{V} \cdot \vec{n}) dA \end{aligned}$$



# Conservation of Mass

- We can write the net flow rate:

Net mass flow rate:  $\dot{m}_{net} = \int_{CS} \delta \dot{m} = \int_{CS} \rho V_n dA = \int_{CS} \rho (\vec{V} \cdot \vec{n}) dA$



# Conservation of Mass

---

- We can write the net flow rate:

$$\frac{dm_{CV}}{dt} + \dot{m}_{out} - \dot{m}_{in} = 0$$

$$\frac{dm_{CV}}{dt} = \frac{d}{dt} \int_{CS} \rho dV$$

$$\dot{m}_{net} = \int_{CS} \delta \dot{m} = \int_{CS} \rho V_n dA = \int_{CS} \rho (\vec{V} \cdot \vec{n}) dA$$

$$\frac{d}{dt} \int_{CS} \rho dV + \int_{CS} \rho (\vec{V} \cdot \vec{n}) dA = 0$$



# Conservation of Mass

---

- We can write the net flow rate:

$$\frac{d}{dt} \int_{CS} \rho dV + \int_{CS} \rho (\vec{V} \cdot \vec{n}) dA = 0$$

$$\frac{d}{dt} \int_{CS} \rho dV + \sum_{out} \rho |V_n| A - \sum_{in} \rho |V_n| A = 0$$

# Conservation of Mass

---

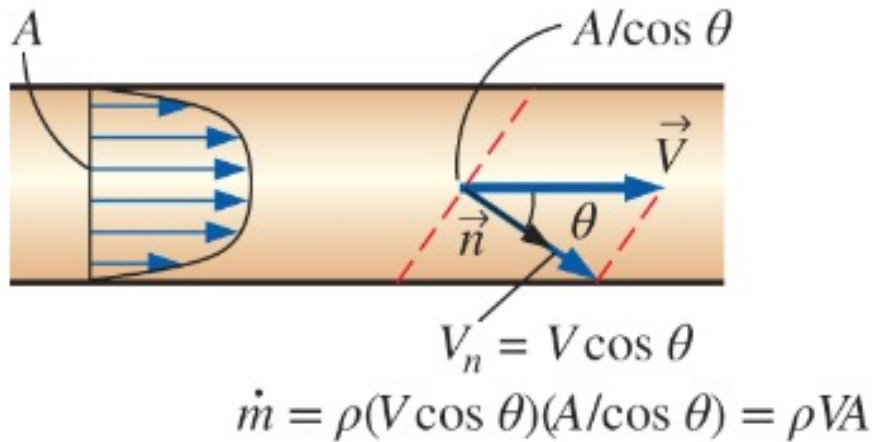
- We can write the net flow rate:

$$\frac{d}{dt} \int_{CS} \rho dV = \sum_{in} \dot{m} - \sum_{out} \dot{m}$$

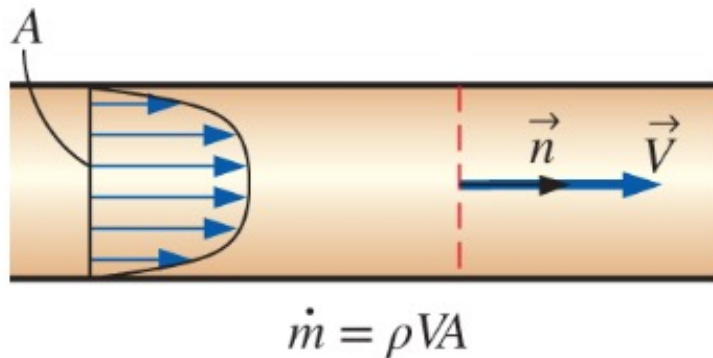
$$\frac{dm_{CV}}{dt} = \sum_{in} \dot{m} - \sum_{out} \dot{m}$$

# Conservation of Mass

- A control surface



*at an angle to the flow*



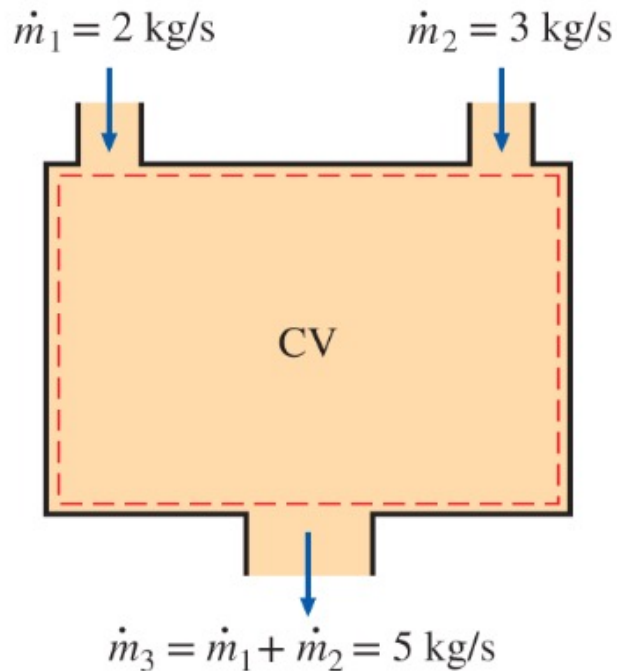
*normal to the flow*

# Conservation of Mass

---

- Steady-flow:

$$\sum_{in} \dot{m} = \sum_{out} \dot{m}$$



# Conservation of Mass

---

- Steady-flow for a single stream:

$$\dot{m}_1 = \dot{m}_2 \quad \rightarrow \quad \rho_1 V_1 A_1 = \rho_2 V_2 A_2$$

# Conservation of Mass

---

- Special case: Incompressible flow:

$$\sum_{in} \dot{V} = \sum_{out} \dot{V}$$

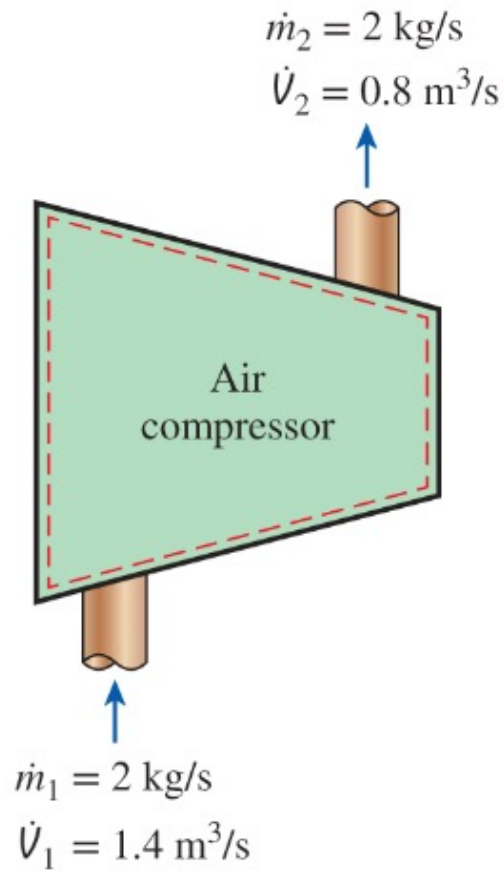
- Steady, incompressible flow (single stream):

$$\dot{V}_1 = \dot{V}_2 \quad \rightarrow \quad V_1 A_1 = V_2 A_2$$

# Conservation of Mass

---

- Let's look at the following steady-state:



# **CLASS ACTIVITY**



# Class Activity

---

- A garden hose attached with a nozzle is used to fill a 10-gallon bucket. The inner diameter of the hose is 2 cm and it reduces to 0.8 cm at the nozzle exits. If it takes 50 s to fill the bucket with water, determine:
  - a) The volume and mass flow rates of water through the hose
  - b) The average velocity of water at the nozzle exit

# Class Activity

---

- Solution (a):

$$\dot{V} = 10 \text{ gal} \times \left( \frac{3.7854 \text{ L}}{1 \text{ gal}} \right) = 37.854 \text{ L}$$

$$\dot{V} = \frac{V}{\Delta t} = \frac{37.854 \text{ L}}{50 \text{ s}} = 0.757 \frac{\text{L}}{\text{s}}$$

$$\dot{m} = \rho \dot{V} = \left( 1000 \frac{\text{kg}}{\text{m}^3} \right) \left( \frac{1000 \frac{\text{kg}}{\text{L}}}{1 \frac{\text{kg}}{\text{m}^3}} \right) \left( 0.757 \frac{\text{L}}{\text{s}} \right) = 0.757 \frac{\text{kg}}{\text{s}}$$

# Class Activity

---

- Solution (b):

$$A_e = \pi \times r_e^2 = \pi \times (0.4 \text{ cm})^2 = 0.5027 \text{ cm}^2 = 0.5027 \times 10^{-4} \text{ m}^2$$

$$V_e = \frac{\dot{V}}{A_e} = \frac{0.757 \text{ L/s}}{0.5027 \times 10^{-4} \text{ m}^2} \left( \frac{1 \text{ m}^3}{1000 \text{ L}} \right) = 15.1 \text{ m/s}$$

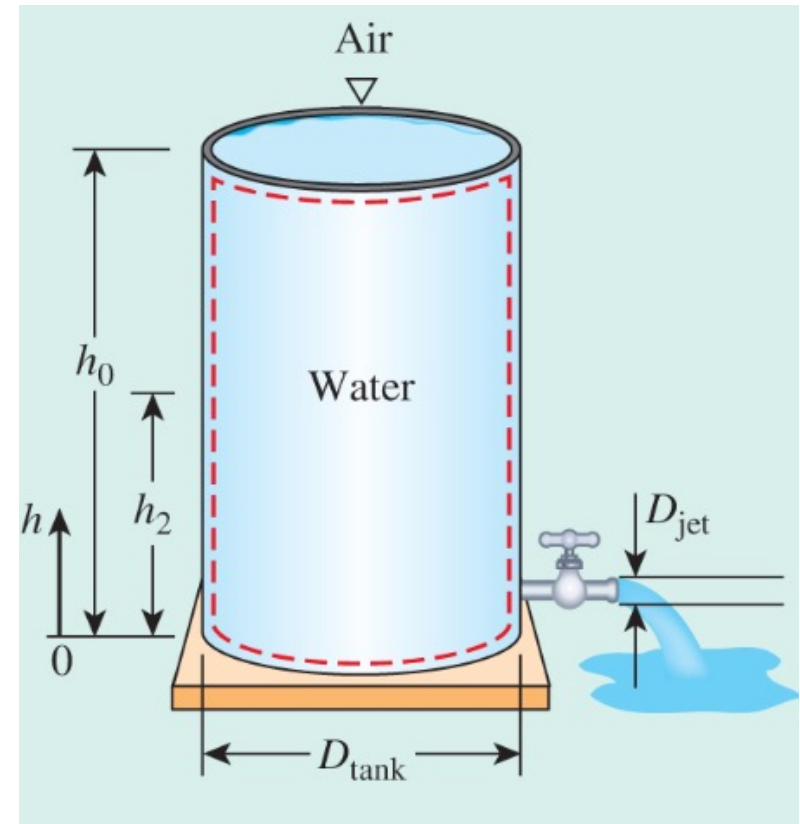
# **CLASS ACTIVITY**

# Class Activity

- A 4-ft high, 3-ft diameter cylindrical water tank whose top is open to the atmosphere is initially filled with water. Now the discharge plug near the bottom of the tank is pulled out, and a water jet whose diameter is 0.5 in streams.

The average velocity of the jet is approximated as  $V = \sqrt{2gh}$  where  $h$  is the height of water in the tank measured from the center of the hole (a variable) and  $g$  is the gravitational acceleration.

Determine how long it takes for the water level in the tank to drop 2 ft from the bottom.



# Class Activity

---

- Solution:

$$\frac{dm_{CV}}{dt} = \dot{m}_{in} - \dot{m}_{out}$$

$$\dot{m}_{in} = 0$$

$$\dot{m}_{out} = (\rho VA)_{out} = \rho \sqrt{2gh} A_{jet}$$

$$m_{CV} = \rho \mathcal{V} = \rho A_{tank} h$$

# Class Activity

---

- Solution:

$$\frac{dm_{CV}}{dt} = \dot{m}_{in} - \dot{m}_{out}$$

$$-\rho\sqrt{2gh} \left( \frac{D_{tank}^2}{4} \right) = \frac{d}{dt} (\rho A_{tank} h)$$

$$-\rho\sqrt{2gh} \left( \frac{D_{tank}^2}{4} \right) = \rho \left( \frac{\pi D_{tank}^2}{4} \right) \frac{dh}{dt}$$

$$dt = \frac{D_{tank}^2}{D_{jet}^2} \frac{dh}{\sqrt{2gh}}$$

# Class Activity

---

- Solution:

$$dt = -\frac{D_{\text{tank}}^2}{D_{\text{jet}}^2} \frac{dh}{\sqrt{2gh}} \quad \begin{cases} t = 0 \rightarrow & h = h_0 \\ t = t \rightarrow & h = h_2 \end{cases}$$

$$\int_0^t dt = -\frac{D_{\text{tank}}^2}{D_{\text{jet}}^2 \sqrt{2g}} \int_{h_1}^{h_2} \frac{dh}{\sqrt{h}}$$

$$t = \frac{(\sqrt{h_0} - \sqrt{h_2})}{\sqrt{\frac{g}{2}}} \left( \frac{D_{\text{tank}}}{D_{\text{jet}}} \right)^2$$



# Class Activity

---

- Solution:

$$t = \frac{(\sqrt{4 \text{ ft}} - \sqrt{2 \text{ ft}})}{\sqrt{\frac{32.2}{2}}} \left( \frac{3 \times 12 \text{ in}}{0.5 \text{ in}} \right)^2 = 757 \text{ s} = 12.6 \text{ min}$$

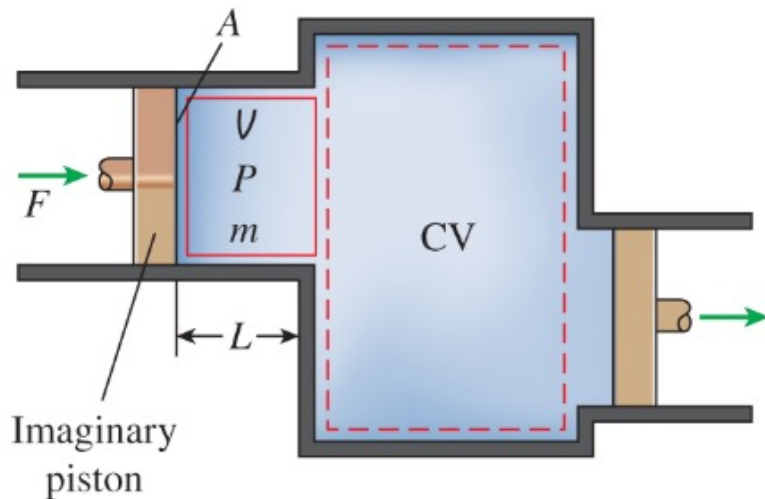
*What does the entire flow exit the system?*

$$t = \frac{(\sqrt{4 \text{ ft}} - \sqrt{0 \text{ ft}})}{\sqrt{\frac{32.2}{2}}} \left( \frac{3 \times 12 \text{ in}}{0.5 \text{ in}} \right)^2 = 43.1 \text{ min}$$

# **FLOW WORK AND ENERGY OF FLOWING FLUID**

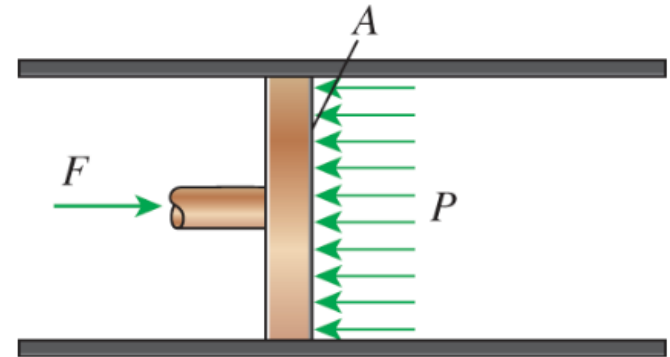
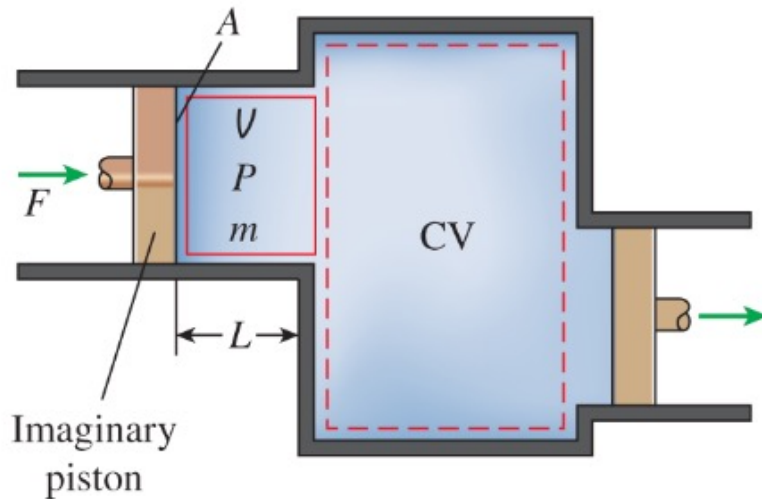
# Flow Work and Energy of Flowing Fluid

- Flow work (or flow energy): The work (or energy) required to push the mass into or out of the control volume. This work is necessary for maintaining a continuous flow through a control volume



# Flow Work and Energy of Flowing Fluid

- If the fluid pressure is  $P$  and the cross-sectional area of the fluid element is  $A$ , the force applied on the fluid by the imaginary piston is:



$$F = PA$$

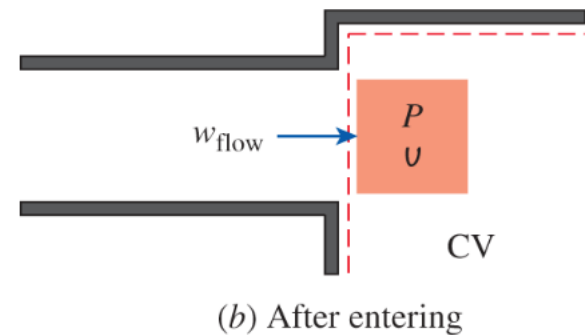
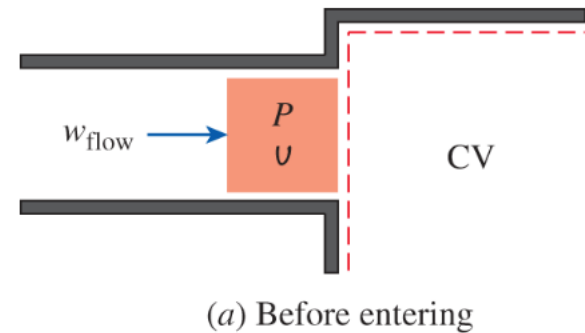
# Flow Work and Energy of Flowing Fluid

- To push the entire fluid element into the control volume, this force must act through a distance  $L$

$$F = PA$$

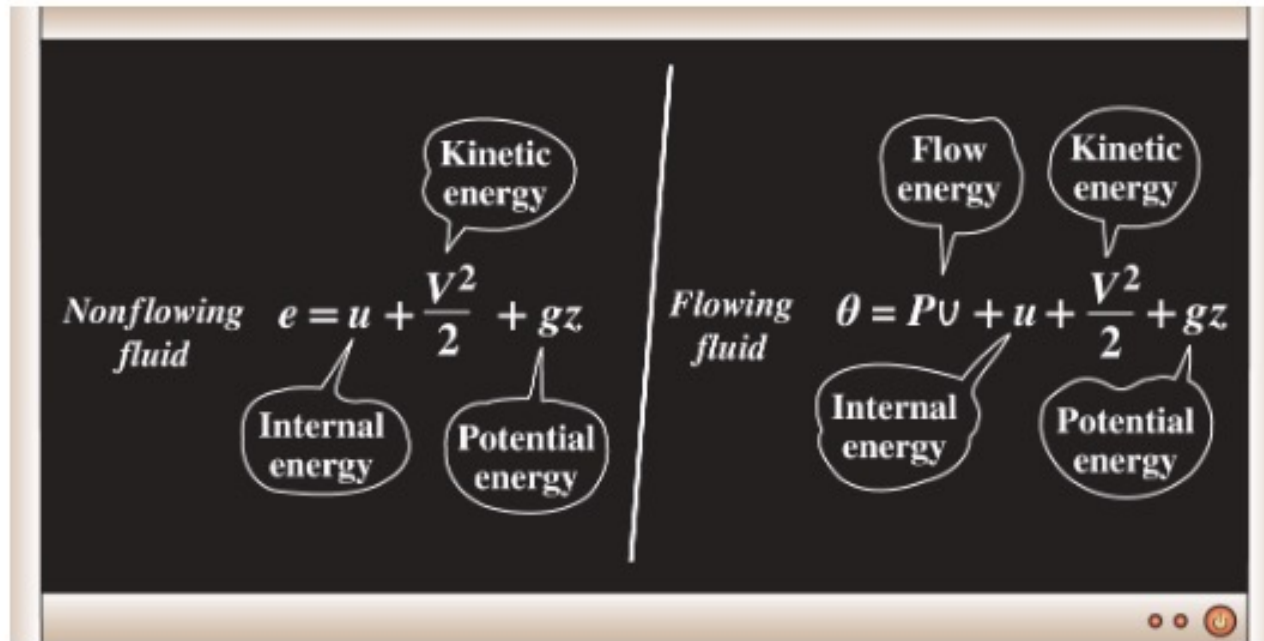
$$W_{flow} = FL = PAL = P\mathcal{V}$$

$$w_{flow} = Pv$$



# Flow Work and Energy of Flowing Fluid

- Total energy of a flowing fluid is:



$$\theta = Pv + e = Pv + (u + ke + pe) = (u + Pv) + ke + pe$$

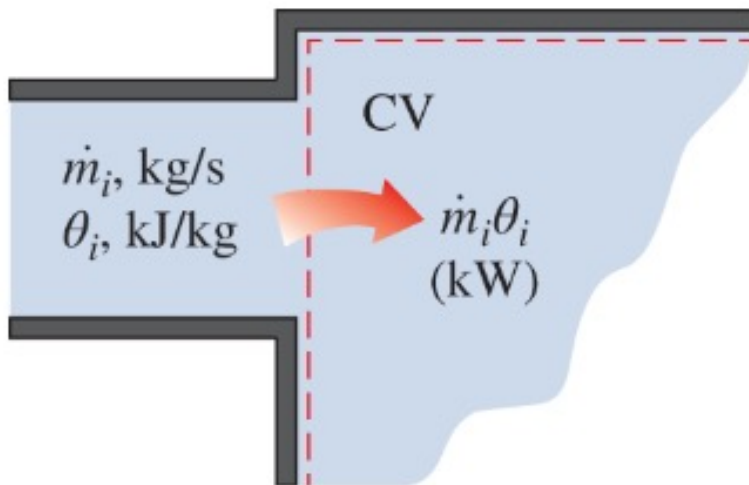
# Flow Work and Energy of Flowing Fluid

---

- Energy transport by mass:

$$E_{mass} = m\theta = m\left(h + \frac{V^2}{2} + gz\right)$$

$$\dot{E}_{mass} = \dot{m}\theta = \dot{m}\left(h + \frac{V^2}{2} + gz\right)$$



# Flow Work and Energy of Flowing Fluid

---

- What does happen when mass is not constant:

$$E_{in,mass} = \int_{m_i} \theta_i \delta m_i = \int_{m_i} \left( h_i + \frac{V_i^2}{2} + gz_i \right) \delta m_i$$

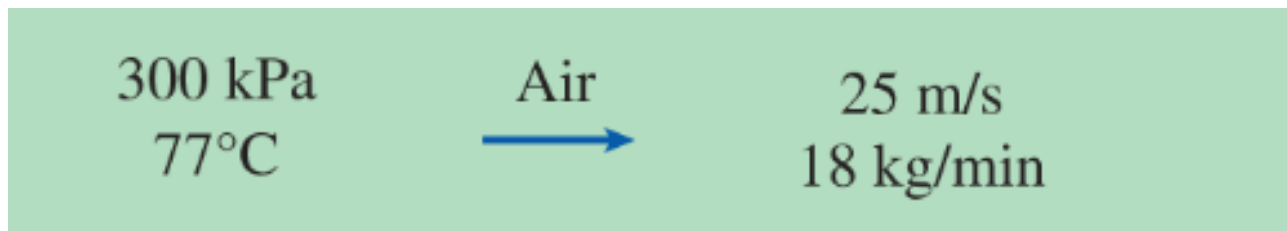


# **CLASS ACTIVITY**

# Class Activity

---

- Air flows steadily in a pipe at 300 kPa, 77 °C, and 25 m/s at a rate of 18 kg/min. Determine:
  - a) The diameter of the pipe
  - b) The rate of flow energy
  - c) The rate of energy transport by mass
  - d) The error involved in part c if the kinetic energy is neglected.



# Class Activity

---

- Solution (assumptions):
  - The flow is steady
  - The potential energy is negligible
  - $R = 0.287 \text{ kJ/kg-K}$
  - $c_p = 1.008 \text{ kJ/kg-K}$  (at 350 K from Table A-2b)

# Class Activity

---

- Solution (a):

$$v = \frac{RT}{P} = \frac{(0.287 \frac{\text{kJ}}{\text{kg} \cdot \text{K}})(77 + 273)}{300 \text{ kPa}} = 0.3349 \frac{\text{m}^3}{\text{kg}}$$

$$A = \frac{\dot{m}v}{V} = \frac{\left(\frac{18 \text{ kg}}{60 \text{ s}}\right) \left(0.3349 \frac{\text{m}^3}{\text{kg}}\right)}{25 \frac{\text{m}}{\text{s}}} = 0.004018 \text{ m}^2$$

$$D = \sqrt{\frac{4A}{\pi}} = \sqrt{\frac{4(0.004018 \text{ m}^2)}{\pi}} = 0.0715 \text{ m}$$

# Class Activity

---

- Solution (b):

$$\dot{W}_{flow} = \dot{m}Pv = \left(\frac{18 \text{ kg}}{60 \text{ s}}\right) (300 \text{ kPa}) \left(0.3349 \frac{\text{m}^3}{\text{kg}}\right) = 30.14 \text{ kW}$$

# Class Activity

---

- Solution (c):

$$\dot{E}_{mass} = \dot{m}(h + ke) = \dot{m}\left(c_p T + \frac{V^2}{2}\right)$$

$$\dot{E}_{mass} = \left(\frac{18 \text{ kg}}{60 \text{ s}}\right) \left[ \left(1.008 \frac{\text{kJ}}{\text{kg} \cdot \text{K}}\right) (77 + 273 \text{ K}) + \left(\frac{1}{2}\right) \left(25 \frac{\text{m}}{\text{s}}\right)^2 \left(\frac{1 \frac{\text{kJ}}{\text{kg}}}{1000 \frac{\text{m}^2}{\text{s}^2}}\right) \right]$$

$$\dot{E}_{mass} = 105.94 \text{ kW}$$

# Class Activity

---

- Solution (d):

$$\dot{E}_{mass} = \dot{m}(h + ke) = \dot{m}h = \dot{m}c_p T$$

$$\dot{E}_{mass} = \dot{m}c_p T = \left(\frac{18 \text{ kg}}{60 \text{ s}}\right) \left(1.005 \frac{\text{kJ}}{\text{kg} \cdot \text{K}}\right) (77 + 273\text{K}) = 105.84 \text{ kW}$$